

Appendix A

Thermodynamic Model

In this section the first order differential equation describing the pressure changes inside the muscle valve system is formulated. The discussion is based on the works of Daerden (104) and Brun (66).

The first law of thermodynamics is applied to a muscle with its valve island of 6 on/off valves. The muscle itself and its tubing until the different input and exhaust valve orifices are taken as control volume V . Figure (A.1) gives a schematic representation where the two inlet valves and the four exhaust valves are respectively depicted as one inlet and one exhaust. The first law is given in its rate form and

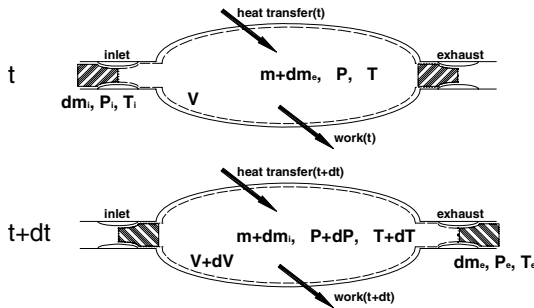


Fig. A.1 Muscle and valves on time step t and $t + dt$.

expresses that the variation of the total energy of an amount of fluid is equal to the sum of the work done by the exerted forces and the net heat transfer with the surrounding. Assuming a uniform thermodynamic state inside the control volume the first law of thermodynamics can be written as follows (variation referred to time):

$$dU + dE_k + dE_p = \delta W + \delta Q \quad (\text{A.1})$$

with:

dU = variation of the fluid's total internal energy

dE_k = variation of the fluid's total kinetic energy

dE_p = variation of the fluid's total potential energy

δW = work done by external forces

δQ = the net transfer of heat across the boundary

The pressurized air can be regarded as an ideal gas for which the following relations hold:

$$PV = mrT \quad (\text{A.2})$$

$$u = c_v(T - T_0) \quad (\text{A.3})$$

$$h = c_p(T - T_0) \quad (\text{A.4})$$

$$c_p = c_v + r \quad (\text{A.5})$$

with:

$$P = \text{absolute pressure} \quad (\text{A.6})$$

$$V = \text{air volume} \quad (\text{A.7})$$

$$m = \text{air mass} \quad (\text{A.8})$$

$$T = \text{temperature} \quad (\text{A.9})$$

$$r = \text{dry air gas constant} = 287 \text{Jkg}^{-1} \text{K}^{-1} \quad (\text{A.10})$$

$$u = \text{specific internal energy} \quad (\text{A.11})$$

$$h = \text{specific enthalpy} \quad (\text{A.12})$$

$$c_v = \text{constant volume specific heat} = 718 \text{Jkg}^{-1} \text{K}^{-1} \text{ for dry air at } 300\text{K} \quad (\text{A.13})$$

$$c_p = \text{constant pressure specific heat} = 1005 \text{Jkg}^{-1} \text{K}^{-1} \text{ for dry air at } 300\text{K} \quad (\text{A.14})$$

$$T_0 = \text{reference temperature which is taken zero} \quad (\text{A.15})$$

To calculate the different variations in equation A.1 for the open muscle-valve system, the constant mass ($m + dm_i + dm_e$) is studied at two instant time steps t and $t + dt$ as depicted in figure (A.1). At time t , pressurized air with mass dm_i is about to enter the control volume V while mass $m + dm_e$ is inside this volume. At $t + dt$ mass dm_e is leaving while the mass inside the control volume is $m + dm_i$. Evaluating equation A.3 between the two time steps results in:

$$dU = [(m + dm_i)c_v(T + dT) + dm_e c_v T_e] - [(m + dm_e)c_v T + dm_i c_v T_i] \quad (\text{A.16})$$

While neglecting second order terms, equation A.16 leads to:

$$dU = mc_v dT + dm_i c_v (T - T_i) + dm_e c_v (T_e - T) \quad (\text{A.17})$$

Neglecting furthermore the kinetic energy of the air inside the muscle against the kinetic energy of the inlet and exhaust, the variation of kinetic and potential energy becomes:

$$dE_k = dm_e \frac{C_e^2}{2} - dm_i \frac{C_i^2}{2} \quad (\text{A.18})$$

$$dE_p = dm_e g z_e - dm_i g z_i \quad (\text{A.19})$$

The work exchanged with the environment, while assuming reversibility, is expressed as:

$$dW = -PdV + P_i dV_i - P_e dV_e \quad (\text{A.20})$$

with the first term, the work done by the muscle and the other two terms associated with the work needed to transport dm_i and dm_e in and out the muscle volume.

Combining the first law of thermodynamics (A.1) with equations (A.17), (A.18), (A.19) and (A.20) gives:

$$\begin{aligned} mc_v dT + c_v T (dm_i - dm_e) &= -PdV \\ &+ dm_i \left(c_v T_i + P_i v_i + \frac{C_i^2}{2} + g z_i \right) \\ &- dm_e \left(c_v T_e + P_e v_e + \frac{C_e^2}{2} + g z_e \right) + \delta Q \end{aligned} \quad (\text{A.21})$$

with v_i and v_e the specific volume of inlet and exhaust. Taking into account conservation of mass and the definition of enthalpy:

$$dm = dm_i - dm_e \quad (\text{A.22})$$

$$h = u + Pv \quad (\text{A.23})$$

Differentiating the perfect gas law (A.2) gives:

$$d(PV) = PdV + VdP = mrdT + rTdm \quad (\text{A.24})$$

Using (A.24), equation (A.21) can be transformed to:

$$\begin{aligned} \frac{c_v}{r} d(PV) &= -PdV \\ &+ dm_i \left(h_i + \frac{C_i^2}{2} + g z_i \right) \\ &- dm_e \left(h_e + \frac{C_e^2}{2} + g z_e \right) + \delta Q \end{aligned} \quad (\text{A.25})$$

Flows through small orifices, such as valves and tubes, are assumed to be adiabatic and since no mechanical work is exchanged with the surroundings, for these situations is stated:

$$h + \frac{C^2}{2} = \text{constant} \quad (\text{A.26})$$

Thus for inlet and exhaust can be written:

$$h_i + \frac{C_i^2}{2} = h_s = c_p T_s \quad (\text{A.27})$$

$$h_e + \frac{C_e^2}{2} = h = c_p T \quad (\text{A.28})$$

with h_s and T_s the enthalpy and temperature of the pressurized air supply buffer, h and T are the enthalpy and temperature of the pressurized air inside the muscle volume. For equations (A.27) and (A.28) kinetic energy is neglected since the considered volumes are assumed large enough. Taking into account these two equations and the definition $\gamma = c_p/c_v$ and relation (A.5), the energy balance (A.25) can be rewritten in the following form, if potential energy of the air masses is neglected:

$$dP = -\frac{\gamma}{V} (PdV + rT_s dm_i - rT dm_e + (\gamma - 1)\delta Q) \quad (\text{A.29})$$

If furthermore an adiabatic process is considered, $\delta Q = 0$, equation (A.29) becomes:

$$dP = \frac{\gamma}{V} (-PdV + rT_s dm_i - rT dm_e) \quad (\text{A.30})$$

Expression (A.30) is valid for the so called isentropic process, where adiabatic and reversibility conditions are assumed. The non-ideal conditions can be represented in analogy with the polytropic process, by substituting γ with a polytropic coefficient n in equation (A.30) ($n = 1.2$):

$$dP = \frac{n}{V} (-PdV + rT_s dm_i - rT dm_e) \quad (\text{A.31})$$

with dm_i and dm_e determined by air flows through the different inlet and exhaust valves and dependent on the number of valves that are opened.

