

# Appendix A

## Neural Controller Stability Analysis

Now, the stability analysis is required for the closed loop system in (2.5) controlled by the discrete-time N-SM or N-SML controllers. In order to analyze the proposed control algorithms stability, the separation principle in discrete-time is used [54], which divides the analysis in two parts [36].

### 1. Minimization of the identification error.

It can be achieved by on-line identification using EKF based algorithm as proved in Theorem 4.1 in [36]. As results of this theorem, the proposed RHONN model in (2.31) used to identify the class of system under study given in (2.5) ensures that the identification errors are semi globally uniformly ultimately bounded; moreover the RHONN weights remained bounded [36].

### 2. The tracking error convergence to a small bound region. It is achieved by the proposed neural controllers as will be demonstrated in the following analysis.

## A.1 N-SM Controller

Due to identification error boundedness, there exists a bounded vector valued function  $\Delta_{i,k}$ , which is smaller; then [76]

$$x_k = \chi_k + \Delta_{i,k} \quad i = 1, \dots, r \quad (\text{A.1})$$

with  $\|\Delta_{i,k}\| \leq \gamma_i$ ,  $\gamma_i > 0$  and  $r$  the number of the state. The sliding surface at  $k + 1$  is defined using  $x_k$  as follows

$$s_{k+1} = x_{k+1} - x_{ref,k+1} \quad (\text{A.2})$$

Using (4.1), the sliding surface in (A.2) is calculated as

$$s(x_k, k+1) = \hat{f}(x_k, k) + B(x_k)u(x_k, k) + \Delta(x_k, k) - x_{ref,k+1} \quad (\text{A.3})$$

and the equivalent control is determined as

$$u_{eq}(x_k, k) = -B(x_k)^{-1} \left[ \hat{f}x_k, k + \Delta(x_k, k) - x_{ref,k+1} \right] \quad (\text{A.4})$$

The discrete-time sliding mode controller proposed in (2.14) includes two cases:

- When  $\|u_{eqn}(x_k, k)\| \leq u_0$

The applied control law in this case is  $u_n(x_k, k) = u_{eqn}(x_k, k) + u_{sn}(x_k, k)$  then, by substituting  $u_n(x_k, k)$  in (A.3), the sliding surface is rewritten as

$$s(x_k, k+1) = k_c s(x_k, k) + \Delta(x_k, k) \quad (\text{A.5})$$

Let define the Lyapunov function candidate  $V(x_k, k) = s(x_k, k)^T s(x_k, k)$ , then

$$\begin{aligned} \Delta V(x_k, k) &= s(x_k, k+1)^T s(x_k, k+1) - s(x_k, k)^T s(x_k, k) \\ &= (k_c s(x_k, k) + \Delta(x_k, k))^T (k_c s(x_k, k) + \Delta(x_k, k)) - s(x_k, k)^T s(x_k, k) \\ &\leq (\|k_c\| \|s(x_k, k)\| + \|\Delta(x_k, k)\|)^2 - \|s(x_k, k)\|^2 \\ &\leq (\|k_c\| \|s(x_k, k)\| + \gamma)^2 - \|s(x_k, k)\|^2 \\ &\leq \|k_c\|^2 \|s(x_k, k)\|^2 + 2\|k_c\| \|s(x_k, k)\| \gamma + \gamma^2 - \|s(x_k, k)\|^2 \\ &\leq -(1 - \|k_c\|^2) \|s(x_k, k)\|^2 + 2\|k_c\| \|s(x_k, k)\| \gamma + \gamma^2 \\ &\leq -(1 - \theta_1)\eta_1 \|s(x_k, k)\|^2 - \theta_1\eta_1 \|s(x_k, k)\|^2 + 2\|k_c\| \|s(x_k, k)\| \gamma + \gamma^2 \\ &\leq -(1 - \theta_1)\eta_1 \|s(x_k, k)\|^2 + \gamma^2 + (-\theta_1\eta_1 \|s(x_k, k)\| + 2\|k_c\| \gamma) \|s(x_k, k)\| \end{aligned}$$

with  $\theta_1 < 1$ ,  $\eta_1 = (1 - \|k_c\|^2)$ ,  $\|k_c\| < 1$ , and in the region  $\|s(x_k, k)\| \geq \frac{2\gamma\|k_c\|}{\theta_1\eta_1}$ , we obtain

$$\begin{aligned} \Delta V(x_k, k) &\leq -(1 - \theta_1)\eta_1 \|s(x_k, k)\|^2 + \gamma^2 \\ \Delta V(x_k, k) &\leq -(1 - \theta_2)\eta_2 \|s(x_k, k)\|^2 - \theta_2\eta_2 \|s(x_k, k)\|^2 + \gamma^2 \\ \Delta V(x_k, k) &\leq -(1 - \theta_2)\eta_2 \|s(x_k, k)\|^2 \end{aligned}$$

with  $\theta_2 < 1$ ,  $\eta_2 = (1 - \theta_1)\eta_1$ .

