

APPENDIX A: THE CONSTITUTIVE EQUATIONS

A.1 INTRODUCTION

In order to provide a reasonably self-contained basis for the development of 1-D constitutive equations of thin-walled beam theory, in general, and of the theory presented in this work, in particular, and to avoid any interruption in the course of our exposition, several elements on the constitutive equations of the 3-D anisotropic elasticity theory will be summarized. For similar reasons, a number of basic elements which concern the 3-D constitutive equations of piezoelectric materials, strictly related to the developments in Chapter 3 will also be supplied.

A.2 LINEARLY ELASTIC 3-D ANISOTROPIC CONTINUUM

The kinematic equations and the equations of motion are valid for every continuous medium, irrespective of its physical properties.

For an elastic body, in addition to the mentioned equations, the ones defining its physical properties have to be considered. If the strains are small enough, the physical behavior of an elastic anisotropic material can be approximated by a linear relationship between stress and strain components; such a material is termed *Hookean*, (see Wempner, 1981).

Composite materials consist of a reinforcement, generally in the form of fibers of high thermal and mechanical performance, dispersed in a surrounding matrix in an appropriate pattern. Such materials exhibit a blend of properties superior to those of their individual constituents. The matrix bonds the reinforcement together and distributes loads among the fibers; the reinforcement supports the mechanical loads. Due to their high specific stiffness and strength, resistance to corrosion as well as due to their tailorability property,

such structures are prime candidates in aeronautical, aerospace, naval, and civil applications; they can operate in severe and complex environmental conditions.

Changes in temperature are commonplace in composite materials, both during fabrication and use. There are two important effects emerging from the existence of the temperature field (see Daniel and Ishai, 1994; Vinson and Sierakowski, 2002). First, most materials expand when heated and contract when cooled, and secondly, the stiffness changes so that it becomes softer and weaker as it is heated. The combination of high temperature and high humidity has a detrimental effect on the structural performance of fiber-reinforced polymeric composites. To utilize in full the potential of composite materials, a method to predict their deformation and response characteristics due to changes in temperature and moisture absorption must be available (see Daniel and Ishai, 1994).

The ingestion of moisture leads to a linear variation of deformation with swelling, similar to the thermal loading. The constitutive equations incorporating changes in the temperature (T) and the specific moisture concentration (M) from some reference levels T_0 and M_0 , respectively, at which the body is considered to be stress free, can be expressed in Cartesian tensor form as:

$$\sigma_{ij} = C_{ijmn}\epsilon_{mn} - \lambda_{ij}T - \mu_{ij}M \quad (i, j, m, n = 1, 2, 3), \quad (\text{A.1})$$

where C_{ijmn} , λ_{ij} and μ_{ij} are the elasticity, the stress-temperature tensor and the stress-moisture tensor of the anisotropic body, respectively; these are fourth and second rank tensors, respectively.

This linear relationship characterizes the anisotropic Hookean solid and it represents an extension of the Duhamel-Neumann thermoelastic constitutive equations to the hygrothermoelastic case. We employ conventions of *free* (unrepeated) and *summation* (repeated) indices.

The components C_{ijmn} , λ_{ij} and μ_{ij} , satisfy the following symmetry conditions:

$$\begin{aligned} C_{ijmn} &= C_{jimn}, \quad C_{ijmn} = C_{ijnm}, \quad C_{ijmn} = C_{mnij}, \\ \lambda_{ij} &= \lambda_{ji}, \\ \mu_{ij} &= \mu_{ji}. \end{aligned} \quad (\text{A.2a-e})$$

The first symmetry relation follows directly from the symmetry of the stress tensor $\sigma_{ij} = \sigma_{ji}$, while the second arises from the symmetry of the strain tensor ϵ_{ij} . The third symmetry relation is a consequence of the interchangeability rule

$$\frac{\partial^2 \mathcal{W}}{\partial \epsilon_{ij} \partial \epsilon_{mn}} = \frac{\partial^2 \mathcal{W}}{\partial \epsilon_{mn} \partial \epsilon_{ij}}, \quad (\text{A.3})$$

where, for the present case

$$\mathcal{W} = \frac{1}{2} C_{ijmn} \epsilon_{ij} \epsilon_{mn} - \lambda_{ij} \epsilon_{ij} T - \mu_{ij} \epsilon_{ij} M. \quad (\text{A.4})$$

This form of (A.4) implies the existence of a free-reference-state characterized by $\sigma_{ij} = 0$ and $\epsilon_{ij} = 0$ at $T = T_0$ and $M = M_0$.

By virtue of these symmetries, the elastic and thermal coefficients can be represented in matrix form as:

$$[C_{ijmn}] \equiv \begin{bmatrix} C_{1111} & C_{1122} & C_{1133} & C_{1123} & C_{1131} & C_{1112} \\ & C_{2222} & C_{2233} & C_{2223} & C_{2231} & C_{2212} \\ & & C_{3333} & C_{3323} & C_{3331} & C_{3312} \\ & & & C_{2323} & C_{2331} & C_{2312} \\ & \text{Symm.} & & & C_{3131} & C_{3112} \\ & & & & & C_{1212} \end{bmatrix}, \quad (\text{A.5a})$$

$$[\lambda_{ij}] \equiv \begin{bmatrix} \lambda_{11} & \lambda_{12} & \lambda_{13} \\ & \lambda_{22} & \lambda_{23} \\ & \text{Symm.} & \lambda_{33} \end{bmatrix}, \quad (\text{A.5b})$$

$$[\mu_{ij}] \equiv \begin{bmatrix} \mu_{11} & \mu_{12} & \mu_{13} \\ & \mu_{22} & \mu_{23} \\ & \text{Symm.} & \mu_{33} \end{bmatrix}. \quad (\text{A.5c})$$

A body exhibiting such mechanical and thermal properties is termed *triclinic*.

For this general case, the matrices C_{ijmn} , λ_{ij} , and μ_{ij} are fully populated; there are 36 *nonzero* elastic coefficients, and 9 thermal and 9 hygrothermal coefficients. The symmetry properties in the general case of anisotropy, reduce the number of *independent* elastic coefficients to 21, and the thermal and hygrothermal ones to 6 each.

A.3 MATERIAL SYMMETRY

The various symmetries can reduce the number of independent coefficients C_{ijmn} , λ_{ij} , and μ_{ij} . We examine the effect on the strain energy density function of a rotation of axes.

A.3-1 One Surface of Symmetry (Monoclinic Hookean Material)

Suppose that through each point of the elastic body there is a surface $x_3 = 0$ with respect to which the energy functional remains invariant when x_3 is replaced by $-x_3$, then

$$\begin{aligned} C_{1123} &= C_{1131} = C_{2223} = C_{2231} = C_{2312} = C_{3112} = 0, \\ C_{3323} &= C_{3331}, \\ \lambda_{13} &= \lambda_{23} = 0, \\ \mu_{13} &= \mu_{23} = 0. \end{aligned} \quad (\text{A.6a-d})$$

Within this kind of symmetry, the elastic coefficients containing the index 3 either once or thrice, and the hygrothermal coefficients containing the index 3 once should vanish. The matrices in Eqs. (A.5) reduce to

$$[C_{ijmn}] \equiv \begin{bmatrix} C_{1111} & C_{1122} & C_{1133} & 0 & 0 & C_{1112} \\ & C_{2222} & C_{2233} & 0 & 0 & C_{2212} \\ & & C_{3333} & 0 & 0 & C_{3312} \\ & & & C_{2323} & C_{2331} & 0 \\ \text{Symm.} & & & & C_{3131} & 0 \\ & & & & & C_{1212} \end{bmatrix}, \quad (\text{A.7a})$$

$$[\lambda_{ij}] \equiv \begin{bmatrix} \lambda_{11} & \lambda_{12} & 0 \\ \text{Symm.} & \lambda_{22} & 0 \\ & & \lambda_{33} \end{bmatrix}, \quad (\text{A.7b})$$

$$[\mu_{ij}] \equiv \begin{bmatrix} \mu_{11} & \mu_{12} & 0 \\ \text{Symm.} & \mu_{22} & 0 \\ & & \mu_{33} \end{bmatrix}. \quad (\text{A.7c})$$

Such a material, referred to as *monoclinic* or *monotropic* (see Wempner, 1981), is characterized by 13 *independent* elastic coefficients, 4 thermal and 4 hygro coefficients. The axis x_3 normal to the surface of symmetry is referred to as the *principal material direction*.

A.3-2 Three Planes of Symmetry (Orthotropic Material)

If the material exhibits elastic and thermal symmetries at each point with respect to the orthogonal surfaces $x_3=0$ and $x_2=0$, then *additional* coefficients, besides the ones given by Eqs. (A.6), will vanish:

$$\begin{aligned} C_{1112} &= C_{3312} = C_{2331} = C_{2212} = 0, \\ \lambda_{12} &= 0, \quad \mu_{12} = 0. \end{aligned} \quad (\text{A.8a-c})$$

For this material

- (i) The elastic coefficients in Eqs. (A.6) and (A.8), containing the indices 1 and/or 3 once or thrice, and the thermal and the hygrothermal coefficients including the indices 1 and/or 3 once should vanish,
- (ii) There are number of 9 *independent* elastic coefficients, as well as 3 thermal and 3 hygrothermal coefficients that characterize the orthotropic solid,
- (iii) Being two orthogonal surfaces of material symmetry, the third mutually orthogonal surface will constitute another surface of symmetry, and,

- (iv) The coordinate axes normal to the surfaces of elastic symmetry are referred to as the *principal material directions*.

For an orthotropic body, the thermoelastic matrices are

$$[C_{ijmn}] \equiv \begin{bmatrix} C_{1111} & C_{1122} & C_{1133} & 0 & 0 & 0 \\ & C_{2222} & C_{2233} & 0 & 0 & 0 \\ & & C_{3333} & 0 & 0 & 0 \\ & & & C_{2323} & 0 & 0 \\ \text{Symm.} & & & & C_{3131} & 0 \\ & & & & & C_{1212} \end{bmatrix}, \quad (\text{A.9a})$$

$$[\lambda_{ij}] \equiv \begin{bmatrix} \lambda_{11} & 0 & 0 \\ & \lambda_{22} & 0 \\ \text{Symm.} & & \lambda_{33} \end{bmatrix}, \quad (\text{A.9b})$$

$$[\mu_{ij}] \equiv \begin{bmatrix} \mu_{11} & 0 & 0 \\ & \mu_{22} & 0 \\ \text{Symm.} & & \mu_{33} \end{bmatrix}. \quad (\text{A.9c})$$

A.3-3 Transverse Isotropy

A transversely isotropic material is one in which, at each point there is one principal material direction about which there is rotational symmetry. If, the principal direction is parallel at each point to the axis x_3 , and there is rotational symmetry about the x_3 axis, then the surface $x_1 - x_2$ is the surface of isotropy; the matrices of elastic, thermal and hygrothermal coefficients become

$$[C_{ijmn}] \equiv \begin{bmatrix} C_{1111} & C_{1122} & C_{1133} & 0 & 0 & 0 \\ & C_{1111} & C_{1133} & 0 & 0 & 0 \\ & & C_{3333} & 0 & 0 & 0 \\ & & & C_{3131} & 0 & 0 \\ \text{Symm.} & & & & C_{3131} & 0 \\ & & & & & (C_{1111} - C_{1122})/2 \end{bmatrix}, \quad (\text{A.10a})$$

$$[\lambda_{ij}] \equiv \begin{bmatrix} \lambda_{11} & 0 & 0 \\ & \lambda_{11} & 0 \\ \text{Symm.} & & \lambda_{33} \end{bmatrix}, \quad (\text{A.10b})$$

$$[\mu_{ij}] \equiv \begin{bmatrix} \mu_{11} & 0 & 0 \\ & \mu_{11} & 0 \\ \text{Symm.} & & \mu_{33} \end{bmatrix}. \quad (\text{A.10c})$$

There are 5 *independent* elastic coefficients, 2 thermal hygrothermal coefficients. One of the materials exhibiting the transverse isotropy properties is the *pyrolytic graphite*, which due to its special thermo-mechanical properties is likely to be an excellent candidate for being used in the thermal protection of aerospace vehicles. Moreover, *unidirectional fiber reinforced composites* belong to the class of transverse isotropy; in such a case the isotropy plane is normal to the axis of the fibers. If the fibers are aligned in the x_1 -direction, the x_2 - x_3 plane is the plane of isotropy, and the indices 2 and 3 are interchangeable. For this case

$$\begin{aligned} C_{2222} &= C_{3333}, & C_{1133} &= C_{1122}, \\ C_{3131} &= C_{1212}, & C_{2323} &= (C_{2222} - C_{2233})/2, \\ \lambda_{22} &= \lambda_{33}, & \mu_{22} &= \mu_{33}. \end{aligned} \quad (\text{A.11a-f})$$

The matrices of elastic and thermal coefficients become

$$[C_{ijmn}] \equiv \begin{bmatrix} C_{1111} & C_{1122} & C_{1122} & 0 & 0 & 0 \\ & C_{2222} & C_{2233} & 0 & 0 & 0 \\ & & C_{2222} & 0 & 0 & 0 \\ & & & (C_{2222} - C_{2233})/2 & 0 & 0 \\ \text{Symm.} & & & & C_{1212} & 0 \\ & & & & & C_{1212} \end{bmatrix}, \quad (\text{A.12a})$$

$$[\lambda_{ij}] \equiv \begin{bmatrix} \lambda_{11} & 0 & 0 \\ & \lambda_{22} & 0 \\ \text{Symm.} & & \lambda_{22} \end{bmatrix}, \quad (\text{A.12b})$$

$$[\mu_{ij}] \equiv \begin{bmatrix} \mu_{11} & 0 & 0 \\ & \mu_{22} & 0 \\ \text{Symm.} & & \mu_{22} \end{bmatrix}. \quad (\text{A.12c})$$

A.3-4 Isotropic Hookean Material

If there are no preferred directions, the role of the indices 1,2 and 3 are fully interchangeable. In this case

$$\begin{aligned} C_{1111} &= C_{2222} = C_{3333}, \\ C_{1122} &= C_{1133} = C_{2233}, \\ C_{1212} &= C_{3131} = C_{2323} \equiv (C_{1111} - C_{1122})/2, \\ \lambda_{11} &= \lambda_{22} = \lambda_{33}, \\ \mu_{11} &= \mu_{22} = \mu_{33}. \end{aligned} \quad (\text{A.13a-e})$$

As a result, the matrices of elastic and hygrothermal coefficients become

$$[C_{ijmn}] \equiv \begin{bmatrix} C_{1111} & C_{1122} & C_{1122} & 0 & 0 & 0 \\ & C_{1111} & C_{1122} & 0 & 0 & 0 \\ & & C_{1111} & 0 & 0 & 0 \\ & & & (C_{1111} - C_{1122})/2 & 0 & 0 \\ & & & & (C_{1111} - C_{1122})/2 & 0 \\ & & & & & (C_{1111} - C_{1122})/2 \end{bmatrix}, \tag{A.14a}$$

Symm.

$$[\lambda_{ij}] \equiv \begin{bmatrix} \lambda_{11} & 0 & 0 \\ & \lambda_{11} & 0 \\ & & \lambda_{11} \end{bmatrix}, \tag{A.14b}$$

Symm.

$$[\mu_{ij}] \equiv \begin{bmatrix} \mu_{11} & 0 & 0 \\ & \mu_{11} & 0 \\ & & \mu_{11} \end{bmatrix}. \tag{A.14c}$$

Symm.

The body has two independent elastic coefficients, and one thermal and one hygrothermal coefficient. An account of the elastic and hygrothermal coefficients for various types of elastic symmetries is presented in Table A.1.

Table A.1: Summary of 3-D elastic and hygrothermal coefficients

Type of material symmetry	Number of elastic independent coefficients	Number of thermal (hygro) independent coefficients	Number of nonzero elastic coefficients	Number of nonzero thermal (hygro) coefficients
Triclinic	21	6	36	9
Monoclinic	13	4	20	5
Orthotropic	9	3	12	3
Transversely isotropic	5	2	12	3
Isotropic	2	1	12	3

For orthotropic, transversely isotropic, and isotropic materials the shear and normal components of stress and strain tensors are not coupled.

Triclinic and isotropic materials represent two extreme types of anisotropy: the former has one surface of symmetry, the latter an infinity. For a thorough discussion of the various types of anisotropy, see Bogdanovich and Pastore (1996).

A.4 ALTERNATIVE FORM OF THE 3-D CONSTITUTIVE EQUATIONS

Recall the general constitutive equations of elastic anisotropic Hookean solids:

$$\sigma_{ij} = C_{ijmn}\epsilon_{mn} - \lambda_{ij}T - \mu_{ij}M, \quad (\text{A.15a})$$

$$\epsilon_{ij} = S_{ijmn}\sigma_{mn} + \alpha_{ij}T + \beta_{ij}M. \quad (\text{A.15b})$$

Equation (A.15b) represents the inverted counterpart of Eq. (A.15a). Here S_{ijmn} denotes the compliance tensor, while α_{ij} and β_{ij} denote the thermal expansion and the moisture-swelling tensors, respectively. Their components follow the same pattern of *nonzero* and number of independent coefficients in the various symmetry cases as their inverted counterparts. In order to establish the relationship between (S_{ijmn}, C_{ijmn}) , (λ_{ij}, μ_{ij}) , and $(\alpha_{ij}, \beta_{ij})$ we will replace σ_{mn} in Eq. (A.15b) by its expression in Eq. (A.15a). This yields (Librescu, 1990)

$$\epsilon_{ij} = S_{ijmn} [C_{mnpq}\epsilon_{pq} - \lambda_{mn}T - \mu_{mn}M] + \alpha_{ij}T + \beta_{ij}M. \quad (\text{A.16})$$

Collection and identification of the coefficients associated with ϵ_{ij} , T and M results in :

$$\begin{aligned} S_{ijmn}C_{mnpq} &= (\delta_{ip}\delta_{jq} + \delta_{iq}\delta_{jp})/2, \\ \alpha_{ij} - S_{ijmn}\lambda_{mn} &= 0, \\ \beta_{ij} - S_{ijmn}\mu_{mn} &= 0, \end{aligned} \quad (\text{A.17a-c})$$

where δ_{ij} denotes the Kronecker delta ($\delta_{ij} = 0$ if $i \neq j$, and $\delta_{ij} = 1$ if $i = j$). Equation (A.17a) expresses the fact that C_{ijmn} and S_{mnpq} are inverse tensors, whereas the remaining two equations establish a relationship between α_{ij} and λ_{ij} , and between β_{ij} and μ_{ij} , respectively. Multiplication of equations (A.17b) and (A.17c) by C_{mnpq} and employment of Eq. (A.17a) yields

$$\lambda_{pq} = \alpha_{ij}C_{ijpq}, \quad \mu_{pq} = \beta_{ij}C_{ijpq}. \quad (\text{A.18a,b})$$

In light of Eqs. (A.18) and (A.17b, c), the constitutive equations (A.15) become

$$\begin{aligned} \sigma_{ij} &= C_{ijmn} [\epsilon_{mn} - \alpha_{mn}T - \beta_{mn}M], \\ \epsilon_{ij} &= S_{ijmn} [\sigma_{mn} + \lambda_{mn}T + \mu_{mn}M]. \end{aligned} \quad (\text{A.19a,b})$$

This form of the constitutive equations will be used later.