Appendix B

Kinematics and Dynamics of the Biped Lucy during a Single Support Phase

B.1 Kinematics

The biped model during a single support phase is depicted in figure B.1. For the following derivations it is supposed that both legs are identical. Hereby assuming all inertial properties and the length of the upper and lower leg to be pairwise equal. l_i , m_i and I_i are respectively the length, mass and moment of inertia with respect to the local COG G_i of link i . The location of the COG's G_i are given by $J_1G_1 = J_6G_5 = \alpha l_1$, $J_2G_2 = J_5G_4 = \beta l_2$ and $J_3G_3 = \gamma l_3$ and for the foot $J_6G_6 = \sigma l_6$ where $0 < \alpha, \beta, \gamma, \sigma < 1$. The position of each link i is given by the angle θ_i , measured with respect to the horizontal axis.

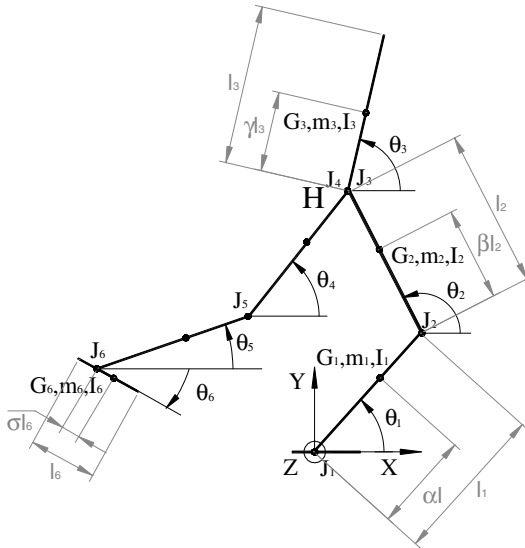


Fig. B.1 Model of the biped during a single support phase.

The hip takes a central position, so the location of the different COG's is calculated with reference to this point.

$$X_H = l_1 \cos \theta_1 + l_2 \cos \theta_2 \quad (\text{B.1a})$$

$$Y_H = l_1 \sin \theta_1 + l_2 \sin \theta_2 \quad (\text{B.1b})$$

The vectors defining the position of the local COG's of each of the five links are calculated as:

$$\overline{OG}_1 = (X_H, Y_H)^T - (1 - \alpha) l_1 (\cos \theta_1, \sin \theta_1)^T - l_2 (\cos \theta_2, \sin \theta_2)^T \quad (\text{B.2a})$$

$$\overline{OG}_2 = (X_H, Y_H)^T - (1 - \beta) l_2 (\cos \theta_2, \sin \theta_2)^T \quad (\text{B.2b})$$

$$\overline{OG}_3 = (X_H, Y_H)^T + \gamma l_3 (\cos \theta_3, \sin \theta_3)^T \quad (\text{B.2c})$$

$$\overline{OG}_4 = (X_H, Y_H)^T - (1 - \beta) l_2 (\cos \theta_4, \sin \theta_4)^T \quad (\text{B.2d})$$

$$\overline{OG}_5 = (X_H, Y_H)^T - (1 - \alpha) l_1 (\cos \theta_5, \sin \theta_5)^T - l_2 (\cos \theta_4, \sin \theta_4)^T \quad (\text{B.2e})$$

$$\begin{aligned} \overline{OG}_6 = (X_H, Y_H)^T + \sigma l_6 (\cos \theta_6, \sin \theta_6)^T \\ - l_1 (\cos \theta_5, \sin \theta_5)^T - l_2 (\cos \theta_4, \sin \theta_4)^T \end{aligned} \quad (\text{B.2f})$$

The position of the global COG of the robot, stance foot not included, is given by:

$$\overline{OG} = (X_G, Y_G)^T \quad (\text{B.3})$$

with:

$$\begin{aligned} X_G = X_H + a_1 \cos \theta_1 + a_2 \cos \theta_2 + a_3 \cos \theta_3 \\ + a_4 \cos \theta_4 + a_5 \cos \theta_5 + a_6 \cos \theta_6 \end{aligned} \quad (\text{B.3a})$$

$$\begin{aligned} Y_G = Y_H + a_1 \sin \theta_1 + a_2 \sin \theta_2 + a_3 \sin \theta_3 \\ + a_4 \sin \theta_4 + a_5 \sin \theta_5 + a_6 \sin \theta_6 \end{aligned} \quad (\text{B.3b})$$

and:

$$\begin{aligned} a_1 &= -(1 - \alpha) \eta_1 l_1 \\ a_2 &= -[\eta_1 + (1 - \beta) \eta_2] l_2 \\ a_3 &= \gamma \eta_3 l_3 \\ a_4 &= -[\eta_1 + \eta_6 + (1 - \beta) \eta_2] l_2 \\ a_5 &= -[\eta_6 + (1 - \alpha) \eta_1] l_1 \\ a_6 &= \sigma \eta_6 l_6 \end{aligned}$$

and:

$$\eta_i = \frac{m_i}{2(m_1 + m_2) + m_3 + m_6}$$

The first and second derivative of (B.3a) and (B.3b), which are required for the derivation of the dynamic model and the ZMP, are straightforward and thus not explicitly listed here.

B.2 Dynamics

With the swing foot included, the robot has 6 DOF during the single support phase if the robot is assumed to move only in the sagittal plane. These degrees of freedom are represented by the 6-dimensional vector:

$$\mathbf{q} = [\theta_1 \ \theta_2 \ \theta_3 \ \theta_4 \ \theta_5 \ \theta_6]^T \quad (\text{B.4})$$

The dynamics are represented by 6 equations of motion of which the i th equation can be written with the Lagrange formulation as:

$$\frac{d}{dt} \left\{ \frac{\partial K}{\partial \dot{q}_i} \right\} - \frac{\partial K}{\partial q_i} + \frac{\partial U}{\partial q_i} = Q_i \quad (i = 1 \dots 6) \quad (\text{B.5})$$

with K and U , respectively the total kinetic and gravitational energy of the robot, Q_i are the generalized forces associated with the generalized coordinates q_i .

The total kinetic energy can be found by the summation of the separate kinetic energy values of each link:

$$K = \sum_{i=1}^6 K_i = \frac{1}{2} \sum_{i=1}^6 (m_i v_{G_i}^2 + I_i \dot{\theta}_i^2) \quad (\text{B.6})$$

with $\bar{v}_{G_i} = (\dot{X}_{G_i}, \dot{Y}_{G_i})^T$ the velocity of the COG of link i and $\dot{\theta}_i$ the angular velocity. The expression of the total kinetic energy is quite long and is not explicitly listed here.

The gravitational (potential) energy is given by:

$$U = MgY_G \quad (\text{B.7})$$

The generalized forces are the different net torques acting on each link of the robot (see figure B.2):

$$\mathbf{Q} = \boldsymbol{\tau} = \begin{bmatrix} \tau_1 \\ \tau_2 \\ \tau_3 \\ \tau_4 \\ \tau_5 \\ \tau_6 \end{bmatrix} = \begin{bmatrix} \tau_{K_S} - \tau_{A_S} \\ \tau_{H_S} - \tau_{K_S} \\ -\tau_{H_S} - \tau_{H_a} \\ \tau_{H_a} - \tau_{K_a} \\ \tau_{K_a} - \tau_{A_a} \\ \tau_{A_a} \end{bmatrix} \quad (\text{B.8})$$

The H , K and A stands for ‘‘Hip’’, ‘‘Knee’’ and ‘‘Ankle’’ respectively, a stands for ‘‘air’’, and s for ‘‘stance’’. Expression (B.8) gives the relations between net torques and applied joint torques.