So, for the region  $\|s(x_k, k)\| \geq \frac{\gamma}{\sqrt{\theta_2\beta_1}}$ , yielding  $\Delta V(x_k, k) \leq 0$  and the solution of system (A.5) is ultimately bounded.

- When  $\|u_{eqn}(x_k, k)\| > u_0$

The following control law is applied  $u_n(x_k, k) = u_0 \frac{u_{eqn}(x_k, k)}{\|u_{eqn}(x_k, k)\|}$ . By substituting this control law in (A.3), the sliding surface is rewritten as

$$\begin{aligned} u_{eq}(x_k, k) &= B(x_k)^{-1} \left[ s(x_k, k) - x_{ref,k} + x_k + \hat{f}(x_k, k) \right] \\ &= B(x_k)^{-1} \left[ s(x_k, k) + \hat{f}_s(x_k, k) \right] \end{aligned} \quad (\text{A.6})$$

with  $\hat{f}_s(x_k, k) = -s(x_k, k) + \hat{f}(x_k, k)$ , and the sliding surface is calculated as

$$\begin{aligned} s(x_k, k+1) &= \left[ s(x_k, k) - x_{ref,k} + x_k + \hat{f}(x_k, k) + B(x_k)u_c(x_k, k) + \Delta(x_k, k) \right] \\ &= \left[ s(x_k, k) + \hat{f}_s(x_k, k)B(x_k)u_c(x_k, k) + \Delta(x_k, k) \right] \end{aligned} \quad (\text{A.7})$$

Let define the Lyapunov function candidate  $V(x_k, k) = s(x_k, k)^T s(x_k, k)$ , then

$$\begin{aligned} \Delta V(x_k, k) &= s(x_k, k+1)^T s(x_k, k+1) - s(x_k, k)^T s(x_k, k) \\ &= \left[ s(x_k, k) + \hat{f}_s(x_k, k)B(x_k)u_c(x_k, k) + \Delta(x_k, k) \right] \\ &\quad \left[ s(x_k, k) + \hat{f}_s(x_k, k)B(x_k)u_c(x_k, k) + \Delta(x_k, k) \right]^T \\ &\quad - s(x_k, k)^T s(x_k, k) \\ &= \left[ \left( s(x_k, k) + \hat{f}_s(x_k, k) \right) \left( 1 - u_0 \frac{u_{eq}(x_k, k)}{\|u_{eq}(x_k, k)\|} \right) + \Delta(x_k, k) \right] \\ &\quad \left[ \left( s(x_k, k) + \hat{f}_s(x_k, k) \right) \left( 1 - u_0 \frac{u_{eq}(x_k, k)}{\|u_{eq}(x_k, k)\|} \right) + \Delta(x_k, k) \right]^T \\ &\quad - s(x_k, k)^T s(x_k, k) \\ &\leq \left[ \left\| s(x_k, k) + \hat{f}_s(x_k, k) \right\| \left( 1 - \frac{\|u_0\|}{\|u_{eq}(x_k, k)\|} \right) + \|\Delta(x_k, k)\| \right]^2 - \|s(x_k, k)\|^2 \\ &\leq \left[ \left\| s(x_k, k) + \hat{f}_s(x_k, k) \right\| - \frac{\|u_0\|}{\|B(x_k)^{-1}\|} + \|\Delta(x_k, k)\| \right]^2 - \|s(x_k, k)\|^2 \end{aligned}$$

Suppose that the control law  $\|u_n(x_k)\| \leq u_0$  may vary within the domain [77],  $\left\| \hat{f}_s(x_k, k) + \Delta(x_k, k) \right\| \leq u_0$ , and  $\sigma < u_0$  with  $\sigma = \|f_{n,k} + \Delta(x_k, k)\|$ , then

$$\begin{aligned} \Delta V(x_k, k) &\leq \left[ \|s(x_k, k) + \sigma\| - \frac{\|u_0\|}{\|B(x_k)^{-1}\|} \right]^2 - \|s(x_k, k)\|^2 \\ \Delta V(x_k, k) &\leq 2 \|s(x_k, k)\| \left( \sigma + \frac{\|u_0\|}{\|B(x_k)^{-1}\|} \right) - \left( \sigma + \frac{\|u_0\|}{\|B(x_k)^{-1}\|} \right)^2 \end{aligned}$$

if  $\|B(x_k)^{-1}\| \sigma \leq \|u_0\| \leq 2 \|s(x_k, k)\| + \sigma$  holds then  $\Delta V(x_k, k) \leq 0$ . Hence  $\|s(x_k, k)\|$  and  $\|U_{r,k}^{eq}\|$ , both decreases monotonically. Therefore there will be a time  $k_1$  such that  $\|u_{eqn}(x_k, k)\| \leq \|u_0\|$  for  $k \geq k_1$ . At that time, the control law  $u_n(x_k, k)$  is applied, yielding that the solution of the system (A.5) is ultimately bounded.  $\square$