A.5 TRANSFORMATION OF MATERIAL COEFFICIENTS

In Section A.3.2 the characteristics of an orthotropic material were described assuming that its axes of material symmetry coincide with the geometrical axes (i.e. with the global axes of convenience).

However, the principal directions of orthotropy of the constituent layer materials often do not coincide with the geometrical axes whose orientation is dictated by the character of the problem to be investigated or by the boundary conditions.

In addition, for a laminated composite structure the principal directions of orthotropy are different in each constituent layer, this being part of the implementation of the tailoring technique.

In such cases, the transformation of the constitutive relations with respect to the global coordinate system is required.

If the constitutive behavior in the principal material coordinate system ($x'_1, x'_2, x'_3 (\equiv x_3)$) is known, the constitutive law in the axes of convenience, (x_1, x_2, x_3), as represented in Fig. A.1, is found by applying the tensor transformation rules.

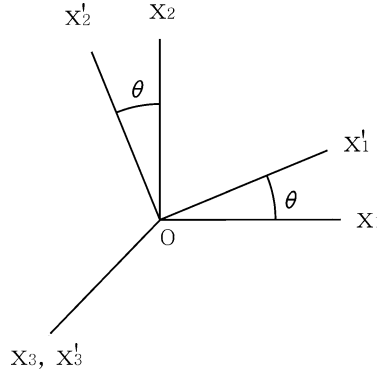


Figure A.1: Original and rotated material coordinate systems

The constitutive equations referred to the primed coordinate system, after an in-plane rotation of the $x_1 - x_2$ axes about the x_3 -axis, assume the form:

$$\sigma_{i'j'} = C_{i'j'k'l'}(\epsilon_{k'l'} - \alpha_{k'l'}T - \beta_{k'l'}M), \quad (\text{A.20})$$

where

$$\begin{aligned} \sigma_{i'j'} &= a_{i'l'}a_{j'm}\sigma_{lm}, \\ \epsilon_{k'l'} &= a_{k'p}a_{l'q}\epsilon_{pq}, \\ \alpha_{k'l'} &= a_{k'p}a_{l'q}\alpha_{pq}, \\ \beta_{k'l'} &= a_{k'p}a_{l'q}\beta_{pq}. \end{aligned} \quad (\text{A.21a-d})$$

In Eqs. (A.21), $a_{i'j}$ denotes the direction cosine of the angle between the $x_{i'}$ and x_j axes, so that

$$[a] \equiv a_{i'j} = \begin{bmatrix} m & n & 0 \\ -n & m & 0 \\ 0 & 0 & 1 \end{bmatrix} \quad (\text{A.22})$$

where $m \equiv \cos \theta, n \equiv \sin \theta, \theta \in [0, 2\pi]$ being the counterclockwise rotation angle about the positive x_3 -axis. Note that $[a]$ is an orthogonal matrix which satisfies the relationship $[a][a]^T = \mathbf{I} = [a]^T[a], ([a]^T \equiv [a]^{-1})$ where superscript T denotes the matrix transpose, while \mathbf{I} is the identity matrix whose diagonal elements are the unites.

In matrix form Eq. (A.21a) is given by

$$\{\sigma'\} = [T_3] \{\sigma\}. \tag{A.23}$$

Here

$$T_3(\theta) = \begin{bmatrix} m^2 & n^2 & 0 & 0 & 0 & 2mn \\ n^2 & m^2 & 0 & 0 & 0 & -2mn \\ 0 & 0 & 1 & 0 & 0 & 0 \\ 0 & 0 & 0 & m & -n & 0 \\ 0 & 0 & 0 & n & m & 0 \\ -mn & mn & 0 & 0 & 0 & m^2 - n^2 \end{bmatrix} \tag{A.24}$$

is the transformation matrix, while $\{\sigma'\}$ and $\{\sigma\}$ denote the stress tensors in the primed and unprimed coordinates, respectively. Similarly, the transformation of strains is given by

$$\{\gamma'\} = [\tilde{T}_3] \{\gamma\}, \tag{A.25}$$

where

$$\tilde{T}_3(\theta) = \begin{bmatrix} m^2 & n^2 & 0 & 0 & 0 & mn \\ n^2 & m^2 & 0 & 0 & 0 & -mn \\ 0 & 0 & 1 & 0 & 0 & 0 \\ 0 & 0 & 0 & m & -n & 0 \\ 0 & 0 & 0 & n & m & 0 \\ -2mn & 2mn & 0 & 0 & 0 & m^2 - n^2 \end{bmatrix}, \tag{A.26}$$

and $\gamma_{ij} \equiv \epsilon_{ij}$ for $i = j$ and $\gamma_{ij} \equiv 2\epsilon_{ij}$ for $i \neq j$. Here ϵ_{ij} denote the tensorial strain components. It is apparent that the following relationship between \tilde{T}_3 and T_3 holds valid

$$\tilde{T}_3(\theta) = [T_3^{-1}]^T = [T_3(-\theta)]^T. \tag{A.27}$$

Replacement in Eq. (A.20) of Eqs. (A.21) and (A.25) results, in view of Eqs. (A.24), (A.26) and (A.27), in the constitutive equations expressed in the global coordinates (x_1, x_2, x_3) , where the elastic and hygrothermal properties in the principal material coordinates (x'_1, x'_2, x'_3) are assumed to be known. In matrix form these are given as:

$$\{\sigma\} = [C] (\{\gamma\} - \{\alpha\} T - \{\beta\} M), \tag{A.28}$$

where

$$\begin{aligned} \{\sigma\} &= \{\sigma_{11}, \sigma_{22}, \sigma_{33}, \sigma_{23}, \sigma_{31}, \sigma_{12}\}^T, \\ \{\gamma\} &= \{\epsilon_{11}, \epsilon_{22}, \epsilon_{33}, \gamma_{23}, \gamma_{31}, \gamma_{12}\}^T, \\ \{\alpha\} &\equiv [T_3]^{-1} \{\alpha'\} = \{\alpha_{11}, \alpha_{22}, \alpha_{33}, 0, 0, \alpha_{12}\}^T, \\ \{\beta\} &\equiv [T_3]^{-1} \{\beta'\} = \{\beta_{11}, \beta_{22}, \beta_{33}, 0, 0, \beta_{12}\}^T, \end{aligned} \quad (\text{A.29a-d})$$

while

$$[C] = [T_3]^{-1} [C'] [\tilde{T}_3], \quad (\text{A.30a})$$

where

$$[C'] \equiv [C_{i'j'k'l'}] \text{ and } [T_3(\theta)]^{-1} = [T_3(-\theta)]. \quad (\text{A.30b,c})$$

Expressed in full, Eq. (A.28) becomes

$$\begin{Bmatrix} \sigma_{11} \\ \sigma_{22} \\ \sigma_{33} \\ \sigma_{23} \\ \sigma_{31} \\ \sigma_{12} \end{Bmatrix} = \begin{bmatrix} C_{1111} & C_{1122} & C_{1133} & 0 & 0 & C_{1112} \\ C_{2211} & C_{2222} & C_{2233} & 0 & 0 & C_{2212} \\ C_{3311} & C_{3322} & C_{3333} & 0 & 0 & C_{3312} \\ 0 & 0 & 0 & C_{2323} & C_{2331} & 0 \\ 0 & 0 & 0 & C_{3123} & C_{3131} & 0 \\ C_{1211} & C_{1222} & C_{1233} & 0 & 0 & C_{1212} \end{bmatrix} \begin{Bmatrix} \epsilon_{11} - \alpha_{11}T - \beta_{11}M \\ \epsilon_{22} - \alpha_{22}T - \beta_{22}M \\ \epsilon_{33} - \alpha_{33}T - \beta_{33}M \\ \gamma_{23} \\ \gamma_{31} \\ \gamma_{12} - \alpha_{12}T - \beta_{12}M \end{Bmatrix}. \quad (\text{A.31})$$

where the symmetry of the stiffness matrix $[C]$ is implied. In Eqs. (A.30a) and (A.29c,d), $[C]$, $\{\alpha\}$ and $\{\beta\}$ denote, respectively, the matrices of transformed elastic and hygrothermal coefficients in the global coordinates.

We will use Voigt's contracted notation in place of the tensor indicial notation. This consists of replacing in the elastic and hygrothermal coefficients of a pair of indices by a single one, through the replacement rules: 11 \rightarrow 1; 22 \rightarrow 2; 33 \rightarrow 3; 23 (or 32) \rightarrow 4; 31 (or 13) \rightarrow 5; 12 (or 21) \rightarrow 6.

On the basis of Eqs. (A.30b) and (A.25), Eq. (A.30a) can be written in full as:

$$\begin{aligned} C_{11} &= C_{1'1'}m^4 + 2(C_{1'2'} + 2C_{6'6'})m^2n^2 + C_{2'2'}n^4, \\ C_{12} &= (C_{1'1'} + C_{2'2'} - 4C_{6'6'})m^2n^2 + C_{1'2'}(m^4 + n^4), \\ C_{13} &= C_{1'3'}m^2 + C_{2'3'}n^2, \\ C_{16} &= -C_{2'2'}mn^3 + C_{1'1'}m^3n - (C_{1'2'} + 2C_{6'6'})mn(m^2 - n^2), \\ C_{22} &= C_{2'2'}m^4 + 2(C_{1'2'} + 2C_{6'6'})m^2n^2 + C_{1'1'}n^4, \\ C_{23} &= C_{1'3'}n^2 + C_{2'3'}m^2, \\ C_{26} &= C_{1'1'}mn^3 - C_{2'2'}m^3n + (C_{1'2'} + 2C_{6'6'})mn(m^2 - n^2), \\ C_{33} &= C_{3'3'}, \end{aligned} \quad (\text{A.32a-m})$$

$$\begin{aligned}
C_{36} &= (C_{1'3'} - C_{2'3'})mn, \\
C_{44} &= C_{4'4'}m^2 + C_{5'5'}n^2, \\
C_{45} &= (C_{5'5'} - C_{4'4'})mn, \\
C_{55} &= C_{4'4'}n^2 + C_{5'5'}m^2, \\
C_{66} &= (C_{1'1'} + C_{2'2'} - 2C_{1'2'})m^2n^2 + C_{6'6'}(m^2 - n^2)^2.
\end{aligned}$$

In addition, the elements of matrices α and β are, respectively:

$$\begin{aligned}
\alpha_1 &= \alpha_{1'}m^2 + \alpha_{2'}n^2, \\
\alpha_2 &= \alpha_{2'}m^2 + \alpha_{1'}n^2, \\
\alpha_3 &= \alpha_{3'}, \\
\alpha_6 &= 2(\alpha_{1'} - \alpha_{2'})mn,
\end{aligned} \tag{A.33a-d}$$

and

$$\begin{aligned}
\beta_1 &= \beta_{1'}m^2 + \beta_{2'}n^2, \\
\beta_2 &= \beta_{2'}m^2 + \beta_{1'}n^2, \\
\beta_3 &= \beta_{3'}, \\
\beta_6 &= 2(\beta_{1'} - \beta_{2'})mn.
\end{aligned} \tag{A.34a-d}$$

In Eqs. (A.32) through (A.34), $C_{i'j'}$, $\alpha_{i'}$ and $\beta_{i'}$ denote the elastic and hygrothermal coefficients of an orthotropic material expressed in principal material directions. The relationships in Eqs. (A.31) through (A.34) reveal that in non-principal coordinates (or off-axis coordinates), an orthotropic body is characterized, likewise a monoclinic material, by 13 independent elastic constants and 4 independent thermal/hygrothermal coefficients. In this case, the coupling between normal and shear effects is present. However, C_{16} , C_{26} , C_{36} , α_6 , and β_6 are not independent, but merely linear combinations of the remaining constants. Since the material, in its principal material directions is orthotropic, it is called a *generally orthotropic* material.

Equations (A.32) through (A.34) show that when the principal and the global (geometric) coordinates coincide (implying $\theta = 0$), C_{ij} , α_i , and β_i reduce to $C_{i'j'}$, $\alpha_{i'}$, and $\beta_{i'}$, respectively. In such a case the material behaves as a truly orthotropic one, and the equations (A.31) become:

$$\left\{ \begin{array}{c} \sigma_1 \\ \sigma_2 \\ \sigma_3 \\ \sigma_4 \\ \sigma_5 \\ \sigma_6 \end{array} \right\} = \left[\begin{array}{cccccc} C_{11} & C_{12} & C_{13} & 0 & 0 & 0 \\ C_{12} & C_{22} & C_{23} & 0 & 0 & 0 \\ C_{13} & C_{23} & C_{33} & 0 & 0 & 0 \\ 0 & 0 & 0 & C_{44} & 0 & 0 \\ 0 & 0 & 0 & 0 & C_{55} & 0 \\ 0 & 0 & 0 & 0 & 0 & C_{66} \end{array} \right] \left\{ \begin{array}{c} \epsilon_1 - \alpha_1 T - \beta_1 M \\ \epsilon_2 - \alpha_2 T - \beta_2 M \\ \epsilon_3 - \alpha_3 T - \beta_3 M \\ \gamma_4 \\ \gamma_5 \\ \gamma_6 \end{array} \right\}. \tag{A.35}$$

It should be noticed that when the coordinate transformations have to be implemented, the tensor indices must be employed.

A.6 ALTERNATIVE REPRESENTATIONS

Equations (A.32) through (A.34) show that the stiffness quantities of a generally orthotropic material depend on the angle of orientation of its principal material directions. The technology taking advantage of this property, in the sense of modifying the response of the structure according to predetermined goals by changing fiber orientation angle, is referred to as *structural tailoring*. It has been used successfully in the aeronautical industry and is likely to play a significant role in the design of future generations of aeronautical/aerospace vehicles, of helicopter and turbine blades, of space stations, satellites, etc. For practical purposes an alternative representation of equations (A.32) through (A.34) will be presented. In this representation a separation of the quantities that are invariant to the rotation of axis (and which represent the truly composite material properties) and of those depending on the rotation of the principal material directions is applied.

To this end, and following Tsai and Hahn (1980) we use the elementary trigonometric identities:

$$\begin{aligned}
 m^4 &= (3 + 4 \cos 2\theta + \cos 4\theta)/8, \\
 m^3 n &= (2 \sin 2\theta + \sin 4\theta)/8, \\
 mn^3 &= (2 \sin 2\theta - \sin 4\theta)/8, \\
 m^2 n^2 &= (1 - \cos 4\theta)/8, \\
 n^4 &= (3 - 4 \cos 2\theta + \cos 4\theta)/8.
 \end{aligned}
 \tag{A.36a-e}$$

Direct substitution of Eqs. (A.36) into Eqs. (A.32) through (A.34) yields:

$$\begin{aligned}
 C_{11} &= U'_1 + U'_2 \cos 2\theta + U'_3 \cos 4\theta, \\
 C_{22} &= U'_1 - U'_2 \cos 2\theta + U'_3 \cos 4\theta, \\
 C_{12} &= U'_4 - U'_3 \cos 4\theta, \\
 C_{66} &= U'_5 - U'_3 \cos 4\theta, \\
 C_{16} &= (U'_2 \sin 2\theta)/2 + U'_3 \sin 4\theta, \\
 C_{26} &= (U'_2 \sin 2\theta)/2 - U'_3 \sin 4\theta,
 \end{aligned}
 \tag{A.37a-l}$$

$$\begin{aligned}
 C_{13} &= U'_8 + U'_9 \cos 2\theta, \\
 C_{23} &= U'_8 - U'_9 \cos 2\theta, \\
 C_{55} &= U'_{10} + U'_{11} \cos 2\theta,
 \end{aligned}$$

$$\begin{aligned}C_{45} &= U'_{11} \sin 2\theta, \\C_{44} &= U'_{10} - U'_{11} \cos 2\theta, \\C_{36} &= U'_9 \sin 2\theta,\end{aligned}$$

and

$$\begin{aligned}\alpha_1 &= V'_1 + V'_2 \cos 2\theta, \\ \alpha_2 &= V'_1 - V'_2 \cos 2\theta, \\ \alpha_6 &= V'_2 \sin 2\theta, \\ \alpha_3 &= \alpha_{3'},\end{aligned}\tag{A.38a-d}$$

and

$$\begin{aligned}\beta_1 &= P'_1 + P'_2 \cos 2\theta, \\ \beta_2 &= P'_1 - P'_2 \cos 2\theta, \\ \beta_6 &= P'_2 \sin 2\theta, \\ \beta_3 &= \beta_{3'}.\end{aligned}\tag{A.39a-d}$$

The quantities U'_i , V'_i , and P'_i depend only on the properties of the orthotropic material and are independent of ply-angle θ).

Their expressions are:

$$\begin{aligned}U'_1 &= (3C_{1'1'} + 3C_{2'2'} + 2C_{1'2'} + 4C_{6'6'})/8, \\ U'_2 &= (C_{1'1'} - C_{2'2'})/2, \\ U'_3 &= (C_{1'1'} + C_{2'2'} - 2C_{1'2'} - 4C_{6'6'})/8, \\ U'_4 &= (C_{1'1'} + C_{2'2'} + 6C_{1'2'} - 4C_{6'6'})/8, \\ U'_5 &= (C_{1'1'} + C_{2'2'} - 2C_{1'2'} + 4C_{6'6'})/8, \\ U'_8 &= (C_{1'3'} + C_{2'3'})/2, \\ U'_9 &= (C_{1'3'} - C_{2'3'})/2, \\ U'_{10} &= (C_{5'5'} + C_{4'4'})/2, \\ U'_{11} &= (C_{5'5'} - C_{4'4'})/2, \\ V'_1 &= (\alpha_{1'} + \alpha_{2'})/2, \\ V'_2 &= (\alpha_{1'} - \alpha_{2'})/2, \\ P'_1 &= (\beta_{1'} + \beta_{2'})/2, \\ P'_2 &= (\beta_{1'} - \beta_{2'})/2.\end{aligned}\tag{A.40a-m}$$

In Eqs. (A.37) through (A.39), the positive sign of θ is opposite to that considered in the tensorial transformation, i.e. it is positive when measured from the on-axis (i.e. the symmetry axes) of a ply to the laminate coordinate system. This sign reversal of the angle θ , enables one to emphasize the contribution

of an off-axis ply-angle to the laminate. This representation is useful in the tailoring analysis of laminates. Each constituent lamina could have a different ply orientation.