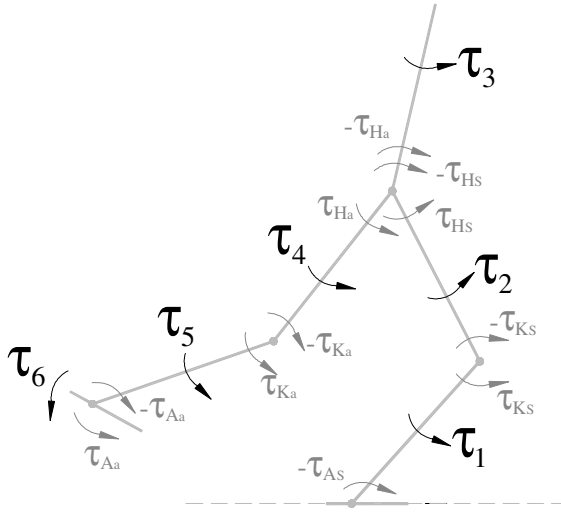


Fig. B.2 Definition of net torques and joint torques.

The 6 equations of motion (B.5) can be written in the following form (376):

$$D(\mathbf{q})\ddot{\mathbf{q}} + C(\mathbf{q}, \dot{\mathbf{q}})\dot{\mathbf{q}} + G(\mathbf{q}) = \boldsymbol{\tau} \quad (\text{B.9})$$

with $D(\mathbf{q})$ the inertia matrix, $C(\mathbf{q}, \dot{\mathbf{q}})$ the centrifugal/coriolis matrix, $G(\mathbf{q})$ the gravitational torque vector and $\boldsymbol{\tau}$ the net torque vector.

The inertia matrix can be calculated with the following relation to the kinetic energy:

$$K = \frac{1}{2} \dot{\mathbf{q}}^T D(\mathbf{q}) \dot{\mathbf{q}} \quad (\text{B.10})$$

The elements of the centrifugal/coriolis matrix c_{kj} can be found with the following expression (376):

$$c_{kj} = \sum_{i=1}^6 c_{ijk} \dot{\theta}_i = \sum_{i=1}^6 \frac{1}{2} \left\{ \frac{\partial d_{kj}}{\partial \theta_i} + \frac{\partial d_{ki}}{\partial \theta_j} - \frac{\partial d_{ij}}{\partial \theta_k} \right\} \dot{\theta}_i \quad (\text{B.11})$$

with c_{ijk} the so called Christoffel symbols and d_{ij} the elements of the inertia matrix $D(\mathbf{q})$. The elements of the gravitational torque vector g_i are given by:

$$g_i = \frac{\partial U}{\partial q_i} \quad (\text{B.12})$$

As a result all the parameters of the dynamic model are given below:

- Inertia matrix $D(\mathbf{q})$:

$$\begin{aligned}
 d_{11} &= I_1 + l_1^2 [(1 + \alpha^2)m_1 + 2m_2 + m_3 + m_6] \\
 d_{12} &= l_1 l_2 [m_1 + (1 + \beta)m_2 + m_3 + m_6] \cos(\theta_1 - \theta_2) = d_{21} \\
 d_{13} &= l_1 l_3 \gamma m_3 \cos(\theta_1 - \theta_3) = d_{31} \\
 d_{14} &= l_1 l_2 [(\beta - 1)m_2 - m_1 - m_6] \cos(\theta_1 - \theta_4) = d_{41} \\
 d_{15} &= l_1^2 [(\alpha - 1)m_1 - m_6] \cos(\theta_1 - \theta_5) = d_{51} \\
 d_{16} &= l_1 l_6 m_6 \sigma \cos(\theta_1 - \theta_6) = d_{61} \\
 d_{22} &= I_2 + l_2^2 [m_1 + (1 + \beta^2)m_2 + m_3 + m_6] \\
 d_{23} &= l_2 l_3 \gamma m_3 \cos(\theta_2 - \theta_3) = d_{32} \\
 d_{24} &= l_2^2 [(\beta - 1)m_2 - m_1 - m_6] \cos(\theta_2 - \theta_4) = d_{42} \\
 d_{25} &= l_1 l_2 [(\alpha - 1)m_1 - m_6] \cos(\theta_2 - \theta_5) = d_{52} \\
 d_{26} &= l_2 l_6 m_6 \sigma \cos(\theta_2 - \theta_6) = d_{62} \\
 d_{33} &= I_3 + \gamma^2 l_3^2 m_3 \\
 d_{34} &= 0 = d_{43} \\
 d_{35} &= 0 = d_{53} \\
 d_{36} &= 0 = d_{63} \\
 d_{44} &= I_2 + l_2^2 [m_1 + (1 - \beta)^2 m_2 + m_6] \\
 d_{45} &= l_1 l_2 [(1 - \alpha)m_1 + m_6] \cos(\theta_4 - \theta_5) = d_{54} \\
 d_{46} &= -l_2 l_6 m_6 \sigma \cos(\theta_4 - \theta_6) = d_{64} \\
 d_{55} &= I_1 + l_1^2 [m_1 (1 - \alpha)^2 + m_6] \\
 d_{56} &= -l_1 l_6 m_6 \sigma \cos(\theta_5 - \theta_6) = d_{65} \\
 d_{66} &= I_6 + l_6^2 m_6 \sigma^2
 \end{aligned}$$

- Centrifugal/coriolis matrix $C(\mathbf{q}, \dot{\mathbf{q}})$:

$$\begin{aligned}
 c_{11} &= 0 = c_{22} = c_{33} = c_{44} = c_{55} = c_{66} \\
 c_{12} &= l_1 l_2 [m_1 + (1 + \beta)m_2 + m_3 + m_6] \sin(\theta_1 - \theta_2) \dot{\theta}_2 \\
 c_{13} &= l_1 l_3 \gamma m_3 \sin(\theta_1 - \theta_3) \dot{\theta}_3 \\
 c_{14} &= -l_1 l_2 [m_1 + (1 - \beta)m_2 + m_6] \sin(\theta_1 - \theta_4) \dot{\theta}_4 \\
 c_{15} &= -l_1^2 [(1 - \alpha)m_1 + m_6] \sin(\theta_1 - \theta_5) \dot{\theta}_5 \\
 c_{16} &= l_1 l_6 m_6 \sigma \sin(\theta_1 - \theta_6) \dot{\theta}_6 \\
 c_{21} &= -l_1 l_2 [m_1 + (1 + \beta)m_2 + m_3 + m_6] \sin(\theta_1 - \theta_2) \dot{\theta}_1 \\
 c_{23} &= l_2 l_3 \gamma m_3 \sin(\theta_2 - \theta_3) \dot{\theta}_3 \\
 c_{24} &= -l_2^2 [m_1 + (1 - \beta)m_2 + m_6] \sin(\theta_2 - \theta_4) \dot{\theta}_4
 \end{aligned}$$