## A.2 N-SML Controller

The DTSMC  $v_n(x_k)$  is selected as (4.39). The convergence analysis of the N-SML control scheme is divided on neural identifier convergence and N-SML controller convergence, taking into account the discrete-time separation principle [54]. For the first analysis, it is achieved by using an EKF based algorithm to train the RHONN identifier as demonstrated in Theorem 4.1 of [36]. The tracking error convergence to a small bound region. It is achieved by the proposed controller as will be demonstrated in the following analysis Due to identification error boundedness, there exists a bounded vector valued function  $\Delta_{i,k}$ , which is smaller; then

$$x_k = \chi_k - \Delta_{i,k} \quad i = 1, \dots, r \quad (\text{A.8})$$

with  $\|\Delta_{i,k}\| \leq \gamma_i$ ,  $\gamma_i > 0$  and  $r$  the number of the state. The sliding surface at  $k + 1$  is defined using  $x_k$  as follows

$$s_{n,k+1} = x_{ref,k+1} - x_{k+1} \quad (\text{A.9})$$

Using (A.8) in (A.9), we obtain

$$s_{n,k+1} = x_{ref,k+1} - \chi_{k+1} + \Delta_{i,k} \quad (\text{A.10})$$

with  $x_{ref,k+1}$  the desired dynamics to be tracked. Using the neural linearization control part (4.39), the sliding surface is expressed as follows

$$s_{n,k+1} = x_{ref,k+1} - v_n(x_k) + \Delta_{i,k} \quad (\text{A.11})$$

The DTSMC in (4.42) includes two cases

- When  $\|v_{cn}(x_k)\| \leq u_{0n}$ , the control signal  $v_{cn,k}$  is applied.

Using (4.43) and (4.45) in (A.10), the sliding surface at  $k + 1$  is

$$s_{n,k+1} = k_n s_{n,k} + \Delta_{i,k} \quad (\text{A.12})$$

Let define the Lyapunov function candidate  $V_k = s_{n,k}^T s_{n,k}$ , then its difference is given as

$$\begin{aligned}
\Delta V_k &= s_{n,k+1}^T s_{n,k+1} - s_{n,k}^T s_{n,k} \\
&= (k_n s_{n,k} + \Delta_{i,k})^T (k_n s_{n,k} + \Delta_{i,k}) - s_{n,k}^T s_{n,k} \\
&\leq (\|k_n\| \|s_{n,k}\| + \|\Delta_{i,k}\|)^2 - \|s_{n,k}\|^2 \\
&\leq (\|k_n\| \|s_{n,k}\| + \gamma_i)^2 - \|s_{n,k}\|^2 \\
&\leq \|k_n\|^2 \|s_{n,k}\|^2 + 2 \|k_n\| \|s_{n,k}\| \gamma_i + \gamma_i^2 - \|s_{n,k}\|^2 \\
&\leq -(1 - \|k_n\|^2) \|s_{n,k}\|^2 + 2 \|k_n\| \|s_{n,k}\| \gamma_i + \gamma_i^2 \\
&\leq -(1 - \theta_1) \eta_1 \|s_{n,k}\|^2 - \theta_1 \eta_1 \|s_{n,k}\|^2 \\
&\quad + 2 \|k_n\| \|s_{n,k}\| \gamma_i + \gamma_i^2 \\
&\leq -(1 - \theta_1) \eta_1 \|s_{n,k}\|^2 + \gamma_i^2 \\
&\quad + (-\theta_1 \eta_1 \|s_{n,k}\| + 2 \|k_n\| \gamma_i) \|s_{n,k}\|
\end{aligned}$$

with  $0 < \theta_1 < 1$ ,  $\eta_1 = (1 - \|k_n\|^2)$  and  $\eta_1 > 0$ , and for the region  $\|s_{n,k}\| \geq \frac{2\gamma_i \|k_n\|}{\theta_1 \eta_1}$ , we obtain