The coordinate transformation (A.32) through (A.34) yield the following invariants

$$\begin{aligned}
 C_{11} + C_{22} + C_{33} + 2(C_{12} + C_{23} + C_{31}) \\
 &= C_{1'1'} + C_{2'2'} + C_{3'3'} + 2(C_{1'2'} + C_{2'3'} + C_{3'1'}), \\
 C_{11} + C_{22} + C_{33} + 2(C_{44} + C_{55} + C_{66}) \\
 &= C_{1'1'} + C_{2'2'} + C_{3'3'} + 2(C_{4'4'} + C_{5'5'} + C_{6'6'}), \\
 \alpha_1 + \alpha_2 + \alpha_3 &= \alpha_{1'} + \alpha_{2'} + \alpha_{3'}, \\
 \beta_1 + \beta_2 + \beta_3 &= \beta_{1'} + \beta_{2'} + \beta_{3'}.
 \end{aligned} \tag{A.41a-d}$$

A.7 ELASTIC COEFFICIENTS OF ORTHOTROPIC MATERIALS IN TERMS OF ENGINEERING CONSTANTS

In practice, the coefficients C_{ij} for the orthotropic material are defined in terms of the engineering constants. To this end, as a first step, from Eq. (A.17a) the relationships between C_{ij} and S_{ij} for an orthotropic body are derived. These are:

$$\begin{aligned}
 C_{11} &= (S_{22}S_{33} - S_{23}^2)/S, \\
 C_{12} &= (S_{13}S_{23} - S_{12}S_{33})/S, \\
 C_{22} &= (S_{33}S_{11} - S_{13}^2)/S, \\
 C_{13} &= (S_{12}S_{23} - S_{13}S_{22})/S, \\
 C_{33} &= (S_{11}S_{22} - S_{12}^2)/S, \\
 C_{23} &= (S_{12}S_{13} - S_{23}S_{11})/S, \\
 C_{44} &= 1/S_{44}, \\
 C_{55} &= 1/S_{55}, \\
 C_{66} &= 1/S_{66},
 \end{aligned} \tag{A.42a-i}$$

where

$$S = S_{11}S_{22}S_{33} - S_{11}S_{23}^2 - S_{22}S_{13}^2 - S_{33}S_{12}^2 + 2S_{12}S_{23}S_{13}. \tag{A.43}$$

The components of the compliance tensor are expressed in terms of the engineering constants as

$$[S_{ij}] = \begin{bmatrix} \frac{1}{E_1} & -\frac{\nu_{21}}{E_2} & -\frac{\nu_{31}}{E_3} & 0 & 0 & 0 \\ -\frac{\nu_{12}}{E_1} & \frac{1}{E_2} & -\frac{\nu_{32}}{E_3} & 0 & 0 & 0 \\ -\frac{\nu_{13}}{E_1} & -\frac{\nu_{23}}{E_2} & \frac{1}{E_3} & 0 & 0 & 0 \\ 0 & 0 & 0 & \frac{1}{G_{23}} & 0 & 0 \\ 0 & 0 & 0 & 0 & \frac{1}{G_{31}} & 0 \\ 0 & 0 & 0 & 0 & 0 & \frac{1}{G_{12}} \end{bmatrix}. \quad (\text{A.44})$$

Substitution of the expressions of S_{ij} as given by (A.44) into the right-hand side members of Eqs. (A.42) and (A.43) gives the relationships between C_{ij} and the engineering coefficients as:

$$\begin{aligned} C_{11} &= E_1(1 - \nu_{23}\nu_{32})/\Delta, \\ C_{22} &= E_2(1 - \nu_{31}\nu_{13})/\Delta, \\ C_{33} &= E_3(1 - \nu_{12}\nu_{21})/\Delta, \\ C_{12} &= E_1(\nu_{21} + \nu_{31}\nu_{23})/\Delta, \\ C_{13} &= E_1(\nu_{31} + \nu_{21}\nu_{32})/\Delta, \\ C_{23} &= E_2(\nu_{32} + \nu_{12}\nu_{31})/\Delta, \\ C_{44} &= G_{23}, \\ C_{55} &= G_{31}, \\ C_{66} &= G_{12}. \end{aligned} \quad (\text{A.45a-i})$$

Here E_1, E_2, E_3 denote the Young's moduli along the principal material directions; G_{12}, G_{23}, G_{31} denote the shear moduli, $\nu_{12}, \nu_{23}, \nu_{31}$ denote the Poisson's ratios while

$$\Delta = 1 - \nu_{12}\nu_{21} - \nu_{32}\nu_{23} - \nu_{13}\nu_{31} - 2\nu_{21}\nu_{13}\nu_{32}. \quad (\text{A.46})$$

From symmetry considerations of the compliance matrix $[S_{ij}]$, the following reciprocal relationship hold

$$\frac{E_i}{\nu_{ij}} = \frac{E_j}{\nu_{ji}} \quad (i, j = 1, 2, 3) \quad (\text{A.47})$$

For a *unidirectional composite reinforced in the x_1 -direction* we have

$$\begin{aligned} E_2 = E_3 &\equiv E, & E_1 &\equiv E', \\ \nu_{13} = \nu_{12} &\equiv \nu', & \nu_{23} = \nu_{32} &\equiv \nu, \\ G_{23} &\equiv G = E/2(1 + \nu), & G_{12} = G_{13} &\equiv G', \\ \nu_{21} = \nu_{31} &= \nu'E/E'. \end{aligned} \quad (\text{A.48a-g})$$

For a transversely isotropic solid whose plane of isotropy is parallel at each point to the plane x_1 - x_2 , the elastic constants are

$$\begin{aligned} E_1 = E_2 &\equiv E; E_3 = E'; & G_{12} &\equiv G (\equiv E/2(1 + \nu)); & G_{23} = G_{31} &\equiv G', \\ \nu_{31} = \nu_{32} &\equiv \nu'; & \nu_{12} = \nu_{21} &\equiv \nu; & \nu_{13} = \nu_{23} &\equiv \nu'E/E'. \end{aligned} \quad (\text{A.49a-g})$$

For these two cases, the associated stiffness coefficients C_{ij} are obtainable by replacing Eqs. (A.48) and (A.49) in (A.45).

For completeness, the elastic stiffness coefficients C_{ij} for this type of transverse isotropy are

$$\begin{aligned} C_{11} &= \frac{E(E\nu'^2 - E')}{\Delta} = C_{22}, \\ C_{12} &= -\frac{E(E\nu'^2 + E'\nu)}{\Delta}, \\ C_{13} &= -\frac{EE'\nu'(1 + \nu)}{\Delta} = C_{23}, \\ C_{33} &= -\frac{E'^2(1 - \nu^2)}{\Delta}, \\ C_{44} &= C_{55} = G', \end{aligned} \quad (\text{A.50a-e})$$

where $\Delta = (1 + \nu)(2E\nu'^2 + E'\nu - E')$.

The pyrolytic graphite and its alloys feature this type of transverse isotropy. Due to its very low thermal conductivity coefficient in the thickness direction as compared to that in the tangential direction, it is used in the design of thermal protection of aerospace vehicle systems.

A.8 ANISOTROPIC THIN-WALLED BEAMS

A.8-1 Introduction

A thin-walled composite beam consists of a number of laminae bonded and/or cured together. The theory of thin-walled beams is essentially a 1-D approximation of the 3-D theory of elasticity. This should result in a system of ordinary differential governing equations expressed in terms of the 1-D displacements measures $u_0(x_2)$, $v_0(x_2)$, $w_0(x_2)$, the 1-D rotational measures $\theta_1(x_2)$, $\theta_2(x_2)$, $\phi(x_2)$ and the warping measure W_M . To obtain the system of 1-D governing

equations, the constitutive equations have to feature the same 1-D character. An intermediate stage, consisting of the derivation of 2-D constitutive equations will be considered; the 2-D stress resultants and stress couples should relate the 2-D strain measures, where $x_1 (\equiv s)$ and $x_2 (\equiv z)$ are the independent coordinates involved in these equations.

A.8-2 3-D Equations for a Lamina

Consider a laminated composite thin-walled beam consisting of N orthotropic laminae referenced to a nonprincipal material coordinate system. For the k -th constituent lamina of the laminate, the constitutive equations are

$$\begin{Bmatrix} \sigma_{ss} \\ \sigma_{zz} \\ \sigma_{nn} \\ \sigma_{zn} \\ \sigma_{sn} \\ \sigma_{sz} \end{Bmatrix}_{(k)} = [C]_{(k)} \begin{Bmatrix} \epsilon_{ss} - \alpha_s T - \beta_s M \\ \epsilon_{zz} - \alpha_z T - \beta_z M \\ \epsilon_{nn} - \alpha_n T - \beta_n M \\ \gamma_{zn} \\ \gamma_{sn} \\ \gamma_{sz} - \alpha_{sz} T - \beta_{sz} M \end{Bmatrix}_{(k)}, \quad (\text{A.51a})$$

where

$$[C]_{(k)} \equiv \begin{bmatrix} C_{11} & C_{12} & C_{13} & 0 & 0 & C_{16} \\ C_{12} & C_{22} & C_{23} & 0 & 0 & C_{26} \\ C_{13} & C_{23} & C_{33} & 0 & 0 & C_{36} \\ 0 & 0 & 0 & C_{44} & C_{45} & 0 \\ 0 & 0 & 0 & C_{45} & C_{55} & 0 \\ C_{16} & C_{26} & C_{36} & 0 & 0 & C_{66} \end{bmatrix}_{(k)}. \quad (\text{A.51b})$$

Since the material properties in Eqs. (A.51) are affiliated to the k -th layer, for the sake of identification, the matrices are associated with the subscript (k) . Unless otherwise specified, the indices s , z , and n will replace the former indices 1, 2, and 3, respectively.

As in the theory of plates and shells (see e.g. Librescu, 1975, 1990), we will invoke the fact that σ_{nn} is negligible when compared with the other stress components. The condition $\sigma_{nn} = 0$ leads to the following transverse normal strain

$$\begin{aligned} \epsilon_{nn} = & -\frac{C_{13}}{C_{33}}(\epsilon_{ss} - \alpha_s T - \beta_s M) - \frac{C_{23}}{C_{33}}(\epsilon_{zz} - \alpha_z T - \beta_z M) \\ & - \frac{C_{36}}{C_{33}}(\gamma_{sz} - \alpha_{sz} T - \beta_{sz} M) + \alpha_n T + \beta_n M. \end{aligned} \quad (\text{A.52})$$

As a result, Eq. (A.51a) is recast in the form:

$$\begin{Bmatrix} \sigma_{ss} \\ \sigma_{zz} \\ \sigma_{zn} \\ \sigma_{sn} \\ \sigma_{sz} \end{Bmatrix}_{(k)} = [\bar{Q}]_{(k)} \begin{Bmatrix} \varepsilon_{ss} - \bar{\alpha}_s T - \bar{\beta}_s M \\ \varepsilon_{zz} - \bar{\alpha}_z T - \bar{\beta}_z M \\ \gamma_{zn} \\ \gamma_{sn} \\ \gamma_{sz} - \bar{\alpha}_{sz} T - \bar{\beta}_{sz} M \end{Bmatrix}_{(k)}, \quad (\text{A.53})$$

where $[\bar{Q}]$ is the matrix of the *reduced elastic coefficients* given by

$$[\bar{Q}] = \begin{bmatrix} \bar{Q}_{11} & \bar{Q}_{12} & 0 & 0 & \bar{Q}_{16} \\ \bar{Q}_{21} & \bar{Q}_{22} & 0 & 0 & \bar{Q}_{26} \\ 0 & 0 & \bar{Q}_{44} & \bar{Q}_{45} & 0 \\ 0 & 0 & \bar{Q}_{54} & \bar{Q}_{55} & 0 \\ \bar{Q}_{61} & \bar{Q}_{62} & 0 & 0 & \bar{Q}_{66} \end{bmatrix}, \quad (\text{A.54a})$$

where

$$\bar{Q}_{IJ} = C_{IJ} - \frac{C_{I3}C_{J3}}{C_{33}} = \bar{Q}_{JI}, \quad (I, J = 1, 2, 6), \quad (\text{A.54b})$$

$$\bar{Q}_{LM} \equiv C_{LM}, \quad (L, M = 4, 5) \quad (\text{A.54c})$$

while

$$\bar{\alpha}_I = \alpha_I - \frac{C_{I3}}{C_{33}}\alpha_3, \quad \bar{\beta}_I = \beta_I - \frac{C_{I3}}{C_{33}}\beta_3, \quad (\text{A.54d})$$

are the *reduced* stress-temperature and stress-moisture coefficients, respectively.

For an orthotropic body:

$$\begin{aligned} \bar{Q}_{11} &= \frac{E_1}{1 - \nu_{12}\nu_{21}}; & \bar{Q}_{12} &= \frac{E_1\nu_{21}}{1 - \nu_{12}\nu_{21}} = \frac{E_2\nu_{12}}{1 - \nu_{12}\nu_{21}} \\ \bar{Q}_{22} &= \frac{E_2}{1 - \nu_{12}\nu_{21}}; & \bar{Q}_{66} &= \bar{C}_{66} = G_{12}; & \bar{Q}_{44} &= G_{23}; & \bar{Q}_{55} &= G_{31}. \end{aligned} \quad (\text{A.55a-f})$$

A.8-3 2-D Stress-Resultants and Stress-Couples

In order to obtain a 2-D variant of the constitutive equations, the 2-D stress resultants and stress couples have to be defined. As in the theory of plates and shells, these quantities are defined so that these 2-D quantities are statically equivalent to their 3-D counterparts. Suppose that the k th lamina occupies the domain $n_{(k-1)} < n < n_{(k)}$, $k = 1 \dots N$, we define:

(a) *The membrane stress resultants*

$$\begin{Bmatrix} N_{ss} \\ N_{zz} \\ N_{sz} \end{Bmatrix} = \sum_{k=1}^N \int_{n_{(k-1)}}^{n_{(k)}} \begin{Bmatrix} \sigma_{ss} \\ \sigma_{zz} \\ \sigma_{sz} \end{Bmatrix}_{(k)} dn, \tag{A.56}$$

(b) *The transverse shear stress resultants*

$$\begin{Bmatrix} N_{zn} \\ N_{sn} \end{Bmatrix} = \sum_{k=1}^N \int_{n_{(k-1)}}^{n_{(k)}} \begin{Bmatrix} \sigma_{zn} \\ \sigma_{sn} \end{Bmatrix}_{(k)} dn, \tag{A.57}$$

and

(c) *The stress couples*

$$\begin{Bmatrix} L_{zz} \\ L_{sz} \end{Bmatrix} = \sum_{k=1}^N \int_{n_{(k-1)}}^{n_{(k)}} \begin{Bmatrix} \sigma_{zz} \\ \sigma_{sz} \end{Bmatrix}_{(k)} ndn. \tag{A.58}$$

Here $n_{(k)}$ and $n_{(k-1)}$ denote the distances from the middle surface of the cross-section to the upper and lower surfaces of the k -th layer, respectively. The $n_{(k)}$ values are signed numbers varying from $n_{(0)} = -h/2$ to $n_{(N)} = h/2$ in a laminate of N layers and of total thickness h (see Fig. A.2). Equations (A.56) through (A.58) show that the stress resultants and stress couples do not depend on n , but are functions of (s, z) coordinates.

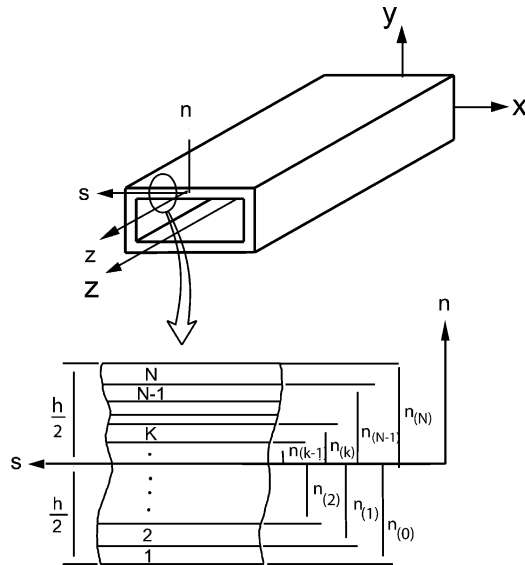


Figure A.2: Geometry of an N -ply laminate

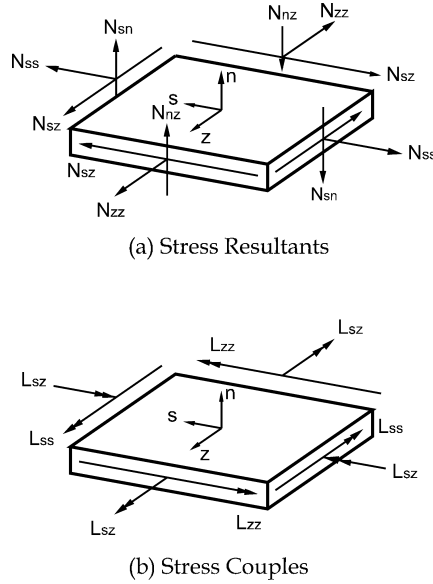


Figure A.3: Stress resultants and stress couples on a beam element

Equations (A.56) through (A.58) involve the assumption that the reference surface of the thin-walled beam is *shallow* in the sense of $h/R_1 \ll 1$, where R_1 denotes the radius of curvature of the mid-surface in the s -direction. The stress resultants have the units of force per unit length, while the stress couples have the units of moment per unit length. The positive sign convention for the stress resultants and stress couples is displayed in Fig. A.3.

The symmetry of the stress tensor and of the shallowness of beam middle surface, implies $N_{sz} = N_{zs}$ and $L_{sz} = L_{zs}$.