$$\begin{aligned}
c_{25} &= -l_1 l_2 [(1 - \alpha) m_1 + m_6] \sin(\theta_2 - \theta_5) \dot{\theta}_5 \\
c_{26} &= l_2 l_6 m_6 \sigma \sin(\theta_2 - \theta_6) \dot{\theta}_6 \\
c_{31} &= -l_1 l_3 \gamma m_3 \sin(\theta_1 - \theta_3) \dot{\theta}_1 \\
c_{32} &= -l_2 l_3 \gamma m_3 \sin(\theta_2 - \theta_3) \dot{\theta}_2 \\
c_{34} &= 0 = c_{35} = c_{43} = c_{53} = c_{63} = c_{36} \\
c_{41} &= l_1 l_2 [m_1 + (1 - \beta) m_2 + m_6] \sin(\theta_1 - \theta_4) \dot{\theta}_1 \\
c_{42} &= l_2^2 [m_1 + (1 - \beta) m_2 + m_6] \sin(\theta_2 - \theta_4) \dot{\theta}_2 \\
c_{45} &= l_1 l_2 [(1 - \alpha) m_1 + m_6] \sin(\theta_4 - \theta_5) \dot{\theta}_5 \\
c_{46} &= -l_2 l_6 m_6 \sigma \sin(\theta_4 - \theta_6) \dot{\theta}_6 \\
c_{51} &= l_1^2 [(1 - \alpha) m_1 + m_6] \sin(\theta_1 - \theta_5) \dot{\theta}_1 \\
c_{52} &= l_1 l_2 [(1 - \alpha) m_1 + m_6] \sin(\theta_2 - \theta_5) \dot{\theta}_2 \\
c_{54} &= -l_1 l_2 [(1 - \alpha) m_1 + m_6] \sin(\theta_4 - \theta_5) \dot{\theta}_4 \\
c_{56} &= -l_1 l_6 m_6 \sigma \sin(\theta_5 - \theta_6) \dot{\theta}_6 \\
c_{61} &= -l_1 l_6 m_6 \sigma \sin(\theta_1 - \theta_6) \dot{\theta}_1 \\
c_{62} &= -l_2 l_6 m_6 \sigma \sin(\theta_2 - \theta_6) \dot{\theta}_2 \\
c_{64} &= l_2 l_6 m_6 \sigma \sin(\theta_4 - \theta_6) \dot{\theta}_4 \\
c_{65} &= l_1 l_6 m_6 \sigma \sin(\theta_5 - \theta_6) \dot{\theta}_5
\end{aligned}$$

- Gravitational torque vector $G(\mathbf{q})$:

$$\begin{aligned}
g_1 &= [(\alpha + 1) m_1 + 2m_2 + m_3 + m_6] g l_1 \cos \theta_1 \\
g_2 &= [m_1 + (\beta + 1) m_2 + m_3 + m_6] g l_2 \cos \theta_2 \\
g_3 &= \gamma m_3 g l_3 \cos \theta_3 \\
g_4 &= [-m_1 + (\beta - 1) m_2 - m_6] g l_2 \cos \theta_4 \\
g_5 &= [(\alpha - 1) m_1 - m_6] g l_1 \cos \theta_5 \\
g_6 &= g l_6 m_6 \sigma \cos \theta_6
\end{aligned}$$

Appendix C

Details of the Electronics

C.1 Joint Micro-controller Board

Figure C.1 gives a detailed overview of the micro-controller board which is provided for each modular unit. This micro-controller board regulates muscle pressure with the bang-bang control structure. Furthermore, it handles sensory inputs originating from two pressure sensors and an encoder, and provides a buffered interface between the central PC and the local micro-controller. The same board architecture is also used for an extra micro-controller, which handles additional sensory information such as absolute robot position, supply pressure conditions, ground reaction forces and control of the treadmill.

The core of the joint controller board is the MC68HC916Y3 micro-controller of Motorola. It has a 16 bit central processor unit, CPU, and a separate processor, TPU, which is designed to handle sensory input and control output without disturbing the CPU.

The micro-controller unit can be debugged and programmed via the serial SDI interface which is a commercially available device. A 10 pin connector is provided to link the essential pins to the SDI debugger module. This interface has only been used during the development of the micro-controller board. Currently, the micro-controllers are programmed via the 16-bit communication interface.

This interface is created with a dual ported RAM unit. This unit provides a buffered structure which communicates with the Cypress micro-controller communication interface board (see C.7). Two dual ported ram chips IDT7130SA (8 bit wide) are used to create the 16-bit parallel bus interface. Each chip has 1Kbyte of memory, the first chip is used to store the lowest byte of the 16-bit data, while the other stores the highest byte. The memory is physically divided into a read data block and a write data block by connecting the R/\overline{W} signal to address line number 8 of the dual ported RAM memory. The highest address line is not used, which means that two memory storage places are provided for 256 16bit wide data. Due to the divided structure into a read and write block, it is never possible to access a memory place from both sides simultaneously, therefore the BUSY and INT pins of the dual ported RAM units are not used.

The connector to the USB interface board redirects the pins of port PF which can be used to generate interrupts on the CPU (MC68HC916Y3) and give acknowledge signals to the communication master. E.g. the Cypress USB micro-controller, which is the communication master controlling the communication sampling rate, generates an interrupt on the CPU of all the Motorola micro-controllers each communication sample. Furthermore these pins are used to reset all the Motorola CPUs and in the other direction, to acknowledge to the communication master that the specific Motorola CPU is ready for a read or write action.

One connector is provided for the interface to the sensors and the valves. These valves are controlled by several TPU signals. The micro-controller board provides 6 separate signals to control the 6 valves of a valve island, but currently only 4 of them are used since three exhaust valves are switched together. The 3 incremental encoder channels are also connected with the TPU, which presents a position signal to the CPU without demanding any processor time. Additionally, one of the two main channel of the encoder are linked with a secondary TPU pin in order to estimate angular joint rotation speed. This speed is determined by a time measurement between two neighboring encoder flanks. The 12-bit digital signals of the two pressure sensors are linked to the micro-controller via the serial SPI interface. Finally, port G is connected with 8 LEDs which are used to visualize the different operation modes of the robot.

Resetting the controller can be done by a local button on the micro-controller board or by the USB micro-controller via the dual ported RAM units. The local reset and micro-controller initialization scheme uses an AND-port (chip 4023) structure as clearly explained in the data sheets. Furthermore are provided an oscillation circuit to generate the clock for the CPU, two RS232 interfaces and a flash EEPROM programming circuit, all described in the data sheets.

The communication software is programmed into the flash EEPROM and works with two essential modes: program and run mode. These modes are selected by the first word of the communication data block, which come with 32bytes each sample. Program mode is selected to load the micro-controller with the specific low-level controller program, such as e.g. the bang-bang controller, and in the run mode this downloaded program is executed while exchanging necessary control data with the central PC. So there is no fixed controller design programmed in the controller but it is downloaded each time the robot is initialized. This creates a fast and flexible experimental low-level control board for which different controller strategies can be implemented easily.