$$\begin{aligned}
\Delta V_k &\leq -(1 - \theta_1) \eta_1 \|s_{n,k}\|^2 + \gamma_i^2 \\
\Delta V_k &\leq -(1 - \theta_2) \beta_1 \|s_{n,k}\|^2 - \theta_2 \beta \|s_{n,k}\|^2 + \Gamma^2 \\
\Delta V_k &\leq -(1 - \theta_2) \beta_1 \|s_{n,k}\|^2
\end{aligned}$$

with  $0 < \theta_2 < 1$ ,  $\beta_1 = (1 - \theta_1) \eta_1$  and  $\beta_1 > 0$ . Therefore  $\Delta V_k \leq 0$ ,  $\forall \|s_{n,k}\| \geq \sqrt{\frac{\gamma_i^2}{\theta_2 \beta_1}}$ , and the solution of system (A.10) is ultimately bounded.

- When  $\|v_{cn}(x_k)\| > v_{0n}$ , the following control law  $u_0 \frac{v_{eqn}(x_k)}{\|v_{eqn}(x_k)\|}$  is applied.

Let us define the equivalent control by imposing  $s_{n,k} - x_{ref,k} + \chi_k = 0$ . The equivalent control is defined as

$$\begin{aligned}
v_{eqn,k} &= s_{n,k} - x_{ref,k} + \chi_k + x_{ref,k+1} \\
&= s_{n,k} + f_{n,k}
\end{aligned} \tag{A.13}$$

with  $f_{n,k} = -x_{ref,k} + \chi_k + x_{ref,k+1}$ . Then, the expression of the sliding mode surface is

$$\begin{aligned}
s_{n,k+1} &= s_{n,k} - x_{ref,k} + x_k + x_{ref,k+1} - v_n(x_k) + \Delta_{i,k} \\
&= s_{n,k} + f_{n,k} - \left( u_0 \frac{v_{eqn,k}}{\|v_{eqn,k}\|} \right) + \Delta_{i,k} \\
&= (s_{n,k} + f_{n,k}) \left( 1 - u_0 \frac{1}{\|v_{eqn,k}\|} \right) + \Delta_{i,k}
\end{aligned}$$

Using the Lyapunov function candidate  $V_k = s_{n,k}^T s_{n,k}$ , then

$$\begin{aligned}
\Delta V_k &= s_{n,k+1}^T s_{n,k+1} - s_{n,k}^T s_{n,k} \\
&\leq \left( \|s_{n,k} + f_{n,k}\| \left( 1 - \|u_0\| \frac{1}{\|v_{eqn,k}\|} \right) + \Delta_{i,k} \right)^T \\
&\quad \left( \|s_{n,k} + f_{n,k}\| \left( 1 - \|u_0\| \frac{1}{\|v_{eqn,k}\|} \right) + \Delta_{i,k} \right) \\
&\quad - \|s_{n,k}\|^2 \\
&\leq (\|s_{n,k} + f_{n,k}\| - \|u_0\| + \|\Delta_{i,k}\|)^2 - \|s_{n,k}\|^2
\end{aligned}$$

Suppose that the control law  $\|u_n(x_k)\| \leq u_{0n}$  may vary within the domain [77],  $\|f_{n,k} + \Delta_{i,k}\| \leq u_{0n}$ , and  $\sigma < u_{0n}$  with  $\sigma = \|f_{n,k} + \Delta_{i,k}\|$  then

$$\begin{aligned}
\Delta V_k &\leq (\|s_{n,k}\| + \sigma - \|u_{0n}\|)^2 - \|s_{n,k}\|^2 \\
&\leq (\|s_{n,k}\| + \sigma - \|u_0\| + \|s_{n,k}\|) \\
&\quad (\|s_{n,k}\| + \sigma - \|u_0\| - \|s_{n,k}\|) \\
&\leq (2\|s_{n,k}\| + \sigma - \|u_0\|) (\sigma - \|u_0\|) \\
&\leq - (2\|s_{n,k}\| + \sigma - \|u_0\|) (\|u_0\| - \sigma)
\end{aligned} \tag{A.14}$$

If  $\|f_{n,k} + \Delta_{i,k}\| \leq \|u_{0n}\| \leq (2\|s_{n,k}\| + \|f_{n,k} + \Delta_{i,k}\|)$  holds, then  $\Delta V_k \leq 0$  [36, 78]. Hence  $\|s_{n,k}\|$  and  $\|v_{eqn,k}\|$ , both decreases monotonically.