A.8-4 A First Step Toward Obtaining the Constitutive Equations of Thin-Walled Open Beams

Upon substituting Eqs. (A.53) into (A.56) through (A.58), we obtain the 2-D constitutive equations. As an example, consider the circumferential (hoop) membrane stress resultant N_{ss} . In conjunction with Eq. (A.53), (see Song, 1990)

$$\begin{aligned}
 N_{ss}(s, z) &= \sum_{k=1}^N \int_{n^{(k-1)}}^{n^{(k)}} \sigma_{ss}^{(k)} dn \\
 &= \sum_{k=1}^N \int_{n^{(k-1)}}^{n^{(k)}} [\bar{Q}_{11}(\epsilon_{ss} - \bar{\alpha}_s T - \bar{\beta}_s M) + \bar{Q}_{12}(\epsilon_{zz} - \bar{\alpha}_z T - \bar{\beta}_z M) \\
 &\quad + \bar{Q}_{16}(\gamma_{sz} - \bar{\alpha}_{sz} T - \bar{\beta}_{sz} M)]_{(k)} dn.
 \end{aligned}
 \tag{A.59}$$

With the definitions of *stretching* and *bending-stretching coupling* stiffness quantities A_{ij} and B_{ij} , respectively

$$A_{ij} = \sum_{k=1}^N \int_{n^{(k-1)}}^{n^{(k)}} \bar{Q}_{ij}^{(k)} dn, \quad B_{ij} = \sum_{k=1}^N \int_{n^{(k-1)}}^{n^{(k)}} \bar{Q}_{ij}^{(k)} n dn, \quad (\text{A.60a,b})$$

as well as those of hygrothermal hoop forces (per unit length)

$$\begin{aligned} N_{ss}^T(s, z) &= \sum_{k=1}^N \int_{n^{(k-1)}}^{n^{(k)}} T(s, z, n) [\bar{Q}_{11}\bar{\alpha}_s + \bar{Q}_{12}\bar{\alpha}_z + \bar{Q}_{16}\bar{\alpha}_{sz}]_{(k)} dn, \\ N_{ss}^M(s, z) &= \sum_{k=1}^N \int_{n^{(k-1)}}^{n^{(k)}} M(s, z, n) [\bar{Q}_{11}\bar{\beta}_s + \bar{Q}_{12}\bar{\beta}_z + \bar{Q}_{16}\bar{\beta}_{sz}]_{(k)} dn, \end{aligned} \quad (\text{A.61a,b})$$

Eq. (A.59) becomes:

$$\begin{aligned} N_{ss}(s, z) &= A_{11}\epsilon_{ss} + A_{12}\epsilon_{zz}^{(0)} + B_{12}\epsilon_{zz}^{(1)} \\ &\quad + A_{16}\gamma_{sz}^{(0)} + 2B_{16}W_M - N_{ss}^T - N_{ss}^M. \end{aligned} \quad (\text{A.62})$$

Proceeding thus for the other stress resultant quantities, we obtain

$$\begin{aligned} \begin{Bmatrix} N_{ss} \\ N_{zz} \\ N_{sz} \end{Bmatrix} &= \begin{bmatrix} A_{11} & A_{12} & A_{16} \\ A_{12} & A_{22} & A_{26} \\ A_{16} & A_{26} & A_{66} \end{bmatrix} \begin{Bmatrix} \epsilon_{ss} \\ \epsilon_{zz}^{(0)} \\ \gamma_{sz}^{(0)} \end{Bmatrix} + \begin{bmatrix} B_{12} & B_{16} \\ B_{22} & B_{26} \\ B_{26} & B_{66} \end{bmatrix} \begin{Bmatrix} \epsilon_{zz}^{(1)} \\ 2W_M \end{Bmatrix} \\ &\quad - \begin{Bmatrix} N_{ss}^T \\ N_{zz}^T \\ N_{sz}^T \end{Bmatrix} - \begin{Bmatrix} N_{ss}^M \\ N_{zz}^M \\ N_{sz}^M \end{Bmatrix}, \end{aligned} \quad (\text{A.63})$$

where, in addition to Eqs. (A.61), we have also

$$\begin{aligned} N_{zz}^T(s, z) &= \sum_{k=1}^N \int_{n^{(k-1)}}^{n^{(k)}} T(s, z, n) [\bar{Q}_{12}\bar{\alpha}_s + \bar{Q}_{22}\bar{\alpha}_z + \bar{Q}_{26}\bar{\alpha}_{sz}]_{(k)} dn, \\ N_{sz}^T(s, z) &= \sum_{k=1}^N \int_{n^{(k-1)}}^{n^{(k)}} T(s, z, n) [\bar{Q}_{16}\bar{\alpha}_s + \bar{Q}_{26}\bar{\alpha}_z + \bar{Q}_{66}\bar{\alpha}_{sz}]_{(k)} dn. \end{aligned} \quad (\text{A.64a,b})$$

N_{zz}^M and N_{sz}^M , are obtainable from Eq (A.64) by applying the substitutions

$$T \rightarrow M; \quad \bar{\alpha}_s \rightarrow \bar{\beta}_s, \quad \bar{\alpha}_z \rightarrow \bar{\beta}_z, \quad \bar{\alpha}_{sz} \rightarrow \bar{\beta}_{sz} \quad (\text{A.65a-d})$$

The transverse shear stress resultants, N_{zn} and N_{sn} , are

$$\begin{aligned} N_{zn}(s, z) &= A_{44}\gamma_{zn} + A_{45}\gamma_{sn}, \\ N_{sn}(s, z) &= A_{45}\gamma_{zn} + A_{55}\gamma_{sn}. \end{aligned} \quad (\text{A.66a,b})$$

By virtue of in-plane cross-section non-deformability implying $\gamma_{sn} = 0$, Eqs. (A.66) reduce to

$$\begin{aligned} N_{zn}(s, z) &= A_{44}\gamma_{zn}, \\ N_{sn}(s, z) &= A_{45}\gamma_{zn}. \end{aligned} \quad (\text{A.67a,b})$$

The transverse shear stiffness quantities A_{LM} are

$$A_{LM} = \sum_{k=1}^N k_{LM}^2 \bar{Q}_{LM}^{(k)} (n^{(k)} - n^{(k-1)}), \quad (L, M = 4, 5) \quad (\text{A.68})$$

where k_{LM}^2 are transverse shear correction factors.

If we assume that transverse shear stresses have a parabolic distribution across the laminate thickness, and define a continuous weighting function (see Vinson and Sierakowski, 2002), we obtain

$$f(n) = \frac{5}{4} \left[1 - \left(\frac{n}{h/2} \right)^2 \right], \quad (\text{A.69})$$

an alternative expression for A_{LM} :

$$A_{LM} = \frac{5}{4} \sum_{k=1}^N \bar{Q}_{LM}^{(k)} \left[n^{(k)} - n^{(k-1)} - \frac{4}{3h^2} (n^{(k)3} - n^{(k-1)3}) \right]. \quad (\text{A.70})$$

For a single layered beam, $N = 1$, $n^{(k)} = h/2$, $n^{(k-1)} = -h/2$, and

$$A_{LM} = \frac{5}{6} \bar{C}_{LM} h \quad (L, M = 4, 5). \quad (\text{A.71})$$

This form can also be obtained from Eq. (A.68), by prescribing Reissner's transverse shear correction factor $k_{LM}^2 \equiv k^2 = 5/6$.

Similarly, the stress couples are expressible in matrix form as:

$$\begin{aligned} \begin{Bmatrix} L_{zz} \\ L_{sz} \end{Bmatrix} &= \begin{bmatrix} B_{12} & B_{22} & B_{26} \\ B_{16} & B_{26} & B_{66} \end{bmatrix} \begin{Bmatrix} \epsilon_{ss} \\ \epsilon_{zz}^{(0)} \\ \gamma_{sz}^{(0)} \end{Bmatrix} + \begin{bmatrix} D_{22} & D_{26} \\ D_{26} & D_{66} \end{bmatrix} \begin{Bmatrix} \epsilon_{zz}^{(1)} \\ 2W_M \end{Bmatrix} \\ &\quad - \begin{Bmatrix} L_{zz}^T \\ L_{sz}^T \end{Bmatrix} - \begin{Bmatrix} L_{zz}^M \\ L_{sz}^M \end{Bmatrix}. \end{aligned} \quad (\text{A.72})$$

In these relationships, both the stiffness quantities B_{ij} and D_{ij} occur. The latter as well as the *thermal and hygric moments* (per unit length) appearing in Eqs. (A.72) are defined as:

$$D_{ij} = \sum_{k=1}^N \int_{n^{(k-1)}}^{n^{(k)}} \bar{Q}_{ij}^{(k)} n^2 dn. \quad (\text{A.73})$$

$$\begin{aligned} L_{zz}^T(s, z) &= \sum_{k=1}^N \int_{n^{(k-1)}}^{n^{(k)}} T(s, z, n) (\bar{Q}_{12} \bar{\alpha}_s + \bar{Q}_{22} \bar{\alpha}_z + \bar{Q}_{26} \bar{\alpha}_{sz})_{(k)} n dn, \\ L_{sz}^T(s, z) &= \sum_{k=1}^N \int_{n^{(k-1)}}^{n^{(k)}} T(s, z, n) (\bar{Q}_{16} \bar{\alpha}_s + \bar{Q}_{26} \bar{\alpha}_z + \bar{Q}_{66} \bar{\alpha}_{sz})_{(k)} n dn, \\ L_{zz}^M(s, z) &= \sum_{k=1}^N \int_{n^{(k-1)}}^{n^{(k)}} M(s, z, n) (\bar{Q}_{12} \bar{\beta}_s + \bar{Q}_{22} \bar{\beta}_z + \bar{Q}_{26} \bar{\beta}_{sz})_{(k)} n dn, \end{aligned} \quad (\text{A.74a-d})$$

$$L_{sz}^M(s, z) = \sum_{k=1}^N \int_{n^{(k-1)}}^{n^{(k)}} M(s, z, n) (\bar{Q}_{16} \bar{\beta}_s + \bar{Q}_{26} \bar{\beta}_z + \bar{Q}_{66} \bar{\beta}_{sz})_{(k)} n dn.$$

A.8-5 Remarks on Stiffness Quantities

Equations (A.63), (A.67), and (A.72) are the 2-D thermoelastic constitutive equations for the laminated composite anisotropic thin-walled beams. In their expression, in contrast to their isotropic single-layer counterpart, a variety of coupling effects are present. These couplings constitute new degrees of freedom in the hands of the designer. Among them, the stiffness quantities B_{ij} induce a coupling between membrane stress resultants (see Eqs. (A.63)) and the extension strain $\epsilon_{zz}^{(1)}$ varying through the beam thickness, and the warping W_M , and between the stress couples (see Eqs. (A.72)) and the in-plane strain components.

The stiffness terms A_{16} , A_{26} introduce a coupling between the circumferential (hoop) and the axial membrane stress resultants and the membrane shear strain components. Since the material properties are piecewise constant through the thickness of the beam, the stiffness quantities can be expressed as:

$$\begin{aligned} A_{ij} &= \sum_{k=1}^N \bar{Q}_{ij}^{(k)} [n^{(k)} - n^{(k-1)}], \quad B_{ij} = \frac{1}{2} \sum_{k=1}^N \bar{Q}_{ij}^{(k)} [n_{(k)}^2 - n_{(k-1)}^2], \\ D_{ij} &= \frac{1}{3} \sum_{k=1}^N \bar{Q}_{ij}^{(k)} [n_{(k)}^3 - n_{(k-1)}^3]. \end{aligned} \quad (\text{A.75a-c})$$

For uniform thermal and moisture fields throughout the thickness [i.e. when $T \equiv T(s, z)$ and $M \equiv M(s, z)$], the expressions for the associated thermal and hygric forces and moments, become

$$\begin{aligned}
 N_{ss}^T(s, z) &= T(s, z) \sum_{k=1}^N \{ \bar{Q}_{11} \bar{\alpha}_s + \bar{Q}_{12} \bar{\alpha}_z + \bar{Q}_{16} \bar{\alpha}_{sz} \}_{(k)} [n_{(k)} - n_{(k-1)}], \\
 N_{zz}^T(s, z) &= T(s, z) \sum_{k=1}^N \{ \bar{Q}_{12} \bar{\alpha}_s + \bar{Q}_{22} \bar{\alpha}_z + \bar{Q}_{26} \bar{\alpha}_{sz} \}_{(k)} [n_{(k)} - n_{(k-1)}], \\
 N_{sz}^T(s, z) &= T(s, z) \sum_{k=1}^N \{ \bar{Q}_{16} \bar{\alpha}_s + \bar{Q}_{26} \bar{\alpha}_z + \bar{Q}_{66} \bar{\alpha}_{sz} \}_{(k)} [n_{(k)} - n_{(k-1)}], \\
 L_{zz}^T(s, z) &= \frac{1}{2} T(s, z) \sum_{k=1}^N \{ \bar{Q}_{11} \bar{\alpha}_s + \bar{Q}_{12} \bar{\alpha}_z + \bar{Q}_{16} \bar{\alpha}_{sz} \}_{(k)} [n_{(k)}^2 - n_{(k-1)}^2], \\
 L_{sz}^T(s, z) &= \frac{1}{2} T(s, z) \sum_{k=1}^N \{ \bar{Q}_{16} \bar{\alpha}_s + \bar{Q}_{26} \bar{\alpha}_z + \bar{Q}_{66} \bar{\alpha}_{sz} \}_{(k)} [n_{(k)}^2 - n_{(k-1)}^2].
 \end{aligned} \tag{A.76a-e}$$

N_{ss}^M , N_{zz}^M , N_{sz}^M , L_{zz}^M and L_{sz}^M , are obtainable from their thermal counterparts, Eqs. (A.76), by using the substitutions, Eq. (A.65). The laminate stacking sequence code specifies the ply composition, the exact location and sequence of various plies.

For an unsymmetric laminate this prescription is done from the bottom of the laminate, to the top, while for a symmetric laminate, from the middle surface to the top surface. Several standard notations are given next: $[0_3^o/90_2^o/45^o/45^o/90_2^o/0_3^o]_T \equiv [0_3^o/90_2^o/45^o]_S$, where subscripts T and S stand for total number of plies and symmetric sequence respectively, $[\pm\theta^o]_n = [+ \theta^o / - \theta^o / + \theta^o / - \theta^o \dots]$; $[\pm 45^o]_n = [+45^o / -45^o / +45^o / -45^o]$, where, the subscript n indicates that the sub-laminate $[\pm\theta^o]$ is repeated n times.

There are a number of special laminate configurations that, in addition to their technological importance, lead to simplifications of the 2-D constitutive law. These simplifications depend basically on the *odd* and *even* character of stiffness \bar{Q}_{ij} and of thermal $\bar{\alpha}_i$ and hygrothermal $\bar{\beta}_i$ expansion coefficients, as well as on the stacking sequence characteristics of the laminate. The odd and even features of \bar{Q}_{ij} , $\bar{\alpha}_i$, and $\bar{\beta}_i$ (i.e. the change and the invariance in sign of the respective quantities respectively, when ply-angle $\theta \rightarrow -\theta$) can easily be revealed when using Eqs. (A.32) through (A.34) and are summarized by

$$\bar{Q}_{ij}(\theta) = \begin{cases} -\bar{Q}_{ij}(-\theta) & \text{if } (ij) = (16), (26), (45), (36) \\ \bar{Q}_{ij}(-\theta) & \text{if } (ij) = (11), (12), (13), (23), (22), (44), (55), (66) \end{cases}$$

$$\bar{\alpha}_i(\theta) = \begin{cases} -\bar{\alpha}_i(-\theta) & \text{if } i = 6 \\ \bar{\alpha}_i(-\theta) & \text{if } i = 1, 2, 3 \end{cases} \quad (\text{A.77a,b})$$

whereas for $\bar{\beta}_i(\theta)$ a similar behavior as for $\bar{\alpha}_i(\theta)$ is obtained. Based on the odd and even properties of a generally orthotropic material, special classes of laminate configurations can be distinguished.

A.8-6 Selected Classes of Laminate Configurations

(a) Symmetric Laminates

This case requires both geometric and material symmetry properties as well as symmetry of the hygrothermal distributions about the middle surface of the beam. For the conditions of symmetry stated above,

$$B_{ij} = 0. \quad (\text{A.78})$$

For the same case, when the thermal and hygrothermal fields are uniform across the laminate thickness, Eqs. (A.74) imply

$$L_{zz}^T = L_{sz}^T = L_{zz}^M = L_{sz}^M = 0, \quad (\text{A.79})$$

which means that there are no hygrothermal induced moments (per unit length).

(b) Balanced Laminates

To every lamina with ply-orientation $+\theta$, there exists a companion laminae of identical material properties and thickness of orientation $-\theta$ somewhere within the laminate. For this case

$$A_{16} = A_{26} = 0, \quad (\text{A.80a})$$

and

$$N_{sz}^T = N_{sz}^M = 0. \quad (\text{A.80b})$$

For a balanced and symmetric laminate, $B_{ij}=0$, $A_{16} = A_{26} = 0$, $N_{sz}^T = N_{sz}^M = 0$ and $L_{sz}^T = L_{sz}^M = 0$. It clearly appears that the use of *unbalanced* angle-ply laminates (i.e, of such laminates for which A_{16} , A_{26} and D_{26} are different from zero), is essential when the tailoring technique should be implemented. A balanced laminate can be symmetric, antisymmetric or asymmetric.

(c) Antisymmetric Laminates

For this case, to every lamina with orientation $+\theta$ and location n with respect to the middle surface of the beam, there is another lamina with

orientation $-\theta$ at the location $-n$. This laminate consists of an even number of plies. This configuration yields the simplification

$$A_{16} = A_{26} = D_{16} = D_{26} = 0. \tag{A.81}$$

Moreover, if the antisymmetric laminate consist of pairs of $+\theta_i$ and $-\theta_i$ orientations, symmetrically located about the middle surface and having the same thickness and elastic properties, besides Eq. (A.81), we have

$$B_{11} = B_{22} = B_{12} = B_{66} = 0. \tag{A.82}$$

Additional simplifications can occur for cross-ply laminated composite thin-walled beam structures. For this case the reader is referred to the monographs by Daniel and Ishai (1994), Bogdanovich and Pastore (1996), Vinson and Sierakowski (2002) and Reddy (2004), where other ply sequence configurations of practical importance are also considered.