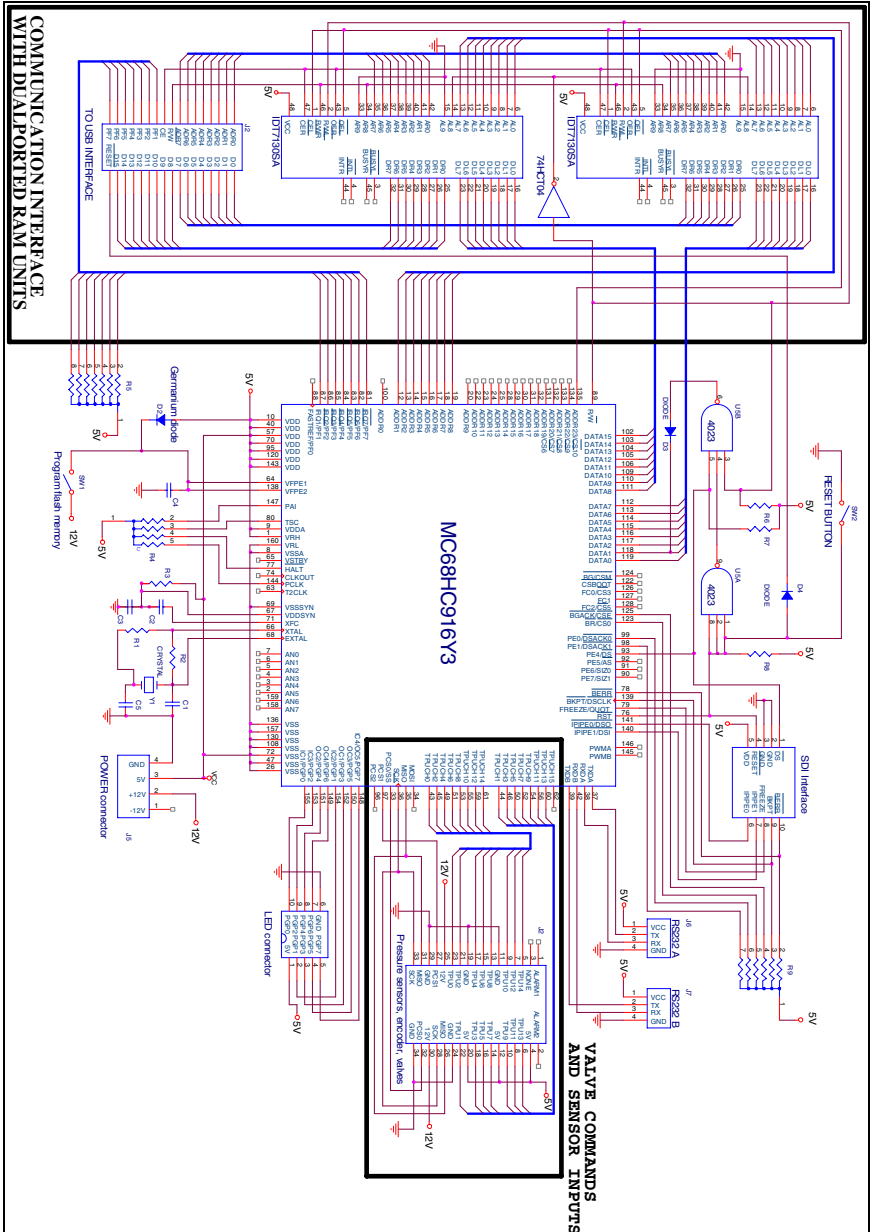


Fig. C.1 Electronic scheme of joint micro-controller board.

C.2 Speed-up Circuitry

In order to enhance the opening time of the Matrix valves, the manufacturer proposes a speed-up in tension circuitry. With a temporal $24V$ during a period of $2.5ms$ and a remaining $5V$ the opening time of the valves is said to be $1ms$. But during practical tests more than double values for the opening time were recorded. The opening tension is therefore increased to $36V$, but the time during which this voltage is applied is decreased to the actual opening time of $1ms$, such that the valves do not get overheated.

Figure C.2 gives the complete electronic scheme of the speed-up circuitry. Four identical schemes are provided, two for inlet and two for exhaust valves, of which one circuit commands three exhaust valves to open and close simultaneously. For each circuitry two LED's are provided in order to visualize valve action, one of them only lights up when the increased voltage is applied. These LEDs are important to check if the pressure control block is properly working. For each circuitry, the micro-controller commands a valve via discrete $5V$ on/off signals. These signals directly activate mosfet Q1 (IRF530) in order to apply $5V$ over the valve. The same signal passes parallel through a one-shot (74LS123) in order to increase the applied voltage over the valve during the first $1ms$ of valve activation. The output of the one-shot therefore temporally activates mosfet Q2 (IRF610) which on its turn commands mosfet Q3 (IRF9540) to branch the $36V$ supply to a valve. Whenever the micro-controller commands a valve to close, by disabling mosfet Q1, the discharge path is connected to the increased supply source via diode D2. This provides a fast discharge of the electromagnetic energy stored in the valve, which results in a faster closing time.

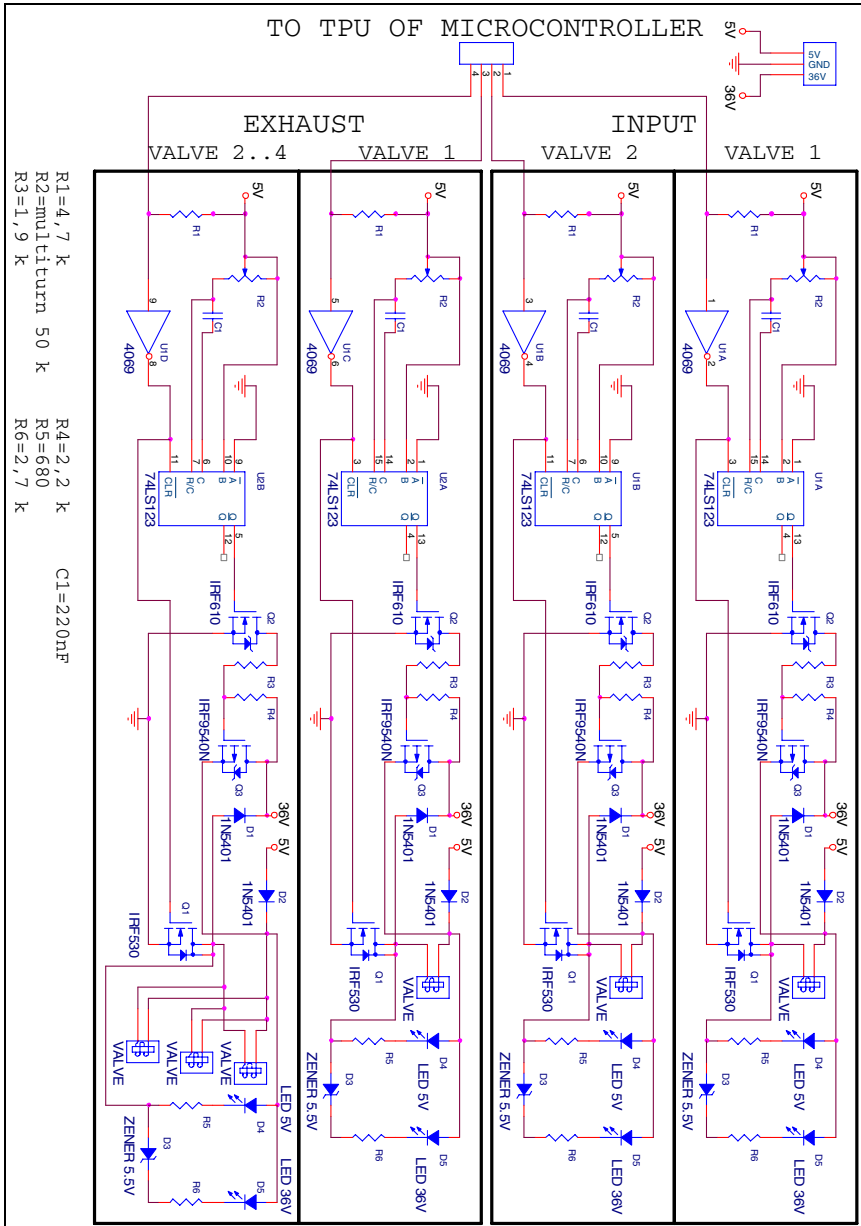


Fig. C.2 Electronic scheme of the speed-up circuitry.