Therefore there will be a time  $k_1$  such that  $\|v_{eqn,k}\| \leq \|u_0\|$  for  $k \geq k_1$ . At that time, the control law  $v_{cn}$  is applied, yielding that the solution of the system (A.12) is ultimately bounded.  $\square$

# Appendix B

## The Wind Turbine Modeling

### B.1 Aerodynamic Model

Consider the horizontal axis wind system shown in Fig. B.1 on which  $v_1$  is upstream wind speed of the turbine and  $v_2$  is downstream wind speed. The kinetic energy of the mass air particle  $m$  that moves with a velocity is given as

$$E = \frac{1}{2}mv^2 \tag{B.1}$$

with  $E$  is the kinetic energy (J),  $m$  is the mass of the air particle (kg) and  $v$  is the speed (or the velocity) of the air particle (m/s), such that

$$m = \rho V \tag{B.2}$$

with  $\rho$  is the air density (kg/m<sup>3</sup>) and  $V$  is the the volume (m<sup>3</sup>). Taking into account the density of the air is constant, the relative change of the mass is calculated as follows

$$\dot{m} = \rho S \dot{x} \tag{B.3}$$

Knowing that the volume is obtained by multiplying the surface  $S$  by the length  $x$ . Only one dimension is considered because the amount of air varies along a single axis. Hence

$$\dot{m} = \rho S v \tag{B.4}$$

By assuming that the wind speed is constant, the power of the air movement will be calculated as

$$P_w = \dot{E} = \frac{1}{2} \rho S v^3 \tag{B.5}$$

Hence the absorbed power by the aerogenerator equals the power difference of downstream and upstream power

$$P_{aero} = P_{w1} - P_{w2} = \frac{1}{2}(\rho_1 S_1 v_1^3 - \rho_2 S_2 v_2^3) \quad (\text{B.6})$$

Taking  $\rho_1 = \rho_2$  and  $\dot{m}_1 = \dot{m}_2$ , we obtain

$$S_1 v_1 = S_2 v_2 \quad (\text{B.7})$$

Substituting (B.7) in (B.6), the absorbed aerogenerator power is expressed as follow

$$P_{aero} = \frac{1}{2} \rho_1 S_1 v_1 (v_1^2 - v_2^2) \quad (\text{B.8})$$

In other hand and according to Newton's second law

$$F = \frac{\partial}{\partial t}(mv) = \frac{\partial m}{\partial t}v + m \frac{\partial v}{\partial t} \quad (\text{B.9})$$

Considering the wind speed is constant and the mass quantity is variable, the force relation is rewritten as

$$F = \dot{m}v \quad (\text{B.10})$$

Substituting (B.4) in (B.10), the applied force  $F$  on the wind turbine blade is expressed as

$$F = \rho S v^2 \quad (\text{B.11})$$

The force variation between the two sides of the aerogenerator multiplied by the wind speed result the expression of absorbed power which is expressed by the following relation

$$P_{aero} = \Delta F v = \rho S_1 v_1 (v_1 - v_2)v \quad (\text{B.12})$$

Note that the power relationship is defined by the wind speed before and after the turbine and another wind speed named  $v$ . To define the value of the last one, we have to compare the two relationships of the wind turbine absorbed power in (B.8) and (B.12), we obtain

$$v = \frac{1}{2}(v_1 + v_2) \quad (\text{B.13})$$

For HAWT, the air flow at wind turbine is presented in Fig. 3.4, with  $v_1$  is the upstream wind speed,  $v_2$  is the downstream wind speed,  $v$  is the speed of the wind passing through the aerogenerator,  $S_1$  and  $S_2$  are the upstream and the downstream sections of the air tube,  $S$  is the surface swept by the blade. The absorbed power by the aerogenerator in case of HAWT is calculated as follow



$$P_{aero} = \frac{1}{2} \rho S v (v_1^2 - v_2^2) \quad (\text{B.14})$$

Using (B.13) in (B.14), we get

$$P_{aero} = \frac{1}{4} \rho S (v_1 + v_2) (v_1^2 - v_2^2) \quad (\text{B.15})$$

and the wind power is given by

$$P_w = \frac{1}{2} \rho S v_1^3 \quad (\text{B.16})$$

The ratio between the extracted power from the wind and the available total power available is

$$C_p = \frac{P_{aero}}{P_w} = \frac{1}{2} \left| 1 - \left( \frac{v_2}{v_1} \right)^2 \right| \left| 1 + \left( \frac{v_2}{v_1} \right)^2 \right| \quad (\text{B.17})$$

Let define the relative speed  $\lambda$  as

$$\lambda = \frac{v_2}{v_1} = \frac{\Omega_t r}{v} \quad (\text{B.18})$$

with  $\rho$  is the air density,  $v$  is the wind speed,  $\Omega_t$  is the rotational speed of the turbine (low shaft), and  $r$  is the radius of a blade. So the expression of the extracted power for the wind is defined as a function of the power coefficient  $C_p$  as follows

$$P_{aero} = \frac{1}{2} C_p \rho S v^3 \quad (\text{B.19})$$

The power coefficient  $C_p$  represents the yield aerodynamics of the WT, which depends on the turbine characteristics defined by the following empirical formula for variable speed wind turbines [57]