A.8-7 Equations for Open Cross-Section Beams

One of the assumptions in general use within the theory of thin-walled beams is that the stress component σ_{ss} is negligible compared with the remaining stress components. This means that N_{ss} can be assumed negligible in the constitutive equations. In light of this assumption and in conjunction with Eq. (A.62)

$$\epsilon_{ss} = \frac{1}{A_{11}}(-A_{12}\epsilon_{zz}^{(0)} - A_{16}\gamma_{sz}^{(0)} - 2B_{16}W_M - B_{12}\epsilon_{zz}^{(1)} + N_{ss}^T + N_{ss}^M). \tag{A.83}$$

Replacement of Eq.(A.83) into the constitutive equations (A.63b,c) yields the final form of 2-D constitutive equations for thin-walled open beams:

$$\left\{ \begin{matrix} N_{zz} \\ N_{sz} \end{matrix} \right\} = \begin{bmatrix} K_{11} & K_{12} & K_{13} & K_{14} \\ K_{21} & K_{22} & K_{23} & K_{24} \end{bmatrix} \left\{ \begin{matrix} \epsilon_{zz}^{(0)} \\ \gamma_{sz}^{(0)} \\ W_M \\ \epsilon_{zz}^{(1)} \end{matrix} \right\} - \left\{ \begin{matrix} \hat{N}_{zz}^T \\ \hat{N}_{sz}^T \end{matrix} \right\} - \left\{ \begin{matrix} \hat{N}_{zz}^M \\ \hat{N}_{sz}^M \end{matrix} \right\},$$

$$\left\{ \begin{matrix} L_{zz} \\ L_{sz} \end{matrix} \right\} = \begin{bmatrix} K_{41} & K_{42} & K_{43} & K_{44} \\ K_{51} & K_{52} & K_{53} & K_{54} \end{bmatrix} \left\{ \begin{matrix} \epsilon_{zz}^{(0)} \\ \gamma_{sz}^{(0)} \\ W_M \\ \epsilon_{zz}^{(1)} \end{matrix} \right\} - \left\{ \begin{matrix} \hat{L}_{zz}^T \\ \hat{L}_{sz}^T \end{matrix} \right\} - \left\{ \begin{matrix} \hat{L}_{zz}^M \\ \hat{L}_{sz}^M \end{matrix} \right\}. \tag{A.84a,b}$$

The constitutive equations (A.67a,b) associated with the transverse shear stress resultants, remain unchanged. Within this variant of constitutive equations, the modified stiffness quantities K_{ij} are defined by:

$$\begin{aligned}
 K_{11} &\equiv A_{22} - A_{12}^2/A_{11}, \\
 K_{12} &\equiv A_{26} - A_{12}A_{16}/A_{11} = K_{21}, \\
 K_{13} &\equiv 2(B_{26} - A_{12}B_{16}/A_{11}), \\
 K_{14} &\equiv B_{22} - A_{12}B_{12}/A_{11} = K_{41}, \\
 K_{22} &\equiv A_{66} - A_{16}^2/A_{11}, \\
 K_{23} &\equiv 2(B_{66} - A_{16}B_{16}/A_{11}), \\
 K_{24} &\equiv B_{26} - A_{16}B_{12}/A_{11} = K_{42}, \\
 K_{43} &\equiv 2(D_{26} - B_{12}B_{16}/A_{11}), \\
 K_{44} &\equiv D_{22} - B_{12}^2/A_{11}, \\
 K_{51} &\equiv B_{26} - B_{16}A_{12}/A_{11}, \\
 K_{52} &\equiv B_{66} - B_{16}A_{16}/A_{11}, \\
 K_{53} &\equiv 2(D_{66} - B_{16}^2/A_{11}), \\
 K_{54} &\equiv D_{26} - B_{12}B_{16}/A_{11},
 \end{aligned} \tag{A.85a-m}$$

while the modified hygrothermal forces and moments are:

$$\begin{aligned}
 \hat{N}_{zz}^T &\equiv N_{zz}^T - A_{12}N_{ss}^T/A_{11}, \\
 \hat{N}_{zz}^M &\equiv N_{zz}^M - A_{12}N_{ss}^M/A_{11}, \\
 \hat{N}_{sz}^T &\equiv N_{sz}^T - A_{16}N_{ss}^T/A_{11}, \\
 \hat{N}_{sz}^M &\equiv N_{sz}^M - A_{16}N_{ss}^M/A_{11}, \\
 \hat{L}_{zz}^T &\equiv L_{zz}^T - B_{12}N_{ss}^T/A_{11}, \\
 \hat{L}_{zz}^M &\equiv L_{zz}^M - B_{12}N_{ss}^M/A_{11}, \\
 \hat{L}_{sz}^T &\equiv L_{sz}^T - B_{16}N_{ss}^T/A_{11}, \\
 \hat{L}_{sz}^M &\equiv L_{sz}^M - B_{16}N_{ss}^M/A_{11}.
 \end{aligned} \tag{A.86a-h}$$

A.8-8 2-D Constitutive Equations for Closed Cross-Section Beams

The procedure of deriving the constitutive relationships for closed cross-section beams is similar to that used for open beams. For thin-walled beams of closed cross-section, the difference lies in the expression of the torsional function. In conjunction with Eq. (2.1-62) the expression of the shear strain component is

$$\gamma_{sz} = \gamma_{sz}^{(0)} + \gamma_{sz}^{(t)}, \tag{A.87a}$$

where

$$\gamma_{sz}^{(0)}(s, z) = \gamma_{xz}(z) \frac{dx}{ds} + \gamma_{yz}(z) \frac{dy}{ds}, \quad (\text{A.87b})$$

and

$$\gamma_{sz}^{(t)} = \Psi W_M. \quad (\text{A.87c})$$

Herein

$$\Psi \equiv \psi + 2n; \quad W_M \equiv \phi', \quad (\text{A.88a,b})$$

where ψ is the torsional function that is expressed by Eq. (2.1-44c) considered in conjunction with (2.1-36c) and (2.1-39a), or in an inclusive form by Eq. (9.1-2).

Proceeding as for open cross-section beams, and assuming that h and G_{sz} are uniform along the beam circumference we obtain the following 2-D constitutive equations

$$\begin{aligned} \begin{Bmatrix} N_{ss} \\ N_{zz} \\ N_{sz} \end{Bmatrix} &= \begin{bmatrix} A_{11} & A_{12} & A_{16} \\ A_{12} & A_{22} & A_{26} \\ A_{16} & A_{26} & A_{66} \end{bmatrix} \begin{Bmatrix} \epsilon_{ss} \\ \epsilon_{zz}^{(0)} \\ \gamma_{sz}^{(0)} \\ + 2\frac{\Omega}{\beta} W_M \end{Bmatrix} \\ &+ \begin{Bmatrix} B_{12} \\ B_{22} \\ B_{26} \end{Bmatrix} \epsilon_{zz}^{(1)} + 2 \begin{Bmatrix} B_{16} \\ B_{26} \\ B_{66} \end{Bmatrix} W_M - \begin{Bmatrix} N_{ss}^T \\ N_{zz}^T \\ N_{sz}^T \end{Bmatrix} - \begin{Bmatrix} N_{ss}^M \\ N_{zz}^M \\ N_{sz}^M \end{Bmatrix}, \end{aligned}$$

$$N_{zn} = A_{44} \gamma_{zn}, \quad (\text{A.89a-d})$$

$$N_{sn} = A_{45} \gamma_{zn},$$

$$\begin{aligned} \begin{Bmatrix} L_{zz} \\ L_{sz} \end{Bmatrix} &= \begin{bmatrix} B_{12} & B_{22} & B_{26} \\ B_{16} & B_{26} & B_{66} \end{bmatrix} \begin{Bmatrix} \epsilon_{ss} \\ \epsilon_{zz}^{(0)} \\ \gamma_{sz}^{(0)} \\ + 2\frac{\Omega}{\beta} W_M \end{Bmatrix} + \begin{Bmatrix} D_{22} \\ D_{66} \end{Bmatrix} \epsilon_{zz}^{(1)} \\ &+ 2 \begin{Bmatrix} D_{26} \\ D_{66} \end{Bmatrix} W_M - \begin{Bmatrix} L_{zz}^T \\ L_{sz}^T \end{Bmatrix} - \begin{Bmatrix} L_{zz}^M \\ L_{sz}^M \end{Bmatrix}. \end{aligned}$$

Here

$$\Omega \equiv \frac{1}{2} \oint r_n ds, \quad \beta \equiv \oint ds. \quad (\text{A.89e,f})$$

Note that the warping measure W_M strongly intervenes in the constitutive equations of both open and closed cross-section beams.

Remark:

One should observe that due to cross-section nondeformability implying that $\gamma_{sn} = 0$, the term $N_{sn} \gamma_{sn}$ in the strain energy becomes immaterial. For this reason, the stiffness A_{45} does not appear in the 1-D stiffnesses a_{ij} .

A.8-9 Final Form of 2-D Constitutive Equations

The smallness of the hoop stress resultant N_{ss} combined with Eq (A.89a) yields

$$\begin{aligned} \epsilon_{ss} = & -\frac{1}{A_{11}} \left(A_{12} \epsilon_{zz}^{(0)} + A_{16} \gamma_{sz}^{(0)} + 2A_{16} \frac{\Omega}{\beta} W_M \right. \\ & \left. + 2B_{16} W_M + B_{12} \epsilon_{zz}^{(1)} - N_{ss}^T - N_{ss}^M \right). \end{aligned} \quad (\text{A.90})$$

Replacing Eq. (A.90) in Eqs. (A.89a,d) leads to

$$\begin{aligned} \begin{Bmatrix} N_{zz} \\ N_{sz} \end{Bmatrix} &= \begin{bmatrix} K_{11} & K_{12} & K_{13} & K_{14} \\ K_{21} & K_{22} & K_{23} & K_{24} \end{bmatrix} \begin{Bmatrix} \epsilon_{zz}^{(0)} \\ \gamma_{sz}^{(0)} \\ W_M \\ \epsilon_{zz}^{(1)} \end{Bmatrix} - \begin{Bmatrix} \hat{N}_{zz}^T \\ \hat{N}_{sz}^T \end{Bmatrix} - \begin{Bmatrix} \hat{N}_{zz}^M \\ \hat{N}_{sz}^M \end{Bmatrix}, \\ \begin{Bmatrix} L_{zz} \\ L_{sz} \end{Bmatrix} &= \begin{bmatrix} K_{41} & K_{42} & K_{43} & K_{44} \\ K_{51} & K_{52} & K_{53} & K_{54} \end{bmatrix} \begin{Bmatrix} \epsilon_{zz}^{(0)} \\ \gamma_{sz}^{(0)} \\ W_M \\ \epsilon_{zz}^{(1)} \end{Bmatrix} - \begin{Bmatrix} \hat{L}_{zz}^T \\ \hat{L}_{sz}^T \end{Bmatrix} - \begin{Bmatrix} \hat{L}_{zz}^M \\ \hat{L}_{sz}^M \end{Bmatrix}. \end{aligned} \quad (\text{A.91a,b})$$

As expected, the constitutive equations (A.89b,c) remain unchanged. The stiffness quantities that are affected by this change are K_{13} , K_{23} , K_{43} , and K_{53} . Their expressions are

$$\begin{aligned} K_{13} &= 2 \left(A_{26} - \frac{A_{12}A_{16}}{A_{11}} \right) \frac{\Omega}{\beta} + 2 \left(B_{26} - \frac{A_{12}B_{16}}{A_{11}} \right), \\ K_{23} &= 2 \left(A_{66} - \frac{A_{16}^2}{A_{11}} \right) \frac{\Omega}{\beta} + 2 \left(B_{66} - \frac{A_{16}B_{16}}{A_{11}} \right), \\ K_{43} &= 2 \left(B_{26} - \frac{B_{12}A_{16}}{A_{11}} \right) \frac{\Omega}{\beta} + 2 \left(D_{26} - \frac{B_{12}B_{16}}{A_{11}} \right), \\ K_{53} &= 2 \left(B_{66} - \frac{B_{16}A_{16}}{A_{11}} \right) \frac{\Omega}{\beta} + 2 \left(D_{66} - \frac{B_{16}^2}{A_{11}} \right). \end{aligned} \quad (\text{A.92a-d})$$

The remaining stiffness quantities K_{ij} defined by Eq. (A.85) remain unaltered. Note that the constitutive equations for multicell closed section beams are similar to those for single cell counterparts. Thus, Eqs.(A.91) and (A.89b,c) can be applied to multicell beam structures as well.

A.8-10 Unified Form of 2-D Constitutive Equations

A unified expression of 2-D constitutive equations, applicable to both open and closed cross-section beams will be displayed next. This is given by

$$\begin{Bmatrix} N_{zz} \\ N_{sz} \\ L_{zz} \\ L_{sz} \\ N_{zn} \\ N_{sn} \end{Bmatrix} = \begin{bmatrix} K_{11} & K_{12} & K_{13} & K_{14} & 0 \\ K_{21} & K_{22} & K_{23} & K_{24} & 0 \\ K_{41} & K_{42} & K_{43} & K_{44} & 0 \\ K_{51} & K_{52} & K_{53} & K_{54} & 0 \\ 0 & 0 & 0 & 0 & A_{44} \\ 0 & 0 & 0 & 0 & A_{45} \end{bmatrix} \begin{Bmatrix} \epsilon_{zz}^{(0)} \\ \gamma_{sz}^{(0)} \\ W_M \\ \epsilon_{zz}^{(1)} \\ \gamma_{zn} \end{Bmatrix} - \begin{Bmatrix} \hat{N}_{zz}^T \\ \hat{N}_{sz}^T \\ \hat{L}_{zz}^T \\ \hat{L}_{sz}^T \\ 0 \\ 0 \end{Bmatrix} - \begin{Bmatrix} \hat{N}_{zz}^M \\ \hat{N}_{sz}^M \\ \hat{L}_{zz}^M \\ \hat{L}_{sz}^M \\ 0 \\ 0 \end{Bmatrix}. \tag{A.93}$$

With the exception of K_{13} , K_{23} , K_{43} , and K_{53} whose expressions are different for open and closed cross-section beams, the remaining stiffness quantities are formally similar for both cases; the constitutive equations, Eqs. (A.93), can be used for both open and closed section beams.

The constitutive equations of thin-walled beam theory used in the literature are also displayed here. In present notations, these are

$$\begin{Bmatrix} N_{zz} \\ N_{sz} \end{Bmatrix} = \begin{bmatrix} K_{11} & K_{12} \\ K_{21} & K_{22} \end{bmatrix} \begin{Bmatrix} \epsilon_{zz}^{(0)} \\ \gamma_{sz}^{(0)} \end{Bmatrix}. \tag{A.94}$$

Clearly, Eqs. (A.93) are much more comprehensive than (A.94) that do not include bending and the thickness-wise contributions, and as such, are likely to describe more accurately the behavior of thin-walled beams.

A.8-11 Two Structural Coupling Configurations – CUS and CAS

We consider two structural configurations which exhibit special structural couplings. Considered first by Rehfield and Atilgan (1989), these structural configurations are referred to as *circumferentially uniform stiffness (CUS)* and *circumferentially asymmetric stiffness (CAS)* configurations. For a thin-walled beam of rectangular cross-section, (CUS) implies the ply-angle distribution $\theta(y) = \theta(-y)$ of the top and bottom walls of the box beam (*flanges*) and of $\theta(x) = \theta(-x)$ of the lateral walls (*webs*). The latter one, (CAS), implies $\theta(y) = -\theta(-y)$, and $\theta(x) = -\theta(-x)$ (see Fig. A.4).

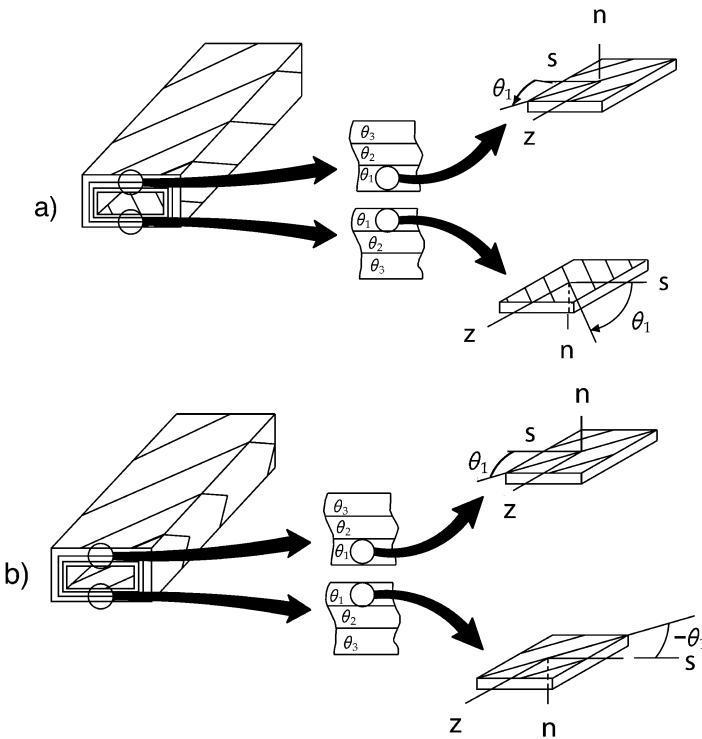


Figure A.4: a) Circumferentially uniform stiffness (CUS) configuration. b) Circumferentially asymmetric stiffness (CAS) configuration

For the CAS configuration (see Eq. A.77), the stiffness quantities \bar{Q}_{16} , \bar{Q}_{26} , \bar{Q}_{36} , \bar{Q}_{45} of each of the layers in the bottom flanges are the negative value of the layers counterpart in the top flanges, that is:

$$\bar{Q}_{16}^{(U)} = -\bar{Q}_{16}^{(L)}; \bar{Q}_{26}^{(U)} = -\bar{Q}_{26}^{(L)}; \bar{Q}_{36}^{(U)} = -\bar{Q}_{36}^{(L)}; \bar{Q}_{45}^{(U)} = -\bar{Q}_{45}^{(L)}. \tag{A.95a-d}$$

This implies that in this case the following relationships hold

$$\begin{aligned} A_{16}^{(U)} &= -A_{16}^{(L)}; A_{26}^{(U)} = -A_{26}^{(L)}; A_{36}^{(U)} = -A_{36}^{(L)}; A_{45}^{(U)} = -A_{45}^{(L)}; \\ D_{16}^{(U)} &= -D_{16}^{(L)}; D_{26}^{(U)} = -D_{26}^{(L)}; D_{36}^{(U)} = -D_{36}^{(L)}; D_{45}^{(U)} = -D_{45}^{(L)}. \end{aligned} \quad (\text{A.96a-h})$$

Similar types of relationships appear also in the opposite webs of the beam.

For CUS, the stiffness quantities \bar{C}_{16} , \bar{C}_{26} , \bar{C}_{36} , \bar{C}_{45} in the opposite members exhibit the same sign, a trend which is valid also for the same components of the matrices $[A]$ and $[D]$. The CUS and CAS configurations are also referred to as the *antisymmetric* and *symmetric* configurations, respectively, see Smith and Chopra (1991).

Similar ply-angle configurations can be implemented in open beams, as well. As an example, in Figs. (13.1-1a) and (13.1-1b), the CAS and CUS are shown in a I beam.

A.8-12 Additional Remarks

The derivation of 2-D form of the constitutive equations is but an intermediate step. The goal is to convert these constitutive equations to a 1-D form, where only the single independent variable z (representing the longitudinal axis of the beam) will be involved. This step is carried out in Chapter 3.

A.9 PIEZOELECTRIC CONSTITUTIVE EQUATIONS

A.9-1 Preliminaries

As is well known, piezoelectric materials generate an electrical charge in response to a mechanical deformation or, conversely, provide mechanical strain when an electric field is applied across them. As a result, piezoelectric materials are excellent candidates for the role of sensors and actuators in the technology of smart structures (see e.g. Bailey and Hubbard, 1985; Tzou and Zhong, 1992; Tzou, 1993; Tzou and Anderson, 1992; Hanagud et al., 1992; Sunar and Rao, 1999; Chopra, 2002). In applications, piezoceramic materials are sandwiched between conductive surfaces (electrodes) and polarized in a suitable direction (see Fig. A.5). To model such structures, the constitutive equations of piezoelectric media have to be considered. In the sequel, several basic elements concerning these equations will be presented; for details, see e.g. Maugin (1988) and Eringen and Maugin (1990).