C.3 Pressure Sensor

Figure C.3 depicts the electronic scheme which conditions the pressure sensor signal. The most important component is the absolute pressure sensor, CPC100AFC, from Honeywell. This sensor measures absolute pressure values up to 100psi (6.9bar) and has an accuracy of about 20mbar . Approximately 100mV for each 100psi is generated, meaning 14.5mV for 1bar . The output of the pressure sensor is amplified by a differential amplifier. The gain of this amplifier is approximately 63.2 . In order to avoid as much as possible noise generation, the amplified pressure signal (V_0) is immediately digitized by a 12bit analog to digital converter. A stable reference voltage for this converter is locally generated by a cascade circuit of two zener diodes. The negative input (-IN) of the AD-converter is augmented with a fixed voltage to roughly compensate atmospheric pressure. The AD-converter chip communicates with the micro-controller unit by a serial SPI interface, which is typically used for communication between chips and micro-controllers. A comparator *LM324* is provided to generate an alarm signal in order to protect the muscle against pressure overload. This signal is not treated by the micro-controller, but immediately acts on the central pressure supply valve (see 2.7.1.2). Whenever the muscle gauge pressure exceeds approximately 4.2bar , the pressure supply is cut-off. The pressure sensor circuit is calibrated each time the robot is initialized. This calibration is performed via an other pneumatic calibration circuit with an additional pressure sensor. In order to pass through the entrance of a muscle, the diameter of the sensor and its electronics is made small (12mm).

C.4 Foot Measurement Board

An electronic scheme of the foot measurement board is shown in figure C.4. The ground reaction forces are measured by load cells of Transducer Techniques (THA-250-Q). A commercial strain gauge amplifier of RS-components (846-171) is used to amplify the signal of the full bridge circuit. The chip requires also a negative supply voltage of $-10V$. To avoid the necessity of a new power supply a DC/DC-converter of Traco Power (TEN 10-2422) is used which makes out of $24V +/ - 12V$. The foot board has one such DC/DC-converter for the gauge amplifiers of the rear and front load cell. The amplified force signal is digitized by a 12 bit AD-converter (LTC 1286). A stable reference voltage for this converter is locally generated by zener diodes. The AD-converter chip communicates with the micro-controller unit by a serial SPI interface. Each foot has also a mechanical switch to detect if the foot

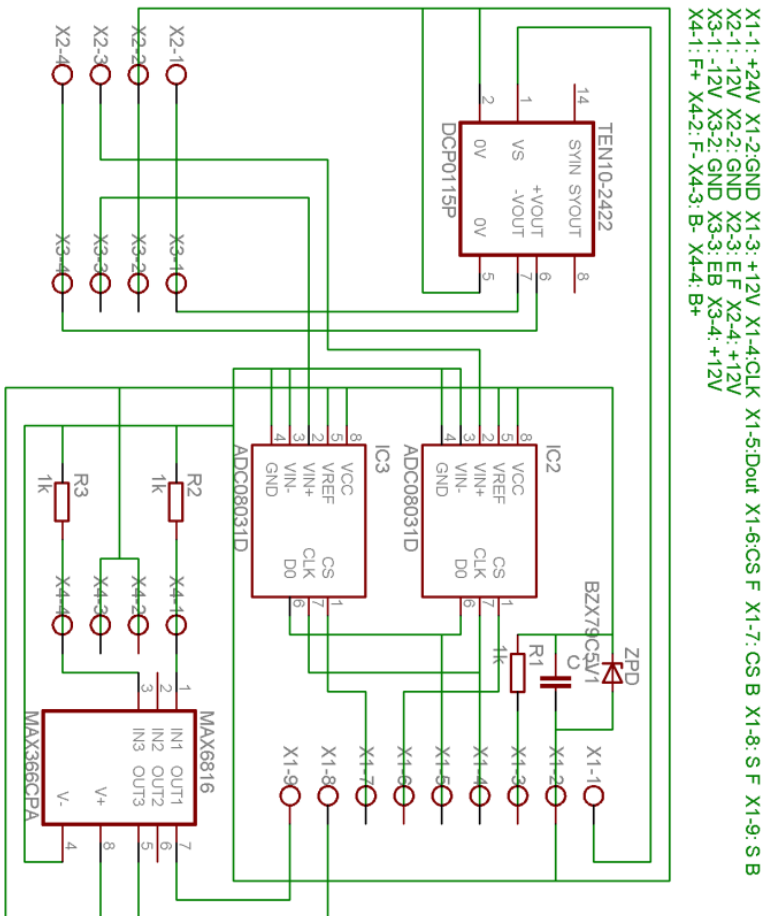


Fig. C.4 Electronic scheme of the foot measurement board.

is on the ground or not. To provide a clean interface with the digital system switch debouncers (MAX6816) are used.

C.5 Treadmill Control Board

The treadmill is powered by a 3 phase synchronous AC Motor controlled by the frequency inverter ACS 350 from ABB. This motor drive contains a vector control to provide enough torque at low rotation speeds. The steering signal and the measured rotation speed of the motor are treated by a separate electrical board which can be seen in figure C.5 This board contains opto-couplers so in case of a fault like an overvoltage on one side, the other side is not corrupted, especially to protect the low voltage electronics of the robot. This board is also connected to the emergency buttons: if an emergency button is pressed the treadmill stops automatically.

The treadmill control board (figure C.5) connects the 7th micro-controller with the frequency inverter ACS 350, which controls the motor of the treadmill. Essential are the opto-couplers 6N139 for galvanic separation of the robot electronics and the frequency inverter. A PWM signal coming from the TPU unit of the 7th micro-controller represents the speed signal. This signal is inverted (74HCT04) and feeds a LED and the opto-coupler 6N139. 4 emergency stops can be connected to this board. These signals are merged by OR-gates and if one of the emergency stops is pressed the power supply to the opto-coupler is turned off. After the opto-coupler a first order filter, formed by a capacity and resistor, makes an analogues signal between 0-10V for the frequency inverter. MC14538B is a monostable multivibrator. When there are no pulses the output is driven so the Darlington transistor is low and the frequency inverter is stopped. When pulses occur they are lengthened by the multivibrator so they form a continuous high signal and the frequency inverter is started. The drive has one programmable transistor output. In this application the frequency inverter is programmed to give the speed signal. Through an opto-coupler the signal is sent to the TPU unit of the 7th micro-controller. A manual switch makes it possible to reverse the rotation speed.

The part of the frequency inverter is fed by the frequency inverter itself. The board is designed so it consumes less than 200mA, the maximum current it can supply.

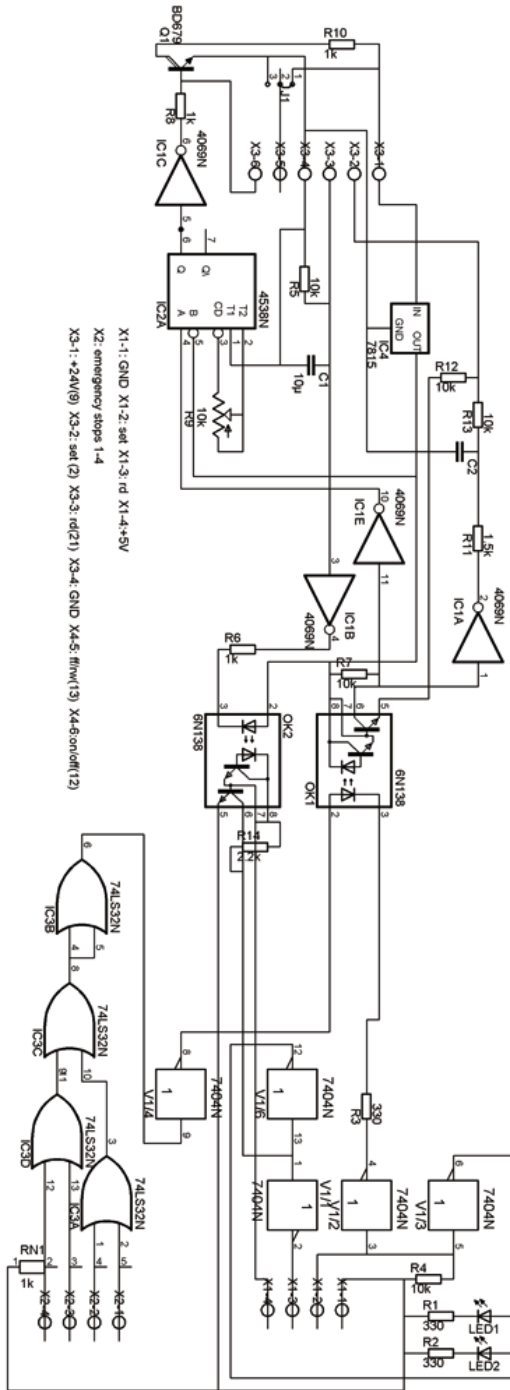


Fig. C.5 Electronic scheme of the treadmill control board.