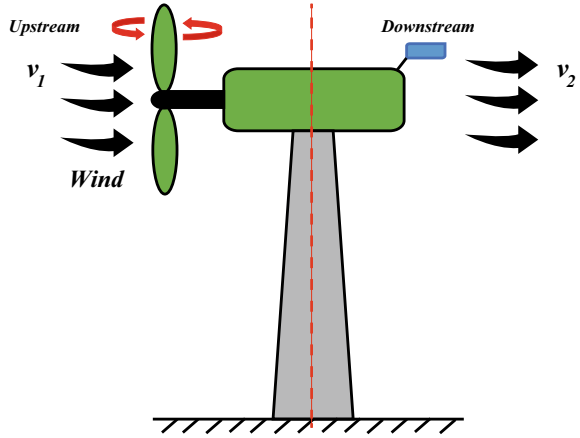
$$C_p(\lambda, \beta) = c_1 \left( c_2 \frac{1}{\Lambda} - c_3 \beta^x - c_4 - c_5 \right) \exp \left( \frac{-c_6}{\Lambda} \right) \quad (\text{B.20})$$

with  $c_1 = 0.5$ ;  $c_2 = 116$ ;  $c_3 = 0.4$ ;  $c_4 = 0$ ;  $c_5 = 5$ ;  $c_6 = 21$  and  $\frac{1}{\Lambda} = \frac{1}{\lambda + 0.08\beta} - \frac{0.035}{1 + \beta^3}$  where  $\lambda$  is the relative speed,  $\beta$  is the blade angle.

## B.2 DC-Link Model

In order to determine the mathematical model of the GSC, balanced grid conditions are considered. The three-phase model of the GSC is written as

**Fig. B.1** HAWT simplified scheme



$$u_{gca,t} - u_{ga,t} = -r_g i_{ga,t} + \frac{d\phi_{ga,t}}{dt} \quad (\text{B.21})$$

$$u_{gcb,t} - u_{gb,t} = -r_g i_{gb,t} + \frac{d\phi_{gb,t}}{dt} \quad (\text{B.22})$$

$$u_{gcc,t} - u_{gc,t} = -r_g i_{gc,t} + \frac{d\phi_{gc,t}}{dt} \quad (\text{B.23})$$

and

$$\phi_{ga,t} = l_g i_{ga,t} \quad (\text{B.24})$$

$$\phi_{gb,t} = l_g i_{gb,t} \quad (\text{B.25})$$

$$\phi_{gc,t} = l_g i_{gc,t} \quad (\text{B.26})$$

where  $i_{g,t}^{abc}$  are the three-phase grid currents (A),  $u_{g,t}^{abc}$  are the three phase grid voltages (V),  $u_{gc,t}^{abc}$  are the three phase GSC voltages (V), which are present the control input of the DC-link voltage,  $r_g$  is the grid line resistance ( $\Omega$ ) and  $l_g$  is the grid inductance (H).

Substituting (B.21)–(B.23) in (B.24)–(B.26), the GSC model can be rewritten as

$$u_{gca,t} - u_{ga,t} = r_g i_{ga,t} + l_g \frac{di_{ga,t}}{dt} \quad (\text{B.27})$$

$$u_{gcb,t} - u_{gb,t} = r_g i_{gb,t} + \frac{di_{gb,t}}{dt} \quad (\text{B.28})$$

$$u_{gcc,t} - u_{gc,t} = r_g i_{gc,t} + \frac{di_{gc,t}}{dt} \quad (\text{B.29})$$

$$l_g \frac{di_{g,t}^{abc}}{dt} = -r_g i_{ga,t}^{abc} - u_{g,t}^{abc} + u_{gc,t}^{abc} \quad (\text{B.30})$$