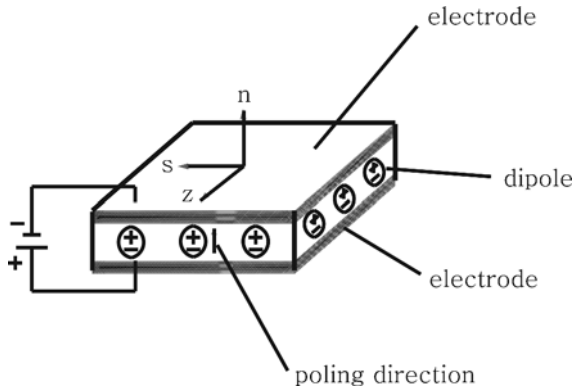


Figure A.5: Actuator layer

A.9-2 Piezoelectric Medium

In contracted indicial notations, the constitutive equations of a 3-D piezoelectric continuum are

$$\begin{aligned}\sigma_i &= C_{ij}^{\mathcal{E}} S_j - e_{ri} \mathcal{E}_r, \\ D_r &= e_{rj} S_j + \epsilon_{r\ell}^S \mathcal{E}_\ell, \quad (r, \ell = 1, 2, 3)\end{aligned}\quad (\text{A.97a,b})$$

In Eqs. (A.97) σ_i and S_j ($i, j = \overline{1, 6}$) denote the stress and strain components, respectively, where

$$S_j = \begin{cases} S_{pr} \text{ for } p = r, j = 1, 2, 3 \\ 2S_{pr} \text{ for } p \neq r, j = 4, 5, 6. \end{cases} \quad (\text{A.98})$$

$C_{ij}^{\mathcal{E}}$, e_{ri} and $\epsilon_{r\ell}^S$ denote the elastic coefficients (measured under constant electric field), the piezoelectric tensor and the dielectric tensor (measured under constant strain), while \mathcal{E}_r and D_r denote the electric field intensity and electric displacement vector, respectively. In Eqs. (A.97) the summation convention over repeated indices is implied.

While Eq. (A.97a) describes the **converse** piezoelectric effect consisting of the generation of mechanical stress or strain when an electric field is applied, Eq. (A.97b) describes the **direct** piezoelectric effect consisting of generation of an electrical charge under a mechanical force. In a piezoelectric adaptive structure the direct effect is used for distributed sensing, while the converse effect is used for the active distributed control.

Equations (A.97) are valid for the most general case of anisotropy, i.e., for triclinic crystals. In this case, the piezoelectric continuum is characterized by 21 independent elastic constants, 18 independent piezoelectric constants, and 6 independent dielectric constants.

For a thickness-polarized continuum, the constitutive equations are the same as for a piezocrystal of the hexagonal $6mm$ symmetry class. This represents a transversely isotropic piezoelectric, the n -axis being a sixfold axis of symmetry.

In this case (see Eringen and Maugin, 1990), the piezoelectric continuum is characterized by five independent elastic coefficients $C_{11} = C_{22}$, $C_{13} = C_{23}$, C_{33} , $C_{44} = C_{55}$, $C_{66}(\equiv (C_{11} - C_{12})/2)$, three independent piezoelectric coefficients $e_{15} = e_{24}$, $e_{31} = e_{32}$ and e_{33} and two independent dielectric constants, $\epsilon_{11} = \epsilon_{22}$ and ϵ_{33} . We consider only this case.

The electric field vector \mathcal{E} is represented in terms of the transverse normal component \mathcal{E}_3 (implying that $\mathcal{E}_1 = \mathcal{E}_2 = 0$).

Note due to the uniform voltage distribution, \mathcal{E}_3 is independent of space (but possibly dependent on time).

A.9-3 2-D Piezoelectric Constitutive Equations

We will assume that the master (or host) structure is composed of m layers, while the actuator is composed of ℓ piezoelectric layers. The actuators can be distributed over the entire span of the beam (see Figs. 6.6-1a and 9.2-1), or can be in the form of patches. Along the circumferential s , spanwise z and thickness n directions, they are distributed according to the law (see Fig. A.6):

$$\begin{aligned} R_{(k)}(n) &= Y(n - n_{(k-1)}) - Y(n - n_{(k)}), \\ R_{(k)}(s) &= Y(s - s_{k1}) - Y(s - s_{k2}), \\ R_{(k)}(z) &= Y(z - z_{k1}) - Y(z - z_{k2}), \end{aligned} \tag{A.99a-c}$$

where $Y(\cdot)$ denotes Heaviside's distribution, $R(\cdot)$ is a spatial function describing the distribution of actuator patches, while $(n_{(k-1)}, n_{(k)})$, (s_{k1}, s_{k2}) and (z_{k1}, z_{k2}) denote the top and bottom heights of the piezopatch measured across the beam thickness, and its location along the beam circumference and span, respectively.

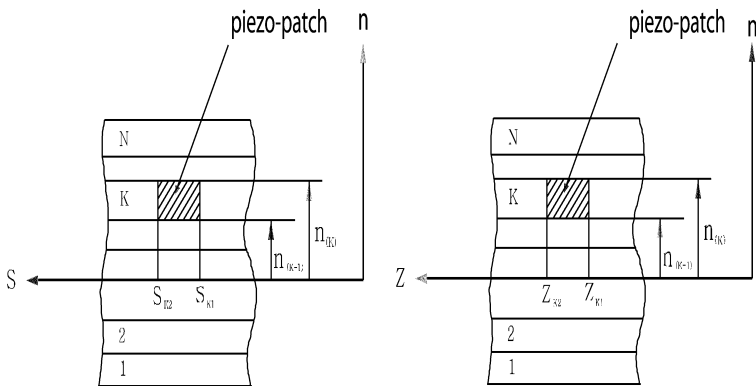


Figure A.6: Piezopatch location.

Proceeding as for Eq. (A.53), we find the 3-D constitutive equations for the transversely isotropic piezoelectric layers expressed in the coordinates (s, z, n) :

$$\begin{Bmatrix} \sigma_{ss} \\ \sigma_{zz} \\ \sigma_{sz} \end{Bmatrix}_{(k)} = \begin{bmatrix} \bar{Q}_{11} & \bar{Q}_{12} & 0 \\ \bar{Q}_{12} & \bar{Q}_{11} & 0 \\ 0 & 0 & \bar{Q}_{66} \end{bmatrix}_{(k)} \begin{Bmatrix} S_{ss} \\ S_{zz} \\ S_{sz} \end{Bmatrix}_{(k)} - \begin{Bmatrix} \bar{e}_{31}^{(k)} \mathcal{E}_3^{(k)} R_{(k)}(n) R_{(k)}(s) R_{(k)}(z) \\ \bar{e}_{31}^{(k)} \mathcal{E}_3^{(k)} R_{(k)}(n) R_{(k)}(s) R_{(k)}(z) \\ 0 \end{Bmatrix}, \tag{A.100a}$$

$$\sigma_{zn}^{(k)} = C_{44}^{(k)} S_{zn}^{(k)}. \tag{A.100b}$$

where

$$\begin{aligned} \bar{Q}_{11} = \bar{Q}_{22} &= \frac{E}{1 - \nu^2}; \quad \bar{Q}_{12} = \frac{E\nu}{1 - \nu^2}, \\ \bar{Q}_{66} = G_{12} &= \frac{E}{2(1 + \nu)}; \quad C_{44} = G_{23} \equiv G', \end{aligned} \tag{A.101a-d}$$

$$\bar{e}_{31} = e_{31} - \frac{E\nu'}{E'(1 - \nu)} e_{33}. \tag{A.101e}$$

In Eq. (A.100a), the last terms identify the actuation stresses induced by the applied electric field.

The 2-D stress resultants and stress couples of the global structure can be obtained via the integration of the 3-D stress components through the total laminate thickness, i.e. through the thickness of the host structure and of piezoelectric layers. Assuming that the hoop stress is small, and replacing the notation for the strain components used in the piezoelectricity theory by the standard one in frequent usage in beam theory (see Eqs. (A.91)), we express the 2-D stress-resultants and stress couples as:

$$\begin{Bmatrix} N_{zz} \\ N_{sz} \end{Bmatrix} = \begin{bmatrix} K_{11} & K_{12} & K_{13} & K_{14} \\ K_{21} & K_{22} & K_{23} & K_{24} \end{bmatrix} \begin{Bmatrix} \epsilon_{zz}^{(0)} \\ \gamma_{sz}^{(0)} \\ W_M \\ \epsilon_{zz}^{(1)} \end{Bmatrix} - \begin{Bmatrix} \tilde{N}_{zz} \\ 0 \end{Bmatrix}, \tag{A.102a,b}$$

$$\begin{Bmatrix} L_{zz} \\ L_{sz} \end{Bmatrix} = \begin{bmatrix} K_{41} & K_{42} & K_{43} & K_{44} \\ K_{51} & K_{52} & K_{53} & K_{54} \end{bmatrix} \begin{Bmatrix} \epsilon_{zz}^{(0)} \\ \gamma_{sz}^{(0)} \\ W_M \\ \epsilon_{zz}^{(1)} \end{Bmatrix} - \begin{Bmatrix} \tilde{L}_{zz} \\ 0 \end{Bmatrix},$$

and

$$N_{zn} = A_{44}\gamma_{zn}, \quad N_{sn} = A_{45}\gamma_{zn}. \quad (\text{A.102c,d})$$

In these equations K_{ij} denote the modified local stiffness coefficients of the adaptive structure, while \tilde{N}_{zz} and \tilde{L}_{zz} denote the piezoelectrically induced stress resultant and stress couple, respectively. Their expressions are:

$$\begin{aligned} \tilde{N}_{zz}(s, z) &= \mathcal{E}_3 \left(1 - \frac{A_{12}}{A_{11}} \right) (n_2 - n_1) \bar{e}_{31} R(s, z), \\ \tilde{L}_{zz}(s, z) &= \mathcal{E}_3 \bar{e}_{31}(s, z) (n_2 - n_1) \left[\frac{1}{2} (n_1 + n_2) - \frac{B_{12}}{A_{11}} \right] R(s, z). \end{aligned} \quad (\text{A.103a,b})$$

Note that in Eqs. (A.102) and (A.103), the stiffness quantities A_{ij} , B_{ij} and D_{ij} as given by Eqs.(A.75) and K_{ij} by Eqs. (A.85) and (A.92), have to be based on the **total** number of constituent layers $N = m + \ell$, for the host (m) and the piezoelectric layers (ℓ). One additional remark is in order here: while the material of the host structure can exhibit any type of anisotropy, the piezoactuators/sensors as considered here feature transversely isotropic properties, the surface of isotropy being parallel at each point to the beam mid-surface.

In Eqs. (A.103) the following notation was used:

$$R_{(k)}(s, z) \equiv R_{(k)}(s) R_{(k)}(z). \quad (\text{A.104})$$

For $z_1 = 0$ and $z_2 = L$, the piezoactuator is spread over the entire beam span.

REFERENCES

- Bailey, T. and Hubbard, J. E., Jr. (1985) "Distributed Piezoelectric-Polymer Active Vibration Control of a Cantilever Beam," *Journal of Guidance, Control and Dynamics*, Vol. 9, No. 5, pp. 605–611.
- Bogdanovich, A. E. and Pastore, C. M. (1996) *Mechanics of Textile and Laminated Composites with Applications to Structural Analysis*, Chapman & Hall, London.
- Chopra, I. (2002) "Review of State of Art of Smart Structures and Integrated Systems," *AIAA Journal*, Vol. 40, No. 11, pp. 2145–2187.
- Daniel, I. M. and Ishai, O. (1994) *Engineering Mechanics of Composite Materials*, Oxford University Press, New York.
- Eringen, A. C. and Maugin, G. A. (1990) *Electrodynamics of Continua I., Foundations and Solid Media*, Springer-Verlag, New York.
- Hanagud, S., Obal, M. W. and Calise, A. J. (1992) "Optimal Vibration Control by the Use of Piezoelectric Sensors and Actuators," *Journal of Guidance, Control and Dynamics*, Vol. 15, No. 5, pp. 1199–1206.
- Librescu, L. (1975) *Elastostatics and Kinetics of Anisotropic and Heterogeneous Shell-Type Structures*, Noordhoff International Publishing, Leyden, Netherlands.
- Librescu, L. (1990) "Theory of Composite Structures-Lecture Notes," Virginia Polytechnic Institute and State University, Blacksburg, VA.

- Maugin, G. A. (1988) *Continuum Mechanics and Electromagnetic Solids*, North-Holland, Amsterdam.
- Reddy, J. N. (2004) *Mechanics of Laminated Composite Plates and Shells: Theory and Analysis*, 2nd Edition, CRC Press, Boca Raton.
- Rehfield, L. W. and Atilgan, A. R. (1989) "Toward Understanding the Tailoring Mechanisms for Thin-Walled Composite Tubular Beams," in *Proceedings of the First USSR-U.S. Symposium on Mechanics of Composite Materials*, 23–26 May, Riga, Latvia SSR, S. W. Tsai, J. M., Whitney, T. W. Chou and R. M. Jones (Eds.), ASME Publ. House, pp. 187–196.
- Smith, E. C. and Chopra, I. (1991) "Formulation and Evaluation of an Analytical Model for Composite Box-Beams," *Journal of the American Helicopter Society*, Vol. 36, No. 3, pp. 23–35.
- Song, O. (1990) "Modeling and Response Analysis of Thin-Walled Beam Structures Constructed of Advanced Composite Material," Ph.D. Thesis Virginia Polytechnic Institute and State University, Blacksburg, VA.
- Sunar, M. and Rao, S. S. (1999) "Recent Advances in Sensing and Control of Flexible Structures via Piezoelectric Material Technology," *Applied Mechanics Reviews*, Vol. 52, No. 1, pp. 1–16.
- Tsai, W. and Hahn, M. T. (1980) *Introduction to Composite Materials*, Technomic Publishing Co., Westport, Connecticut.
- Tzou, H. S. and Zhong, J. P. (1992) "Adaptive Piezoelectric Structures: Theory and Experiment," in *Active Materials and Adaptive Structures, Materials and Structures Series*, G. J. Knowles (Ed.), Institute of Physics Publ., pp. 219–224.
- Tzou, H. S. and Anderson, G. L. (1992) *Intelligent Structural Systems*, Kluwer Academic Publisher, Norwell, MA.
- Tzou, H. S. (1993) *Piezoelectric Shells, Distributed Sensing and Control of Continua*, Kluwer Academic Publishers, Dordrecht.
- Vinson, J. R. and Sierakowski, R. L. (2002) *The Behavior of Structures Composed of Composite Materials*, Kluwer Academic Publishers, Dordrecht.
- Wempner, G. (1981) *Mechanics of Solids with Applications to Thin Bodies*, Sijthoff & Noordhoff, Alphen aan den Rijn, The Netherlands; Rockville, Maryland, USA.

SUBJECT INDEX

A

Acceleration feedback control, 346, 351–353
Active distributed control, 590
Actuating voltage, 150
Adaptive
 beams, 219
 capability, 453
 materials technology, 3
 rotating thin-walled beams, 279
 structure technology, 144, 225
Admissible functions, 130, 142
Aerodynamic indicial functions, 513
Aeroelasticity
 dynamic, 502, 513
 static, 502, 503
Aeroelastic control, 518
Aeroelastic instability 3
 dynamic, 513, 514
 static, 499, 503, 504, 508
Aeroelastic response
 dynamic, 515
 static, 503, 509, 518
Aeroelastic tailoring, 2
Aircraft wing, 2, 122, 500
Angle of incidence, 475
Angle of sweep, 503
Angular velocity, 438
Anisotropic, 2, 557, 558
Anisotropy, 233, 243, 395, 559, 590
Antisymmetric laminates, 582

Augmented performance index, 196
Axisymmetric beam, 399

B

Balanced laminates, 582
Bang-bang control, 207
Base vectors
 contravariant, 272
 covariant, 271, 373
Beam taper ratio, 204
Bending-twist coupling, 131, 136, 504, 508,
 510, 515, 519, 530
Bending-twist divergence, 506
Bernoulli-Euler beam model, 32, 169, 207,
 376, 382
Betti's reciprocal theorem, 112, 116
Bi-convex cross-section, 131, 140, 183, 202,
 507
Bifurcation of natural frequencies, 412, 428,
 448, 457, 466
Bilinear form, 111
Bimoment, 58
Blast load, 165, 170, 203
Boley-Barber-Thornton (BBT) approach, 484,
 491, 492
Boom, 475
Boundary moment control, 279, 453, 489, 518
Boundary-value problem, 113
Box beam, 154
Bredt-Batho equations, 23

Bredt's formula, 34
 Buckling coefficient, 421, 430
 Buckling load, 414, 415, 423, 424, 430, 433, 461

C

Cauchy stress tensor, 55
 Center manifold theory, 470
 Centrifugal
 acceleration, 233, 236, 332, 399
 destiffening, 261, 366
 inertia, 271
 rotatory effect, 241
 stiffening, 243, 248, 259, 261, 280, 289, 366
 warping, 241, 364
 Circular booms, 480
 Circular cross-section beam, 22
 Circular cylindrical shaft, 409
 Circumferentially Asymmetric Stiffness (CAS), 91
 Circumferentially Uniform Stiffness (CUS), 88
 Clapeyron's Theorem, 111
 Closed cross-section beams, 586
 Closed-loop eigenvalues, 153
 Closed/open contour, 8
 Coalescence of two eigenfrequencies, 403, 431, 446
 Coefficients
 hygrothermal, 559, 560, 563
 thermal, 559, 560, 563
 Composite materials, 8, 121, 475, 557
 Composite thin-walled beams
 multicell, 33
 single cell, 18
 Constitutive equations
 1-D elastic body, 426, 480, 557
 2-D elastic body, 575, 584, 586
 3-D elastic body, 557, 564, 574
 Constrained warping, 13
 Constraint equation, 196
 Contour origin, 15, 20
 Contour (primary), 18
 Contravariant metric tensor, 272
 Covariant base vectors, 271, 273
 Control induced damping, 156
 Convergence of the solution, 186, 278
 Converse piezoelectric effect, 3, 279, 590

Coriolis acceleration, 233, 332, 399, 419, 427
 Coriolis effect, 236, 242, 400, 420
 Corrected lift-curve slope coefficient, 503
 Cost functional, 190
 Coupled thermal-structural analysis, 479
 Coupling stiffness quantities, 81
 Covariant symmetric metric tensor, 271
 Critical compressive load, 414
 Critical spinning speed, 405, 448
 Cross-ply beam, 99, 125
 Cross-section non-deformability, 29, 48