C.6 Safety Board

The safety board is provided in order to control the supply pressure flow. It will cut-off the supply pressure in case an emergency situation is met. It can also select a lower calibration supply pressure required for the calibration of the 12 muscle pressure sensors. Figure C.6 shows the electronic scheme of the safety board. There are three valves which control the supply pressure. Opening valve 1 connects the robot to the high supply pressure and valve 2 introduces a lowered calibration pressure. Both valves are activated by a transistor circuit for which signals S1 or S2 have to be logic zero in order to open valve 1 or valve 2 respectively. If these signals are high, than valve 3 is opened in order to depressurize the robot. This happens when the robot is not working or when a pressure alarm or emergency stop is activated. A pressure alarm is induced by the pressure sensors in the muscles, whenever the pressure exceeds approximately $4.5bar$ gauge pressure. In this case a rising flank on the alarm signal switches the output of a D flip-flop to low logic state. The flip-flop is used to remember this emergency state and close the pressure valve until a manual reset is given on the safety board. All alarm signals have their own flip-flop structure with an additional LED such that is can be easily detected in which muscle the alarm signal was generated. An OR structure on all the flip-flop outputs in combination with 4 mechanical emergency stops depressurizes the robot whenever one of them alerts for a dangerous situation. Selection between the high or initialization pressure is done by two external signals, which are commanded by the extra micro-controller.

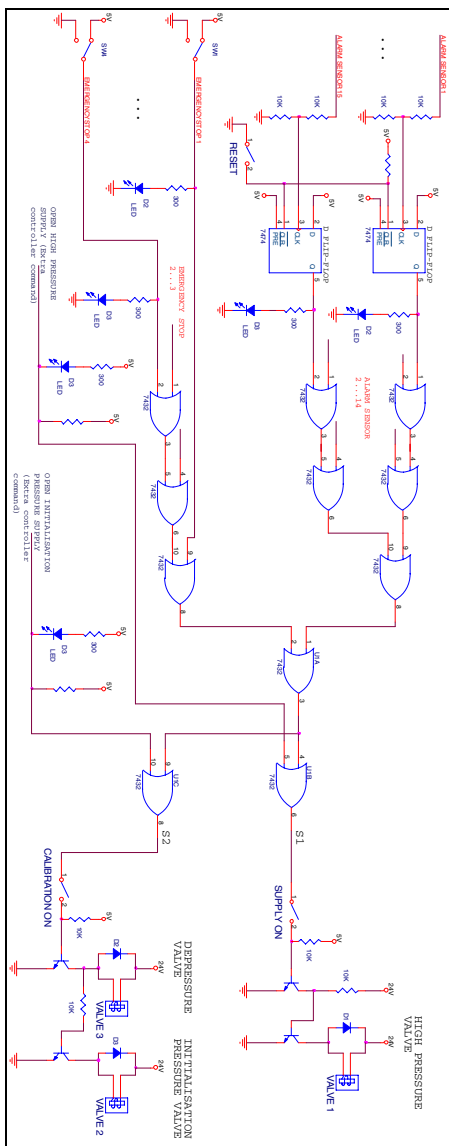


Fig. C.6 Electronic scheme of the safety board.

C.7 Cypress Communication Interface

C.7.1 Why USB 2.0?

Since a lot of extensive calculations are required due to the model based control algorithms, a central PC is used. Therefore a fast communication line between PC and robot hardware is provided. A fast communication line could be an extension of the PC bus by means of a parallel data communication, but this kind of communication is only suitable for short distance applications. For larger distances (several meters) serial communication protocols are preferable. This section will deal about the electronics needed to make the connection and the design choices made to handle the communication between the central PC and the 7 micro-controllers. This section will not deal about how the USB protocol works. For more information (25; 35; 28) is suggested.

Table C.1 shows an overview of some common interfaces.

RS-232 is not fast enough, FireWire is limited too much in allowable distance. For this application it was chosen to use a USB 2.0 communication interface, which has a data transfer rate of 480Mbit/s.

Other advantages for choosing an USB protocol are:

- A USB device can be plugged in anytime, even while the PC is turned on.
- When the PC detects that a USB device has been plugged in, it automatically interrogates the device to learn its capabilities and requirements. From this information, the PC automatically loads the devices driver into the operating system. When the device is unplugged, the operating system automatically logs it off and unloads its driver.
- USB devices do not use DIP switches, jumpers, or configuration programs. There is never an IRQ, DMA, memory, or I/O conflict with a USB device. USB expansion hubs make the bus simultaneously available to dozens of devices.
- Single connector type, the USB defines a single connector used to attach any USB device. Additional connectors can be added with USB hubs.

Table C.1 Comparison between different interfaces

Interface	Type	max # peripherals	transmission speed	max distance
USB	serial	127	USB 1.0: 1.5Mbits/s, low speed USB 1.1: 12Mbits/s, full speed USB 2.0: 480Mbits/s, high speed	5m (with hubs up to 30m)
RS-232	serial	2	20 - 115kbit/s	15-20m
IEEE-1394 (FireWire)	serial	64	400Mbit/s IEEE-1394b: 3.2Gbit/s	4m
Ethernet	serial	1024	10Mbit/s 1Gbit/s 100Mbit/s	4m

C.7.2 EZ-USB FX2

Since the local Motorola controllers (6 joint controllers+1 extra controller) have a 16bit parallel communication bus via the dual ported RAM units, the serial USB bulk communication data blocks have to be divided into 7 blocks of 16bit parallel data. Therefore an extra micro-controller, EZ-USB FX2 from Cypress Semiconductors, is provided to act only as data transfer agent. The main reasons to choose the EZ-USB FX2 were:

- An integrated, high-performance CPU based on the industry-standard 8051 processor.
- A soft (RAM-based) architecture that allows unlimited configuration and upgrades. Full USB throughput. USB devices that use EZ-USB chips are not limited by number of endpoints, buffer sizes, or transfer speeds.
- Automatic handling of most of the USB protocol.
- No external power supply, The 3.3V power supply can be delivered by the 5V power available at the USB connector (which the USB Specification allows to be as low as 4.4V).

This controller runs at 48Mhz and is able to transfer the serial data block of 226bytes to the peripheral 16bit data bus in less than 50μs. Additional to the Cypress development board, an electronic interface has been created to connect the peripheral bus of the Cypress micro-controller to the different dual ported RAM units. Figure C.7 gives the electronic scheme of the interface. Since the Cypress controller works at 3V supply voltage level and the dual ported RAM units at 5V, all lines connecting both parts are buffered via octal supply translating transceiver chips, 74LVC4245. These have a tristate when not enabled, this is important especially for connecting the data lines FD[i] of the Cypress controller to data lines D[i] of the dual ported RAM units. Two chips, U1 and U2, are foreseen for the 16 bit data lines, which work in both directions. The address lines are buffered with U3 which only translates in one direction as is the same for chip U4. The latter connects port PE of the Cypress controller to the other micro-controllers in order to give communication commands. These are: selection of a specific dual ported RAM unit by means of the line decoder chip U6, directing the R/W signal, global reset by software via pin PE5 and two extra general purpose control pins connected to PF1 and PF2 of the Motorola controller. These PF port pins can be controlled interrupt driven. In the other direction, pins PF3 of all the Motorola micro-controllers are connected separately to port PA of the Cypress controller. The pins PF4 of all Motorola controllers are connected together via an AND gate to pin PA7. These signals are used as communication acknowledgement signals, knowing that the Cypress controller is the bus master. Furthermore, a dip switch is provided to act on pin PF5 in order to select between two working modes. Finally, a general purpose interrupt can be generated manually on pin PF7 of all controllers and a manual global reset button is provided also.