The GSC converter can be expressed in  $d - q$  by using the following transformation

$$K_g^{-1} = \frac{3}{2} \begin{bmatrix} \cos(\theta_g) & -\sin(\theta_g) & 1 \\ \cos(\theta_g - \frac{2\pi}{3}) & -\sin(\theta_g - \frac{2\pi}{3}) & 1 \\ \cos(\theta_g + \frac{2\pi}{3}) & \sin(\theta_g + \frac{2\pi}{3}) & 1 \end{bmatrix} \quad (\text{B.31})$$

with  $\theta_g$  is the grid angular position. Now, Applying the Park transformation to (B.30), we get

$$l_g \frac{d}{dt} K_g^{-1} i_{g,t}^{abc} = -r_g K_g^{-1} i_{ga,t}^{abc} - K_g^{-1} u_{g,t}^{abc} + K_g^{-1} u_{gc,t}^{abc} \quad (\text{B.32})$$

then,

$$l_g \frac{d}{dt} i_{g,t}^{dq0} + l_g i_{g,t}^{dq0} \left( (K_g) \frac{d}{dt} (K_g^{-1}) \right) = -r_g i_{ga,t}^{dq0} - u_{g,t}^{dq0} + u_{gc,t}^{dq0} \quad (\text{B.33})$$

such that  $((K_g) \frac{d}{dt} (K_g^{-1})) = W_g = \begin{bmatrix} 0 & -\omega_g^s \\ 0 & \omega_g & 0 \\ 0 & 0 & 0 \end{bmatrix}$ , Hence, the GSC in  $d - q$  reference frame is obtained as

$$\frac{d}{dt} i_{g,t}^{dq0} = -\frac{r_g}{l_g} i_{g,t}^{dq0} - W_g i_{g,t}^{dq0} - u_{g,t}^{dq0} + u_{gc,t}^{dq0} \quad (\text{B.34})$$

where  $di_{gd,t}$ ,  $di_{gq,t}$  are the  $d - q$  component of the grid current,  $u_{gd,t}$ ,  $u_{gq,t}$  are the grid voltage  $d - q$  component, and  $u_{gdc,t} - u_{gqc,t}$  are the GSC voltage  $d - q$  component. Neglecting the harmonics distortion due by the electronic switches, the GSC and electrical transformer losses, and the power balance between the ac and dc sides of the GSC, the DC voltage at the output of the DC link can be determined as [12]

$$\frac{dU_{dc,t}}{dt} = \frac{3}{2c_g U_{dc,t}} (i_{gd,t} u_{gd,t} + i_{gq,t} u_{gq,t}) \quad (\text{B.35})$$

### B.3 DFIG Model

The three phase stator and rotor mathematical model of the DFIG is described as

- The DFIG Stator Equations

$$\frac{d\phi_{s,t}^{abc}}{dt} = u_{s,t}^{abc} - r_s i_{s,t}^{abc} \quad (\text{B.36})$$

$$\phi_{s,t}^{abc} = L_{ss} i_{s,t}^{abc} + M_{sr} i_{r,t}^{abc} \quad (\text{B.37})$$

### • The DFIG Rotor Equations

$$\frac{d\phi_{r,t}^{abc}}{dt} = u_{r,t}^{abc} - r_r i_{r,t}^{abc} \quad (\text{B.38})$$

$$\phi_{r,t}^{abc} = L_{rr} i_{r,t}^{abc} + M_{rs} i_{s,t}^{abc} \quad (\text{B.39})$$

with

$$L_{ss} = \begin{bmatrix} L_{ls} + L_{ss} & M_s & M_s \\ M_s & L_{ls} + L_{ss} & M_s \\ M_s & M_s & L_{ls} + L_{ss} \end{bmatrix}, \quad L_{rr} = \begin{bmatrix} L_{lr} + L_{rr} & M_r & M_r \\ M_r & L_{lr} + L_{rr} & M_r \\ M_r & M_r & L_{lr} + L_{rr} \end{bmatrix},$$

$$M_{sr} = M_{rs}^T = M_{sr} \begin{bmatrix} \cos(\theta) & \cos(\theta + \frac{2\pi}{3}) & \cos(\theta - \frac{2\pi}{3}) \\ \cos(\theta - \frac{2\pi}{3}) & \cos(\theta) & \cos(\theta + \frac{2\pi}{3}) \\ \cos(\theta + \frac{2\pi}{3}) & \cos(\theta - \frac{2\pi}{3}) & \cos(\theta) \end{bmatrix},$$

with  $u_{s,t}^{abc}$  are the three-phase stator voltages (V),  $i_{s,t}^{abc}$  are the three-phase stator currents (A),  $\phi_{s,t}^{abc}$  are the three-phase stator flux (Wb),  $u_{r,t}^{abc}$  are the three-phase rotor voltages (V),  $i_{r,t}^{abc}$  are the three-phase rotor currents (A),  $\phi_{r,t}^{abc}$  are the three-phase rotor flux (Wb),  $r_s$  is the stator winding per phase leakage resistance ( $\Omega$ ),  $r_r$  is the rotor winding per phase leakage resistance ( $\Omega$ ),  $L_{ls}$  is the stator winding per phase leakage inductance,  $L_{ss}$  is the stator self inductance,  $L_{lr}$  is the rotor winding per phase leakage inductance,  $L_{rr}$  is the rotor self inductance,  $M_{sr}$  is the stator to rotor mutual inductance,  $M_{rr}$  is the rotor to stator mutual inductance,  $M_s$  is the stator mutual inductance, and  $M_r$  is the rotor mutual inductance.