D

Damped eigenfrequency, 158, 223, 349, 487
 Damping, 69, 79, 153, 157, 159, 349, 351, 487
 Dirac delta function, 215, 440
 Direct piezoelectric effect, 590
 Displacement field, 11, 28, 527
 Disturbance temperature, 477
 Divergence, 261
 aeroelastic, 499, 503
 speed, 508
 spin speed, 421, 449
 of swept wings, 504–507
 Divergence instability, 404, 411, 421, 429, 437, 445, 448, 452, 457, 461, 462, 466
 Divergence theorem, 54
 Domain of instability, 404, 405, 408
 Dynamic aeroelasticity, 512
 Dynamic pressure, 503
 Dynamic response
 closed-loop, 188, 207, 491
 control, 189
 open-loop, 192, 203, 204, 491
 thermally induced, 487
 to blast pulses, 186
 to sonic boom, 227
 to time-dependent loads, 116, 172, 179

E

Edge loads, 63
 Effect of boundary conditions on
 frequency-spin rate interaction, 407, 408
 divergence and flutter instabilities of spinning beams, 411–416
 Effect of gyroscopic terms on
 divergence instability of spinning beams, 405, 412, 415, 452

- eigenfrequencies, 405, 442
- flutter boundary, 415, 451, 452
- Effect of ply-angle on
 - aeroelastic divergence, 508, 509
 - divergence instability of spinning beams, 407, 409, 412, 413, 415
 - dynamic response, 187, 193, 204, 516
 - eigenfrequencies, 133–138, 261, 544, 548
 - flutter boundary, 451
 - static aeroelastic response, 510–512
 - static response, 540, 546
- Effect of rotatory inertia on
 - dynamic response, 185
 - eigenfrequencies, 138
 - flutter boundary, 403, 416
- Effective angle of attack, 504, 509
- Einstein summation convention, 558, 590
- Eigenvalue problem, 125, 129, 402
- Elastic couplings
 - bending-bending, 398, 444, 475, 481
 - bending-twist, 93, 136, 148, 501, 508
 - extension-bending, 94, 137, 528
 - extension-twist, 245, 353, 364
 - twist-transverse shear, 501
- Electric displacement, 149, 199
- Equations of motion, 64, 65, 71, 84, 87
- Excitation frequency, 178
- Explosive blast, 164
- Extended Galerkin Method (EGM), 117, 128, 142, 177, 186, 195, 250, 274, 278, 288, 337, 346, 364, 436, 491, 492, 512, 515, 539, 549
- F**
- Feedback control, 149, 182, 192, 195, 202, 221, 279, 347, 453, 466, 490, 519
- First Lyapunov quantity, 470
- First order transverse shear deformation theory (FSDT), 73, 75
- Flapping of rotating beams, 245
- Flexibility matrix of twist rate, 35
- Flight speed regimes, 513
- Flutter instability, 403, 411, 414–416, 431–434, 437, 445, 452, 462, 467, 516
- Flutter spin speed, 422
- Free torsion, 13
- Free vibration, 121, 544, 548
- Free warping, 134–136
- Frequency crossing, 259, 260
- Frequency response function, 176
- Functionally graded materials (FGMs), 4, 5, 310, 426
- G**
- Gauss' theorem, 107
- Generalized damping coefficients, 79
- Generalized load, 179, 487
- Generalized (modal) coordinates, 487
- Generalized velocities, 77
- Generally orthotropic material, 568
- Geometrically nonlinear rotating (spinning) beams, 375, 467
- Geometrically non-linear theory, 27, 45, 53, 66
- Governing equations corresponding to:
 - bending-bending elastic coupling, 123, 146, 481, 535
 - bending-twist elastic coupling, 124, 146, 166, 501, 530
 - extension-bending, 124, 528
- Green-Lagrange's strain tensor, 29, 375
- Gyroelastic systems, 437
- Gyroscopic Coriolis force, 242, 412, 446, 457
- Gyroscopic effects, 405, 412, 415, 428, 442, 452
- Gyroscopic system, 395, 401
- H**
- Hamilton's principle, 53, 128, 215, 439
- Heat flux, 477
- Helical multifilament concept, 265, 274
- Higher order shear deformation theory (HSDT), 44, 73
- Higher-order stress couple, 56
- Hub radius, 234, 305
- I**
- I-beam, 526
- Indicial function method, 513, 514
- Induced damping, 156, 159, 281, 349
- Induced strain actuation, 144, 345, 518
- Inertia and gyroscopic terms, 378
- Inertial warping, 241
- Initial conditions, 86
- Instantaneous Optimal Control (IOC), 195, 196, 204, 230

Inverse Fourier transform, 174
 Inverse Laplace transform, 128
 Isotropic, 562, 563

K

Kinematic boundary conditions, 130
 Kinetic energy, 54

L

Lagging of rotating blades, 245
 Lagrange's strain tensor, 30, 55
 Lagrangian multiplier, 197
 Laplace transform, 125, 143, 173, 508, 512
 Linear beam theory, 67
 Linear quadratic regulator (LQR)
 classical, 190, 195, 204, 228
 instantaneous, 204, 230
 Load factor, 503
 Loaded (unloaded) spinning beam, 404

M

Mach number, 514
 Massive bodies, 7
 Membrane shear strain, 14
 Middle surface, 9
 Mid-line, 9
 Mode shapes, 262, 308, 342, 371
 Modified LQR, 196
 Modified shell-stiffness quantities, 81
 Modified strip analysis, 512
 Modulus-weighted thickness, 22, 34
 Monoclinic, 559, 563
 Mori-Tanaka scheme, 328, 330, 435, 436
 Mounted stores, 213
 Multicell, 8, 33

N

Negative definite, 445
 Non-classical effects, 5, 395
 Non-conservative systems, 395
 Non-linear beam theory, 27
 Nonlinear Coriolis effect, 383
 Non-shearable beam theory, 26, 399
 Non-shearable shaft, 411
 Non-uniform cross-section beams, 139, 501

Non-uniform torsion, 13
 Norm, 115, 487
 Normal mode approach, 178
 N-shaped pressure pulse, 172

O

Open/closed cross-section beams, 25
 Open cross-section beams, 14, 525
 Optimal control, 190, 191
 Orthogonality condition, 179
 Orthogonality of eigenmodes, 114, 486
 Orthotropic, 561
 Out-of-phase actuation, 279, 453

P

Partial fraction expansion, 128
 Performance index, 198
 Piezoactuators, 4, 198
 Piezoactuators and sensors, 453, 455
 Piezoelectric coefficients, 590, 591
 Piezoelectric materials, 589
 Piezoelectric strain actuation, 457
 Piezoelectrically
 induced damping, 223, 281, 282
 induced moments, 144, 196, 198, 221, 279,
 346, 454, 519
 induced stress resultant and stress
 couple, 593
 Piezopatch, 228
 Piezosensors, 455
 Piola-Kirchhoff stress tensor, 55
 Plates and shells, 7
 Ply-angle, 131
 Pole, 11, 15, 46
 Positive definite, 109, 445
 Presetting, 296
 Presetting and pretwist, 296
 Pretwist (PT), 353, 364, 370, 417
 Pretwisted rotating beams, 263
 Primary warping, 5, 15, 18, 22, 140
 Principal axes, 264, 297
 Proportional feedback control, 155, 192
 Pure bending divergence, 504
 Pure torsion divergence, 505
 Pyroelectric effect, 491
 Pyrolytic graphite, 562

Q

Quasi-static solution, 485

R

Radiation input function, 478
 Rayleigh quotient method applied to
 aeroelasticity of aircraft wings, 504, 505
 rotating blades, 247, 299, 346
 Rayleigh's dissipation function, 69, 79
 Resonance, 182, 184, 185, 193
 Response
 aeroelastic static, 509–512
 static, 540, 546
 Riccati equation, 198
 Rigid angle of attack, 503
 Robotic manipulator arm, 144, 213, 436
 Rotating coordinate system, 234
 Rotating (whirling) frequencies, 402
 Rotatory inertia, 138, 241, 403
 Rotor blades, 233

S

Saint-Venant torsion, 364
 Saturation constraint, 198
 Second Piola-Kirchhoff stress tensor, 28, 54,
 375
 Secondary warping, 5, 16, 18, 22, 136, 184
 Sectorial area, 15
 Sectorial origin, 15
 Self-adjointness, 113
 Self-excited vibrations, 475
 Sensing and actuation, 149
 Sensor output equation, 199
 Sensor output voltage, 149
 Setting angle, 298, 301, 327
 Shear flow, 19, 22, 33, 34
 Shearable beam theory, 26
 Shell stress resultants, 56, 57
 Shock pulse, 171
 Simple rule of mixture, 434, 435
 Skew-symmetric gyroscopic matrix, 401
 Smart beam, 221
 Smart thin-walled structures, 146, 166
 Solar incidence angle, 488
 Solar radiation, 475
 Solid beams, 7, 104

Sonic boom, 170
 Southwell form, 249
 Spacecraft booms, 213, 475
 Spinning tip rotor, 437
 Stability plots, 451
 State-space form, 153, 177, 190, 196, 222
 Static aeroelastic response, 509
 Steady-state mean temperature, 477
 Stefan-Boltzmann law, 477
 Stiffnesses
 bending-extension, 137
 bending-twist, 137, 148, 501, 504, 508
 Strain energy, 54
 Stress couples, 56, 57
 Stress-resultants and stress-couples, 576
 Structural coupling, 88
 Structural damping, 228
 Structural tailoring, 2, 3, 159, 408, 453, 569
 Subcritical static aeroelastic response, 509
 Swept wings, 504
 Swept-back wing, 511
 Swept-forward wing, 504–507
 Symmetric laminates, 61, 581, 582
 Synergistic effects of
 tailoring and piezoelectric actuation, 467

T

Taper ratios, 140, 203, 204, 321
 Temperature dependence of Young's
 moduli, 287
 Temperature gradient, 313, 434
 Tennis-racket (TR) effect, 241, 362, 364, 366,
 370
 Tension-torsion coupling (TT), 364, 366, 370
 Theorem of positive work, 109
 Theorem of power and energy, 107
 Thermal conductivity, 313, 417
 Thermal degradation, 286, 288, 290, 415, 434,
 488
 Thermal flutter, 525
 Thermal gradient, 286, 319
 Thermally induced vibration, 484
 Thermoelastic coefficients, 481
 Thickness (secondary) warping, 18
 Thick-walled beam, 8
 Thin/thick walled beams, 7, 8
 Thin-walled beam, 1, 8, 14, 404
 Three-cell section beam, 42
 Time-dependent loads, 165, 172

Time-dependent thermal loading, 480
 Timoshenko beam model, 316
 Tip mass, 330
 Tip rotor, 437
 Torsional function, 21, 57, 141, 235
 Torsion-related nonuniform warping, 122
 Transverse isotropy, 180, 370
 Transverse shear, 5, 15, 32, 122, 133, 137, 185, 277, 416, 511, 517, 542–546
 Triclinic, 559
 Twist-rate (rate of twist), 13, 22, 33
 Two-cell section beam, 40

U

Uncoupled thermal-structural analysis, 479
 Unidirectional fiber reinforced composites, 562, 573
 Uniqueness of the solution, 86, 109
 Unshearable beam, 18, 87, 96, 217, 227, 269
 Unsymmetric laminate, 581
 Upper and lower frequency branches, 405, 428, 429

V

Velocity feedback control, 150, 192, 203, 228, 281
 Virtual work principle, 110
 Volume fraction parameter, 318, 428, 432

W

Wagner's approach, 266, 271, 372
 Warping

- displacement, 14
- function, 16, 18, 23, 37
- inertia, 502
- restraint, 33, 134, 364, 370, 506, 517, 537, 543, 547
- torque, 58

 Wash-in, 504, 510, 511
 Wash-out, 504, 511
 Washizu's approach, 266, 271, 372
 Weighting matrices, 197
 Wing aspect-ratio, 503

Mechanics

SOLID MECHANICS AND ITS APPLICATIONS

Series Editor: G.M.L. Gladwell

Aims and Scope of the Series

The fundamental questions arising in mechanics are: *Why?*, *How?*, and *How much?* The aim of this series is to provide lucid accounts written by authoritative researchers giving vision and insight in answering these questions on the subject of mechanics as it relates to solids. The scope of the series covers the entire spectrum of solid mechanics. Thus it includes the foundation of mechanics; variational formulations; computational mechanics; statics, kinematics and dynamics of rigid and elastic bodies; vibrations of solids and structures; dynamical systems and chaos; the theories of elasticity, plasticity and viscoelasticity; composite materials; rods, beams, shells and membranes; structural control and stability; soils, rocks and geomechanics; fracture; tribology; experimental mechanics; biomechanics and machine design.

1. R.T. Haftka, Z. Gürdal and M.P. Kamat: *Elements of Structural Optimization*. 2nd rev.ed., 1990
ISBN 0-7923-0608-2
2. J.J. Kalker: *Three-Dimensional Elastic Bodies in Rolling Contact*. 1990 ISBN 0-7923-0712-7
3. P. Karasudhi: *Foundations of Solid Mechanics*. 1991 ISBN 0-7923-0772-0
4. *Not published*
5. *Not published*.
6. J.F. Doyle: *Static and Dynamic Analysis of Structures*. With an Emphasis on Mechanics and Computer Matrix Methods. 1991 ISBN 0-7923-1124-8; Pb 0-7923-1208-2
7. O.O. Ochoa and J.N. Reddy: *Finite Element Analysis of Composite Laminates*.
ISBN 0-7923-1125-6
8. M.H. Aliabadi and D.P. Rooke: *Numerical Fracture Mechanics*. ISBN 0-7923-1175-2
9. J. Angeles and C.S. López-Cajún: *Optimization of Cam Mechanisms*. 1991
ISBN 0-7923-1355-0
10. D.E. Grierson, A. Franchi and P. Riva (eds.): *Progress in Structural Engineering*. 1991
ISBN 0-7923-1396-8
11. R.T. Haftka and Z. Gürdal: *Elements of Structural Optimization*. 3rd rev. and exp. ed. 1992
ISBN 0-7923-1504-9; Pb 0-7923-1505-7
12. J.R. Barber: *Elasticity*. 1992 ISBN 0-7923-1609-6; Pb 0-7923-1610-X
13. H.S. Tzou and G.L. Anderson (eds.): *Intelligent Structural Systems*. 1992
ISBN 0-7923-1920-6
14. E.E. Gdoutos: *Fracture Mechanics*. An Introduction. 1993 ISBN 0-7923-1932-X
15. J.P. Ward: *Solid Mechanics*. An Introduction. 1992 ISBN 0-7923-1949-4
16. M. Farshad: *Design and Analysis of Shell Structures*. 1992 ISBN 0-7923-1950-8
17. H.S. Tzou and T. Fukuda (eds.): *Precision Sensors, Actuators and Systems*. 1992
ISBN 0-7923-2015-8
18. J.R. Vinson: *The Behavior of Shells Composed of Isotropic and Composite Materials*. 1993
ISBN 0-7923-2113-8
19. H.S. Tzou: *Piezoelectric Shells*. Distributed Sensing and Control of Continua. 1993
ISBN 0-7923-2186-3
20. W. Schiehlen (ed.): *Advanced Multibody System Dynamics*. Simulation and Software Tools.
1993 ISBN 0-7923-2192-8
21. C.-W. Lee: *Vibration Analysis of Rotors*. 1993 ISBN 0-7923-2300-9
22. D.R. Smith: *An Introduction to Continuum Mechanics*. 1993 ISBN 0-7923-2454-4
23. G.M.L. Gladwell: *Inverse Problems in Scattering*. An Introduction. 1993 ISBN 0-7923-2478-1

Mechanics

SOLID MECHANICS AND ITS APPLICATIONS

Series Editor: G.M.L. Gladwell

24. G. Prathap: *The Finite Element Method in Structural Mechanics*. 1993 ISBN 0-7923-2492-7
25. J. Herskovits (ed.): *Advances in Structural Optimization*. 1995 ISBN 0-7923-2510-9
26. M.A. González-Palacios and J. Angeles: *Cam Synthesis*. 1993 ISBN 0-7923-2536-2
27. W.S. Hall: *The Boundary Element Method*. 1993 ISBN 0-7923-2580-X
28. J. Angeles, G. Hommel and P. Kovács (eds.): *Computational Kinematics*. 1993
ISBN 0-7923-2585-0
29. A. Curnier: *Computational Methods in Solid Mechanics*. 1994 ISBN 0-7923-2761-6
30. D.A. Hills and D. Nowell: *Mechanics of Fretting Fatigue*. 1994 ISBN 0-7923-2866-3
31. B. Tabarrok and F.P.J. Rimrott: *Variational Methods and Complementary Formulations in Dynamics*. 1994 ISBN 0-7923-2923-6
32. E.H. Dowell (ed.), E.F. Crawley, H.C. Curtiss Jr., D.A. Peters, R. H. Scanlan and F. Sisto: *A Modern Course in Aeroelasticity*. Third Revised and Enlarged Edition. 1995
ISBN 0-7923-2788-8; Pb: 0-7923-2789-6
33. A. Preumont: *Random Vibration and Spectral Analysis*. 1994 ISBN 0-7923-3036-6
34. J.N. Reddy (ed.): *Mechanics of Composite Materials*. Selected works of Nicholas J. Pagano. 1994
ISBN 0-7923-3041-2
35. A.P.S. Selvadurai (ed.): *Mechanics of Poroelastic Media*. 1996 ISBN 0-7923-3329-2
36. Z. Mróz, D. Weichert, S. Dorosz (eds.): *Inelastic Behaviour of Structures under Variable Loads*. 1995
ISBN 0-7923-3397-7
37. R. Pyrz (ed.): *IUTAM Symposium on Microstructure-Property Interactions in Composite Materials*. Proceedings of the IUTAM Symposium held in Aalborg, Denmark. 1995
ISBN 0-7923-3427-2
38. M.I. Friswell and J.E. Mottershead: *Finite Element Model Updating in Structural Dynamics*. 1995
ISBN 0-7923-3431-0
39. D.F. Parker and A.H. England (eds.): *IUTAM Symposium on Anisotropy, Inhomogeneity and Nonlinearity in Solid Mechanics*. Proceedings of the IUTAM Symposium held in Nottingham, U.K. 1995
ISBN 0-7923-3594-5
40. J.-P. Merlet and B. Ravani (eds.): *Computational Kinematics '95*. 1995 ISBN 0-7923-3673-9
41. L.P. Lebedev, I.I. Vorovich and G.M.L. Gladwell: *Functional Analysis*. Applications in Mechanics and Inverse Problems. 1996
ISBN 0-7923-3849-9
42. J. Menčík: *Mechanics of Components with Treated or Coated Surfaces*. 1996
ISBN 0-7923-3700-X
43. D. Bestle and W. Schiehlen (eds.): *IUTAM Symposium on Optimization of Mechanical Systems*. Proceedings of the IUTAM Symposium held in Stuttgart, Germany. 1996
ISBN 0-7923-3830-8
44. D.A. Hills, P.A. Kelly, D.N. Dai and A.M. Korsunsky: *Solution of Crack Problems*. The Distributed Dislocation Technique. 1996
ISBN 0-7923-3848-0
45. V.A. Squire, R.J. Hosking, A.D. Kerr and P.J. Langhorne: *Moving Loads on Ice Plates*. 1996
ISBN 0-7923-3953-3
46. A. Pineau and A. Zaoui (eds.): *IUTAM Symposium on Micromechanics of Plasticity and Damage of Multiphase Materials*. Proceedings of the IUTAM Symposium held in Sèvres, Paris, France. 1996
ISBN 0-7923-4188-0
47. A. Naess and S. Krenk (eds.): *IUTAM Symposium on Advances in Nonlinear Stochastic Mechanics*. Proceedings of the IUTAM Symposium held in Trondheim, Norway. 1996
ISBN 0-7923-4193-7
48. D. Ieşan and A. Scalia: *Thermoelastic Deformations*. 1996
ISBN 0-7923-4230-5