C.7.3 USB Transfer Types

USB defines four transfer types. These match the requirements of different data types delivered over the bus.

- Bulk Transfers
- Interrupt Transfers
- Isochronous Transfers
- Control Transfers

The EZ-USB FX2 is configured as Bulk Transfer for the robot application. Bulk data is bursty, traveling in packets of 512bytes at high speed. The most important reason to choose for this type is that it has guaranteed accuracy, due to an automatic retry mechanism for erroneous data. The host schedules bulk packets when there is available bus time.

C.7.4 EZ-USB FX2 Architecture

The FX2 packs all the intelligence required by a USB peripheral interface into a compact integrated circuit. As figure C.8 illustrates, an integrated USB transceiver connects to the USB bus pins D+ and D-. A Serial Interface Engine (SIE) decodes and encodes the serial data and performs error correction, bit stuffing, and the other signaling-level tasks required by USB. Ultimately, the SIE transfers parallel data to and from the USB interface.

The high-level USB protocol is not bandwidth-critical, so the FX2s CPU is well-suited for handling host requests over the control endpoint. However, the data rates offered by USB 2.0 are too high for the CPU to process the USB data directly. For this reason, the CPU is not in the high bandwidth data path between endpoint FIFOs and the external interface. Instead, the CPU simply configures the interface, then “gets out of the way” while the unified FX2 FIFOs move the data directly between the USB and the external interface. To do this the General Programmable Interface (GPIF) is used. This internal FX2 timing generator serves as an internal master, interfacing directly to the FIFOs and generating user-programmed control signals for the interface to external logic.

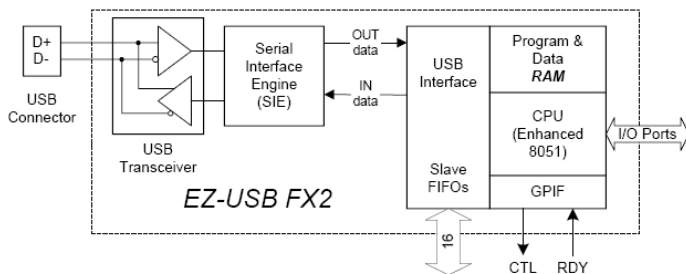


Fig. C.8 FX2 Simplified Block Diagram (28).

C.7.5 FX2 Endpoint Buffers

The USB Specification defines an endpoint as a source or sink of data. Since USB is a serial bus, a device endpoint is actually a FIFO which sequentially empties or fills with USB data bytes. The host selects a device endpoint by sending a 4-bit address and a direction bit. Therefore, USB can uniquely address 32 endpoints, IN0 through IN15 and OUT0 through OUT15.

From the FX2s point of view, an endpoint is a buffer full of bytes received or held for transmission over the bus. The FX2 reads host data from an OUT endpoint buffer, and writes data for transmission to the host to an IN endpoint buffer. The terms “IN” and “OUT” are from the viewpoint of the PC, because the PC is always the master in a USB system. USB devices respond to host requests. USB devices cannot send information among themselves, as they could if USB were a peer-to-peer topology.

EP0 is the default CONTROL endpoint, a bidirectional endpoint that uses a single 64-byte buffer for both IN and OUT data.

Endpoints 2, 4, 6 and 8 are the large, high bandwidth, data moving endpoints. In this application the endpoint 2 is used as OUT endpoint and 6 as IN endpoint.

C.7.6 Firmware

Because the FX2s configuration is soft, one chip can take on the identities of multiple distinct USB devices. The functionality of the controller, called the firmware, isn't stored in the memory of the chip itself, but on the central PC. This has an advantage that it is easily adapted. When first plugged into USB, the FX2 enumerates automatically and downloads firmware and USB descriptor tables over the USB cable. Next, the FX2 enumerates again, this time as a device defined by the downloaded information. This patented two-step process, called ReNumerationTM, happens instantly when the device is plugged in. The new programmed FX2 is visible in the “Device manager”.

C.7.7 Driver

To communicate between the FX2 and the PC the proper device driver has to be installed. A device driver is the software that Windows or a Windows based application uses to interact with a piece of hardware. As a result the application doesn't need to bother about protocols, physical connections, signals and can stay platform independent.

Because the FX2 is not a standard piece of hardware, a class driver can't be used. Examples of class drivers are the human interface device (HID) class, which supports devices like mice, joysticks, and keyboards. Another is the monitor class, which controls image position, size, and alignment on video displays. Custom drivers are an alternative to class drivers.

Because we didn't want to write our own driver with for example Driver Developer's Kit (DDK), the generic driver "WinRT for USB" of BSQUARE is used. Using this, developers can write application-level Win32 hardware control programs for USB hardware and eliminate the need for device driver toolkits or custom device driver development.

Communicating with the FX2 is then possible with easy to understand functions as for example "WinRTBulkTransfer" and "FindBulkEndpoints".

Appendix D

Publication List Related to Lucy

- Trajectory Planning for the Walking Biped 'Lucy'
VERMEULEN Jimmy, VERRELST Bjorn, VANDERBORGHT Bram, LEFEBER Dirk
International Journal of Robotics Research, Vol. 25, No. 9, 2006, pp. 867-887
- Controlling a Bipedal Walking Robot Actuated by Pleated Pneumatic Artificial Muscles
VANDERBORGHT Bram, VERRELST Bjorn, VAN HAM Ronald, LEFEBER Dirk
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- Second Generation Pleated Pneumatic Artificial Muscle and Its Robotic Applications
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Advanced Robotics, Vol. 20 No. 7, 2006, pp. 783-805
- Exploiting Natural Dynamics to Reduce Energy Consumption by Controlling the Compliance of Soft Actuators
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- A real-time joint trajectory planner for dynamic walking bipeds in the sagittal plane
VERMEULEN Jimmy, VERRELST Bjorn, LEFEBER Dirk, KOOL Patrick, VANDERBORGHT Bram
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VAN HAM Ronald, DAERDEN Frank, VERRELST Bjorn, VANDERBORGHT Bram, LEFEBER Dirk
International Journal of Fluid Power, No. 6, 2005, pp. 53-58
- Treadmill walking of the pneumatic biped Lucy: Walking at different speeds and step-lengths Authors: VANDERBORGHT Bram, VERRELST BJORN, VAN HAM Ronald, VAN DAMME Michael, VERSLUYS Rino, LEFEBER DIRK
Reference: International Applied Mechanics, vol. 44, n. 7, 2008, pp. 134 - 142
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- Development of a compliance controller to reduce energy consumption for bipedal robots Authors: VANDERBORGHT Bram, VERRELST BJORN, VAN HAM Ronald, VAN DAMME Michael, BEYL Pieter, LEFEBER DIRK
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- Comparison of Mechanical Design and Energy Consumption of Adaptable, Passive-compliant Actuators Authors: VANDERBORGHT Bram, VAN HAM Ronald, LEFEBER DIRK, Sugar Thomas, Hollander Kevin
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For a publication list including conference papers please visit

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