The DFIG stator variables can be expressed in  $d - q$  reference frame by using the following transformation

$$K_s^{-1} = \frac{3}{2} \begin{bmatrix} \cos(\theta_s) & -\sin(\theta_s) & 1 \\ \cos(\theta_s - \frac{2\pi}{3}) & -\sin(\theta_s - \frac{2\pi}{3}) & 1 \\ \cos(\theta_s + \frac{2\pi}{3}) & \sin(\theta_s + \frac{2\pi}{3}) & 1 \end{bmatrix} \quad (\text{B.40})$$

$$\frac{d}{dt} K_s^{-1} \phi_{s,t}^{dq0} = K_s^{-1} u_{s,t}^{dq0} - r_s K_s^{-1} i_{s,t}^{dq0} \quad (\text{B.41})$$

$$K_s^{-1} \phi_{s,t}^{dq0} = K_s^{-1} L_{ss} i_{s,t}^{dq0} + K_s^{-1} M_{sr} i_{r,t}^{dq0} \quad (\text{B.42})$$

then,

$$\frac{d}{dt} \phi_{s,t}^{dq0} = u_{s,t}^{dq0} - r_s i_{s,t}^{dq0} - \left( K_s \frac{d}{dt} K_s^{-1} \right) \phi_{s,t}^{dq0} \quad (\text{B.43})$$

$$\phi_{s,t}^{dq0} = K_s L_{ss} i_{s,t}^{dq0} + K_s M_{sr} K_s^{-1} i_{r,t}^{dq0} \quad (\text{B.44})$$

$$\text{such that } \left( (K_s) \frac{d}{dt} (K_s^{-1}) \right) = W_s = \begin{bmatrix} 0 & -\omega_s \\ 0 & \omega_s & 0 \\ 0 & 0 & 0 \end{bmatrix},$$

$$\frac{d}{dt} \phi_{s,t}^{dq0} = u_{s,t}^{dq0} - r_s i_{s,t}^{dq0} - W_s i \phi_{s,t}^{dq0} \quad (\text{B.45})$$

$$\phi_{s,t}^{dq0} = l_s i_{s,t}^{dq0} + l_m i_{r,t}^{dq0} \quad (\text{B.46})$$

with  $K_s L_{ss} i K_s^{-1} = l_s$  and  $K_s M_{sr} K_s^{-1} = l_m$ .

The DFIG rotor variables can be expressed in  $d - q$  reference frame by using the same transformation as in () by changing  $(\theta_s - \theta_r)$

$$\frac{d}{dt} K_r^{-1} \phi_{r,t}^{dq0} = K_r^{-1} u_{r,t}^{dq0} - r_r K_r^{-1} i_{r,t}^{dq0} \quad (\text{B.47})$$

$$K_r^{-1} \phi_{r,t}^{dq0} = K_r^{-1} L_{rr} i_{r,t}^{dq0} + K_r^{-1} M_{rs} i_{s,t}^{dq0} \quad (\text{B.48})$$

then,

$$\frac{d}{dt} \phi_{r,t}^{dq0} = u_{r,t}^{dq0} - r_r i_{r,t}^{dq0} - \left( K_r \frac{d}{dt} K_r^{-1} \right) \phi_{r,t}^{dq0} \quad (\text{B.49})$$

$$\phi_{r,t}^{dq0} = K_r L_{rr} i K_r^{-1} i_{r,t}^{dq0} + K_r M_{rs} K_r^{-1} i_{s,t}^{dq0} \quad (\text{B.50})$$

$$\text{such that } \left( (K_r) \frac{d}{dt} (K_r^{-1}) \right) = W_r = \begin{bmatrix} 0 & 0 & \omega_r - \omega_s \\ 0 & \omega_s - \omega_r & 0 \\ 0 & 0 & 0 \end{bmatrix},$$

$$\frac{d}{dt} \phi_{r,t}^{dq0} = u_{r,t}^{dq0} - r_r i_{r,t}^{dq0} - W_r \phi_{r,t}^{dq0} \quad (\text{B.51})$$

$$\phi_{r,t}^{dq0} = l_r i_{r,t}^{dq0} + l_m i_{s,t}^{dq0} \quad (\text{B.52})$$

with  $K_r L_{rr} i K_r^{-1} = l_r$  and  $K_r M_{rs} K_r^{-1} = l_m$ .

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