Mechanics

SOLID MECHANICS AND ITS APPLICATIONS

Series Editor: G.M.L. Gladwell

49. J.R. Willis (ed.): *IUTAM Symposium on Nonlinear Analysis of Fracture*. Proceedings of the IUTAM Symposium held in Cambridge, U.K. 1997 ISBN 0-7923-4378-6
50. A. Preumont: *Vibration Control of Active Structures*. An Introduction. 1997 ISBN 0-7923-4392-1
51. G.P. Cherepanov: *Methods of Fracture Mechanics: Solid Matter Physics*. 1997 ISBN 0-7923-4408-1
52. D.H. van Campen (ed.): *IUTAM Symposium on Interaction between Dynamics and Control in Advanced Mechanical Systems*. Proceedings of the IUTAM Symposium held in Eindhoven, The Netherlands. 1997 ISBN 0-7923-4429-4
53. N.A. Fleck and A.C.F. Cocks (eds.): *IUTAM Symposium on Mechanics of Granular and Porous Materials*. Proceedings of the IUTAM Symposium held in Cambridge, U.K. 1997 ISBN 0-7923-4553-3
54. J. Roorda and N.K. Srivastava (eds.): *Trends in Structural Mechanics*. Theory, Practice, Education. 1997 ISBN 0-7923-4603-3
55. Yu.A. Mitropolskii and N. Van Dao: *Applied Asymptotic Methods in Nonlinear Oscillations*. 1997 ISBN 0-7923-4605-X
56. C. Guedes Soares (ed.): *Probabilistic Methods for Structural Design*. 1997 ISBN 0-7923-4670-X
57. D. François, A. Pineau and A. Zaoui: *Mechanical Behaviour of Materials*. Volume I: Elasticity and Plasticity. 1998 ISBN 0-7923-4894-X
58. D. François, A. Pineau and A. Zaoui: *Mechanical Behaviour of Materials*. Volume II: Viscoplasticity, Damage, Fracture and Contact Mechanics. 1998 ISBN 0-7923-4895-8
59. L.T. Tenek and J. Argyris: *Finite Element Analysis for Composite Structures*. 1998 ISBN 0-7923-4899-0
60. Y.A. Bahei-El-Din and G.J. Dvorak (eds.): *IUTAM Symposium on Transformation Problems in Composite and Active Materials*. Proceedings of the IUTAM Symposium held in Cairo, Egypt. 1998 ISBN 0-7923-5122-3
61. I.G. Goryacheva: *Contact Mechanics in Tribology*. 1998 ISBN 0-7923-5257-2
62. O.T. Bruhns and E. Stein (eds.): *IUTAM Symposium on Micro- and Macrostructural Aspects of Thermoplasticity*. Proceedings of the IUTAM Symposium held in Bochum, Germany. 1999 ISBN 0-7923-5265-3
63. F.C. Moon: *IUTAM Symposium on New Applications of Nonlinear and Chaotic Dynamics in Mechanics*. Proceedings of the IUTAM Symposium held in Ithaca, NY, USA. 1998 ISBN 0-7923-5276-9
64. R. Wang: *IUTAM Symposium on Rheology of Bodies with Defects*. Proceedings of the IUTAM Symposium held in Beijing, China. 1999 ISBN 0-7923-5297-1
65. Yu.I. Dimitrienko: *Thermomechanics of Composites under High Temperatures*. 1999 ISBN 0-7923-4899-0
66. P. Argoul, M. Frémond and Q.S. Nguyen (eds.): *IUTAM Symposium on Variations of Domains and Free-Boundary Problems in Solid Mechanics*. Proceedings of the IUTAM Symposium held in Paris, France. 1999 ISBN 0-7923-5450-8
67. F.J. Fahy and W.G. Price (eds.): *IUTAM Symposium on Statistical Energy Analysis*. Proceedings of the IUTAM Symposium held in Southampton, U.K. 1999 ISBN 0-7923-5457-5
68. H.A. Mang and F.G. Rammerstorfer (eds.): *IUTAM Symposium on Discretization Methods in Structural Mechanics*. Proceedings of the IUTAM Symposium held in Vienna, Austria. 1999 ISBN 0-7923-5591-1

Mechanics

SOLID MECHANICS AND ITS APPLICATIONS

Series Editor: G.M.L. Gladwell

69. P. Pedersen and M.P. Bendsøe (eds.): *IUTAM Symposium on Synthesis in Bio Solid Mechanics*. Proceedings of the IUTAM Symposium held in Copenhagen, Denmark. 1999
ISBN 0-7923-5615-2
70. S.K. Agrawal and B.C. Fabien: *Optimization of Dynamic Systems*. 1999
ISBN 0-7923-5681-0
71. A. Carpinteri: *Nonlinear Crack Models for Nonmetallic Materials*. 1999
ISBN 0-7923-5750-7
72. F. Pfeifer (ed.): *IUTAM Symposium on Unilateral Multibody Contacts*. Proceedings of the IUTAM Symposium held in Munich, Germany. 1999
ISBN 0-7923-6030-3
73. E. Lavendelis and M. Zkrzhevsky (eds.): *IUTAM/IFToMM Symposium on Synthesis of Non-linear Dynamical Systems*. Proceedings of the IUTAM/IFToMM Symposium held in Riga, Latvia. 2000
ISBN 0-7923-6106-7
74. J.-P. Merlet: *Parallel Robots*. 2000
ISBN 0-7923-6308-6
75. J.T. Pindera: *Techniques of Tomographic Isodyne Stress Analysis*. 2000
ISBN 0-7923-6388-4
76. G.A. Maugin, R. Drouot and F. Sidoroff (eds.): *Continuum Thermomechanics*. The Art and Science of Modelling Material Behaviour. 2000
ISBN 0-7923-6407-4
77. N. Van Dao and E.J. Kreuzer (eds.): *IUTAM Symposium on Recent Developments in Non-linear Oscillations of Mechanical Systems*. 2000
ISBN 0-7923-6470-8
78. S.D. Akbarov and A.N. Guz: *Mechanics of Curved Composites*. 2000
ISBN 0-7923-6477-5
79. M.B. Rubin: *Cosserat Theories: Shells, Rods and Points*. 2000
ISBN 0-7923-6489-9
80. S. Pellegrino and S.D. Guest (eds.): *IUTAM-IASS Symposium on Deployable Structures: Theory and Applications*. Proceedings of the IUTAM-IASS Symposium held in Cambridge, U.K., 6–9 September 1998. 2000
ISBN 0-7923-6516-X
81. A.D. Rosato and D.L. Blackmore (eds.): *IUTAM Symposium on Segregation in Granular Flows*. Proceedings of the IUTAM Symposium held in Cape May, NJ, U.S.A., June 5–10, 1999. 2000
ISBN 0-7923-6547-X
82. A. Lagarde (ed.): *IUTAM Symposium on Advanced Optical Methods and Applications in Solid Mechanics*. Proceedings of the IUTAM Symposium held in Futuroscope, Poitiers, France, August 31–September 4, 1998. 2000
ISBN 0-7923-6604-2
83. D. Weichert and G. Maier (eds.): *Inelastic Analysis of Structures under Variable Loads*. Theory and Engineering Applications. 2000
ISBN 0-7923-6645-X
84. T.-J. Chuang and J.W. Rudnicki (eds.): *Multiscale Deformation and Fracture in Materials and Structures*. The James R. Rice 60th Anniversary Volume. 2001
ISBN 0-7923-6718-9
85. S. Narayanan and R.N. Iyengar (eds.): *IUTAM Symposium on Nonlinearity and Stochastic Structural Dynamics*. Proceedings of the IUTAM Symposium held in Madras, Chennai, India, 4–8 January 1999
ISBN 0-7923-6733-2
86. S. Murakami and N. Ohno (eds.): *IUTAM Symposium on Creep in Structures*. Proceedings of the IUTAM Symposium held in Nagoya, Japan, 3-7 April 2000. 2001
ISBN 0-7923-6737-5
87. W. Ehlers (ed.): *IUTAM Symposium on Theoretical and Numerical Methods in Continuum Mechanics of Porous Materials*. Proceedings of the IUTAM Symposium held at the University of Stuttgart, Germany, September 5-10, 1999. 2001
ISBN 0-7923-6766-9
88. D. Durban, D. Givoli and J.G. Simmonds (eds.): *Advances in the Mechanis of Plates and Shells The Avinoam Libai Anniversary Volume*. 2001
ISBN 0-7923-6785-5
89. U. Gabbert and H.-S. Tzou (eds.): *IUTAM Symposium on Smart Structures and Structonic Systems*. Proceedings of the IUTAM Symposium held in Magdeburg, Germany, 26–29 September 2000. 2001
ISBN 0-7923-6968-8

Mechanics

SOLID MECHANICS AND ITS APPLICATIONS

Series Editor: G.M.L. Gladwell

90. Y. Ivanov, V. Cheshkov and M. Natova: *Polymer Composite Materials – Interface Phenomena & Processes*. 2001 ISBN 0-7923-7008-2
91. R.C. McPhedran, L.C. Botten and N.A. Nicorovici (eds.): *IUTAM Symposium on Mechanical and Electromagnetic Waves in Structured Media*. Proceedings of the IUTAM Symposium held in Sydney, NSW, Australia, 18-22 Januari 1999. 2001 ISBN 0-7923-7038-4
92. D.A. Sotiropoulos (ed.): *IUTAM Symposium on Mechanical Waves for Composite Structures Characterization*. Proceedings of the IUTAM Symposium held in Chania, Crete, Greece, June 14-17, 2000. 2001 ISBN 0-7923-7164-X
93. V.M. Alexandrov and D.A. Pozharskii: *Three-Dimensional Contact Problems*. 2001 ISBN 0-7923-7165-8
94. J.P. Dempsey and H.H. Shen (eds.): *IUTAM Symposium on Scaling Laws in Ice Mechanics and Ice Dynamics*. Proceedings of the IUTAM Symposium held in Fairbanks, Alaska, U.S.A., 13-16 June 2000. 2001 ISBN 1-4020-0171-1
95. U. Kirsch: *Design-Oriented Analysis of Structures. A Unified Approach*. 2002 ISBN 1-4020-0443-5
96. A. Preumont: *Vibration Control of Active Structures. An Introduction (2nd Edition)*. 2002 ISBN 1-4020-0496-6
97. B.L. Karihaloo (ed.): *IUTAM Symposium on Analytical and Computational Fracture Mechanics of Non-Homogeneous Materials*. Proceedings of the IUTAM Symposium held in Cardiff, U.K., 18-22 June 2001. 2002 ISBN 1-4020-0510-5
98. S.M. Han and H. Benaroya: *Nonlinear and Stochastic Dynamics of Compliant Offshore Structures*. 2002 ISBN 1-4020-0573-3
99. A.M. Linkov: *Boundary Integral Equations in Elasticity Theory*. 2002 ISBN 1-4020-0574-1
100. L.P. Lebedev, I.I. Vorovich and G.M.L. Gladwell: *Functional Analysis. Applications in Mechanics and Inverse Problems (2nd Edition)*. 2002 ISBN 1-4020-0667-5; Pb: 1-4020-0756-6
101. Q.P. Sun (ed.): *IUTAM Symposium on Mechanics of Martensitic Phase Transformation in Solids*. Proceedings of the IUTAM Symposium held in Hong Kong, China, 11-15 June 2001. 2002 ISBN 1-4020-0741-8
102. M.L. Munjal (ed.): *IUTAM Symposium on Designing for Quietness*. Proceedings of the IUTAM Symposium held in Bangkok, India, 12-14 December 2000. 2002 ISBN 1-4020-0765-5
103. J.A.C. Martins and M.D.P. Monteiro Marques (eds.): *Contact Mechanics*. Proceedings of the 3rd Contact Mechanics International Symposium, Praia da Consolação, Peniche, Portugal, 17-21 June 2001. 2002 ISBN 1-4020-0811-2
104. H.R. Drew and S. Pellegrino (eds.): *New Approaches to Structural Mechanics, Shells and Biological Structures*. 2002 ISBN 1-4020-0862-7
105. J.R. Vinson and R.L. Sierakowski: *The Behavior of Structures Composed of Composite Materials*. Second Edition. 2002 ISBN 1-4020-0904-6
106. Not yet published.
107. J.R. Barber: *Elasticity*. Second Edition. 2002 ISBN Hb 1-4020-0964-X; Pb 1-4020-0966-6
108. C. Miehe (ed.): *IUTAM Symposium on Computational Mechanics of Solid Materials at Large Strains*. Proceedings of the IUTAM Symposium held in Stuttgart, Germany, 20-24 August 2001. 2003 ISBN 1-4020-1170-9

Mechanics

SOLID MECHANICS AND ITS APPLICATIONS

Series Editor: G.M.L. Gladwell

109. P. Ståhle and K.G. Sundin (eds.): *IUTAM Symposium on Field Analyses for Determination of Material Parameters – Experimental and Numerical Aspects*. Proceedings of the IUTAM Symposium held in Abisko National Park, Kiruna, Sweden, July 31 – August 4, 2000. 2003
ISBN 1-4020-1283-7
110. N. Sri Namachchivaya and Y.K. Lin (eds.): *IUTAM Symposium on Nonlinear Stochastic Dynamics*. Proceedings of the IUTAM Symposium held in Monticello, IL, USA, 26 – 30 August, 2000. 2003
ISBN 1-4020-1471-6
111. H. Sobieckzy (ed.): *IUTAM Symposium Transsonicum IV*. Proceedings of the IUTAM Symposium held in Göttingen, Germany, 2–6 September 2002, 2003
ISBN 1-4020-1608-5
112. J.-C. Samin and P. Fiset: *Symbolic Modeling of Multibody Systems*. 2003
ISBN 1-4020-1629-8
113. A.B. Movchan (ed.): *IUTAM Symposium on Asymptotics, Singularities and Homogenisation in Problems of Mechanics*. Proceedings of the IUTAM Symposium held in Liverpool, United Kingdom, 8-11 July 2002. 2003
ISBN 1-4020-1780-4
114. S. Ahzi, M. Cherkaoui, M.A. Khaleel, H.M. Zbib, M.A. Zikry and B. LaMatina (eds.): *IUTAM Symposium on Multiscale Modeling and Characterization of Elastic-Inelastic Behavior of Engineering Materials*. Proceedings of the IUTAM Symposium held in Marrakech, Morocco, 20-25 October 2002. 2004
ISBN 1-4020-1861-4
115. H. Kitagawa and Y. Shibutani (eds.): *IUTAM Symposium on Mesoscopic Dynamics of Fracture Process and Materials Strength*. Proceedings of the IUTAM Symposium held in Osaka, Japan, 6-11 July 2003. Volume in celebration of Professor Kitagawa's retirement. 2004
ISBN 1-4020-2037-6
116. E.H. Dowell, R.L. Clark, D. Cox, H.C. Curtiss, Jr., K.C. Hall, D.A. Peters, R.H. Scanlan, E. Simiu, F. Sisto and D. Tang: *A Modern Course in Aeroelasticity*. 4th Edition, 2004
ISBN 1-4020-2039-2
117. T. Burczyński and A. Osyczka (eds.): *IUTAM Symposium on Evolutionary Methods in Mechanics*. Proceedings of the IUTAM Symposium held in Cracow, Poland, 24-27 September 2002. 2004
ISBN 1-4020-2266-2
118. D. Ieşan: *Thermoelastic Models of Continua*. 2004
ISBN 1-4020-2309-X
119. G.M.L. Gladwell: *Inverse Problems in Vibration*. Second Edition. 2004
ISBN 1-4020-2670-6
120. J.R. Vinson: *Plate and Panel Structures of Isotropic, Composite and Piezoelectric Materials, Including Sandwich Construction*. 2005
ISBN 1-4020-3110-6
121. *Forthcoming*
122. G. Rega and F. Vestroni (eds.): *IUTAM Symposium on Chaotic Dynamics and Control of Systems and Processes in Mechanics*. Proceedings of the IUTAM Symposium held in Rome, Italy, 8–13 June 2003. 2005
ISBN 1-4020-3267-6
123. E.E. Gdoutos: *Fracture Mechanics. An Introduction. 2nd edition*. 2005
ISBN 1-4020-3267-6
124. M.D. Gilchrist (ed.): *IUTAM Symposium on Impact Biomechanics from Fundamental Insights to Applications*. 2005
ISBN 1-4020-3795-3
125. J.M. Huyghe, P.A.C. Raats and S. C. Cowin (eds.): *IUTAM Symposium on Physicochemical and Electromechanical Interactions in Porous Media*. 2005
ISBN 1-4020-3864-X
126. H. Ding and W. Chen: *Elasticity of Transversely Isotropic Materials*. 2005
ISBN 1-4020-4033-4
127. W. Yang (ed): *IUTAM Symposium on Mechanics and Reliability of Actuating Materials*. Proceedings of the IUTAM Symposium held in Beijing, China, 1–3 September 2004. 2005
ISBN 1-4020-4131-6
128. J.P. Merlet: *Parallel Robots*. 2005
ISBN 1-4020-4132-2

Mechanics

SOLID MECHANICS AND ITS APPLICATIONS

Series Editor: G.M.L. Gladwell

129. G.E.A. Meier and K.R. Sreenivasan (eds.): *IUTAM Symposium on One Hundred Years of Boundary Layer Research*. Proceedings of the IUTAM Symposium held at DLR-Göttingen, Germany, August 12–14, 2004. 2005 ISBN 1-4020-4149-7
130. H. Ulbrich and W. Günthner (eds.): *IUTAM Symposium on Vibration Control of Nonlinear Mechanisms and Structures*. 2005 ISBN 1-4020-4160-8
131. L. Librescu and O. Song: *Thin-Walled Composite Beams*. Theory and Application. 2006 ISBN 1-4020-3457-1
132. G. Ben-Dor, A. Dubinsky and T. Elperin: *Applied High-Speed Plate Penetration Dynamics*. 2006 ISBN 1-4020-3452-0